IPT train

Problem 9: Rail track divination

April 22, 2019

Benjamin L. Larsen Danish team

International physicists' tournament

Danmarks Tekniske Universitet





Agenda

IPT train

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Sound generation

Fuler-Rernoulli

Wave number

Sound generation

Sound power

Sound power level

Experiment

Speed of sound

Summary

Issues with the model

Sound generation

Euler-Bernoulli

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Problem

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Issues with the model

The sound of an approaching train, propagating in metals, reaches our ears earlier than the train arrives. Is it possible to estimate the distance to the train and speed of its movement using this phenomenon? Estimate the accuracy and precision of your method.



Sound generation

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Summary

- Comes from roughness of wheel or track
- Displacement propagates through the rail
- Ground and sleeper damping, rail damping
- $\omega = \frac{2\pi V}{V}$
- λ undulations of the surfaces



Euler-Bernoulli Beam theory

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Issues with the model

$$EI\frac{\partial}{\partial x^4}u(x) + su(x) + m\frac{\partial}{\partial t^2}u(x) = 0$$
 (1)

- ► Valid for small deflections
- Lateral loads only
- E, Young modulus, I is moment area of cross-section, s is stiffness pr. unit length, m mass pr unit length

Non dimensionilization (resonance frequency, unsupported wavenumber)

$$\omega_0 = \sqrt{\frac{s}{m}} \quad k_b = \left(\frac{m\omega^2}{EI}\right)^{1/4} \tag{2}$$



Wave number

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Issues with the model

Damping: $\tilde{E} = E(1 + i\eta_E)$, $\tilde{s} = s(1 + i\eta_s)$ Assume solution of form

$$u(x) = U \exp i\omega t \exp -ikx \tag{3}$$

Insert and solve

$$\tilde{E}Ik^4 + \tilde{s} - m\omega^2 = 0 \Rightarrow k^2 = \pm \sqrt{\frac{m\omega^2 - s}{EI}}$$
 (4)

$$\Rightarrow k(\omega) = k_b(\omega)(1 + i\eta_E)^{-1/4} \left(1 - \frac{\omega_0^2}{\omega^2}(1 + i\eta_S)\right)^{1/4}$$
 (5)

$$k = k_r + ik_i \tag{6}$$

Sources

Beams on Elastic Foundation

Terje Haukaas



Parameter values

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Issues with the model

Standard values for UIC60 rail

Variable	Symbol	Value	Unit
Bending stiffness	ΕI	6.42	MN m ²
Mass pr. length	m	60	kg/m
Foundation stiffness pr. length	s	100	MN / m ²
Damping loss rail	η_{E}	0.02	
Damping loss foundation	η_{s}	0.1	

Sources:

Railway Noise and Vibration

D. Thompson



Wave number plot

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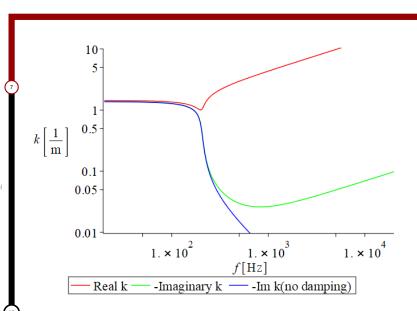
Sound power level

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Summary





Group and phase velocity

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$$v_p = \frac{\omega}{k} = \sqrt{\frac{EI}{m}} (1 + i\eta_E) k^2 + \frac{\omega_0^2}{k^2} (1 + i\eta_S)$$
 (7)

$$v_g = \frac{d\omega}{dk} = \frac{2}{k + \frac{\omega_0^2(1+i\eta_s)}{EIm(1+i\eta_s)}}$$
 (8)



Sound generation

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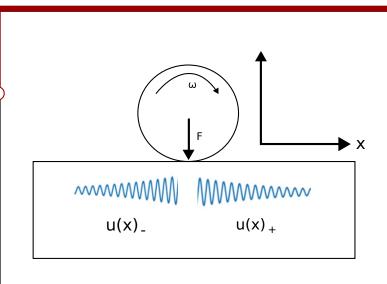
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Harmonic force

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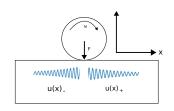
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Issues with the model



