

Isotherms and Streamlines for 2D Lid Driven

Square Cavity



Garepally Srinivas, A.V Ramana Kumari, V. Narayana

Abstract: Analysis of lid driven square cavity flow of air with three different ranges of Ri and Re are analyzed using numerically. Adiabatic temperature is maintained at horizontal walls and isothermal temperature is established at the vertical walls in which the top wall is assumed to slide with a uniform speed. Finite volume method techniques have used to solve non dimensional governing equations. To visualize the flow and thermal characteristics, the control parameters, the Richardson number (Ri) and Reynolds number (Re) and in the range of 0.001 $\leq Ri \leq 10$ and $100 \leq Re \leq 400$ are used for streamlines and isotherms.

Keywords: Reynolds number, Richardson number, Mixed Convection.

I. INTRODUCTION

 ${f M}$ uch attention have been paid for the few decades on heat transfer flows with natural and mixed convective flows which has many applications in food processing, glass production, crystal growth etc. In different cavity configurations many authors [1-2] have considered the flow structure and its temperature distribution.

Moallemi and Jang [3] investigated the buoyancy effect in a 2D cavity with respect to Pr $(0.01 \le Pr \le 50)$, Re $(10^2 \le Re \le 10^2)$ 2000)and constant Grashof number (Gr). They observed that natural convection usually assists the forced convection. With top moving wall, similar cavity problem for different aspect ratios (AR) was analyzed by Prasad et al. [4] numerically in two different ways, namely, the top moving wall is maintained at a higher temperature in comparison with the bottom wall and vice versa. They noticed that when AR = 0.5 and 1.0 and for negatively increasing values of Gr, convection is strong inside the cavity. When AR increased to 2 and for $Gr = -10^5$, Hopf bifurcation showed its presence. When AR value further increased to 10, Sharif [5] showed that along the heated sliding wall of the cavity the Nu shoots up and it decreases rapidly towards the right side of the wall.

In the present paper the contours of streamlines and

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isotherms are used to visualize the behaviour of the flow.

II. PHYSICAL MODEL AND GOVERNING **EQUATIONS**

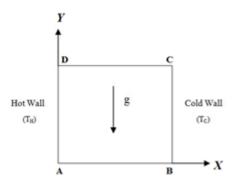


Fig. 1 2D Lid Driven Cavity

Figure 1 represents two dimensional lid driven cavity with velocity U of length L. Air is the working fluid in which isothermal temperature is maintained at left and right walls and adiabatic temperature is maintained at top and bottom walls with $(T_H - T_C > 0$, and the boundary conditions are

$$\Delta \cdot \mathbf{V} = 0 \tag{1}$$

$$(\mathbf{V}.\nabla)\mathbf{V} = \frac{1}{Re}\nabla^{2}\mathbf{V} - \nabla p + RiT^{*}\mathbf{e}$$
(2)

$$(\mathbf{V}.\nabla)T^{*} = \frac{1}{Pr.Re}\nabla^{2}T^{*}$$
(3)

$$(\mathbf{V}.\nabla)T^* = \frac{1}{Pr.Re}\nabla^2 T^* \tag{3}$$

The boundary conditions are as follows

$$V = (1, 0)$$
 at $Y = 1$ and $V = 0$ at $Y = 0, X = 0, 1$.

$$T^* = 0$$
 at $X = 1$ and $T^* = 1$ at $X = 0$ and (4)

$$\frac{\partial T^*}{\partial Y} = 0$$
, at $Y = 0$ and 1.

III. NUMERICAL METHOD

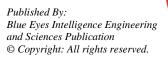
The open source software Open FOAM has been used. For solving derivatives, the Second order upwind differencing scheme was employed to solve the spatial derivates. Divergent terms are discretized by QUICK scheme. Gauss linear method is used to solve Laplacian terms.

IV. RESULTS AND DISCUSSION

Figure 2 displays the streamlines for different Ri and Re. It is found that the primary vortex take part a major role when Ri is small, with the increasing values of Re and it is changing from top position to bottom also it is found two tiny eddies at the bottom corners of the square cavity and also the size of tiny eddies are increases as Ri increases (Figs. 2(a, d)).

The primary major vortex moves towards the boundary walls and giving space for creating another vortex when Ri =1 and with increase of Re.

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Finally two primary vortices formed when Ri =10 and Re increases (Fig. 2(c)). The velocity of the fluid at the centre of the cavity are very small when compared with the velocities at the boundary when Re further increases.

Fig. 2. Streamlines for different Ri and Re values. (Ri, Re) = (a) (0.001, 100), (b) (1, 100), (c) (10, 100),(d) (0.001, 400), (e) (1, 400), (f) (10, 400)

Figure 3 gives the isotherms for different values of Ri and Re. From the figure, it is noticed that vertical isotherms have formed near the boundaries when Ri is small, with the increase of Re, this is due to the conduction. As Re further increases, due to the mechanism from the conduction mode to convection mode, the isotherm leaves from the vertical wall and found horizontal isotherms. Finally when Ri is further increases with the increase of Re, horizontal isotherms are observed in Figs. 3(c, f).

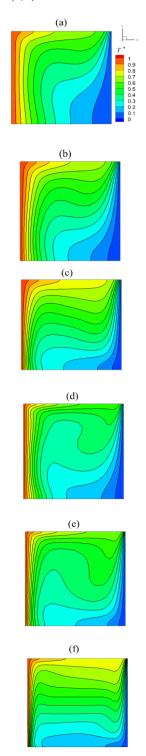


Fig. 3. Isotherms for different Ri and Re values. (Ri, Re) = (a) (0.001, 100), (b) (1, 100), (c) (10, 100),(d) (0.001, 400), (e) (1, 400), (f) (10, 400)





V. CONCLUSION

The present study is carried numerically for the mixed convective flow in a left heated lid driven 2D cavity for different ranges of Ri and Re. The simulated results of streamlines exhibited symmetric nature at the horizontal adiabatic walls. When Ri is small with the increase of Re, the primary vortex inhabited the major part of the cavity and also horizontal isotherms formed near the boundaries due to conduction.

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