Harmonic force $F \exp(i\omega)$

$$U_{-}(x) = A_1 e^{kx} + A_2 e^{ikx}$$

$$u_{+}(x) = A_1 e^{-kx} + A_2 e^{-ikx}$$
 (10)

(9)

Solving with boundary cconditions:

$$u(x) = \frac{-iF}{4Flk^3} \left(e^{-ik|x|} - ie^{-k|x|} \right)$$
 (11)

Velocity amplitude

$$v(x) = i\omega u(x) \tag{12}$$



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Sound power

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Issues with the model

$$W = \rho_0 c_0 S \langle \bar{v}^2 \rangle \sigma = \frac{1}{2} \rho_0 c_0 \sigma \int_{-\infty}^{\infty} |v(x)|^2 dx$$
 (13)

- ▶ Density and speed of sound, S surface perimeter ($\approx 2\pi a$), velocity normal to surface $v(x) = i\omega u(x)$
- Radiation ratio σ, actual sound power to idealized case (plane waves)
- ► Cylinder

$$\sigma = \left(\pi |kaH_0^2(ka) - H_1^2(ka)|^2\right)^{-1} \approx \frac{\pi}{2}(ka)^3(ka << 1)$$

Sources:

Railwav Noise and Vibration

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Issues with the model

Insert

$$W = \frac{\rho_0 c_0 \pi (ka)^3 S \omega F}{16 E l k^3} \int_{-\infty}^{\infty} |\exp -ikx|^2 dx \tag{14}$$

Now at a distance x = x + d symmetric around x = 0

$$W(d) = \frac{\rho_0 c_0 \pi^2 a^4 \omega F}{8EI} \exp(2k_i d) \int_0^\infty \exp(2k_i x) dx$$
 (15)

$$W(d) = \frac{\rho_0 c_0 \pi^2 a^4 \omega F}{8EI} \exp(2k_i d) \left[\frac{\exp(2k_i x)}{2k_i} \right]_0^{\infty}$$
(16)

$$W(f,d) = -\frac{\rho_0 c_0 \pi^3 a^4 fF}{8k_i(f)EI} \exp(2k_i(f)d)$$
 (17)



Sound power level

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$$L(f,d) = 10 \log_{10} \left(\frac{W(f,d)}{W_0} \right)$$
 (18)

- ► Sound power reference level (0 dB)
- ightharpoonup L > 0 can be heard by humans, 40 dB whisper



Sound power level plot



Sound generation

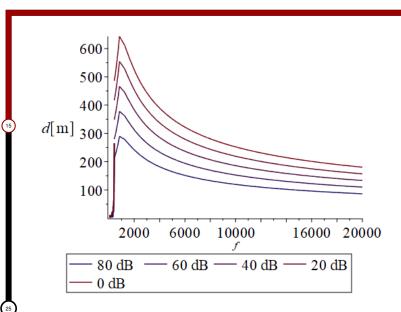
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TRAIN TIME!

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Issues with the model



Sources:

Thomas the tank engine



Nærum regional train

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Issues with the model

► RegioSprinter by Siemens-Duewag



Sources: Kurt Rasmussen



Microphone

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Spectrum

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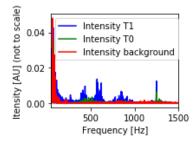
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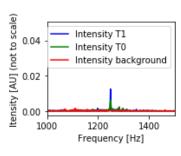
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Determining speed of train

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$$W(f_1, d) = -\frac{\rho_0 c_0 \pi^3 a^4 f_1 F}{8k_i(f_1) EI} \exp(2k_i(f_1) d)$$
 (19)

$$W(f_1, d - VT_1) = -\frac{\rho_0 c_0 \pi^3 a^4 f_1 F}{8k_i(f_1) EI} \exp(2k_i(f_1)(d - VT_1))$$
 (20)

$$\Rightarrow V = \frac{\ln\left(\frac{W(f_1, d - VT_1)}{W(f_1, d)}\right)}{2k_i(f_1)T_1} \approx 10.8 \pm 0.5 \frac{m}{s}$$
 (21)

- Or around 40 km/h (reasonably close to the operating speed at this route)
- ▶ This gives $\lambda = \frac{V}{f_*} \approx 8 \text{ mm}$



Determining distance to train

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- Calibration of microphone:
- $L_{meas} = 10 \log_{10} \left(\frac{CW_{meas}}{W_0} \right)$

$$W_{meas} = W(f_1, d) \Rightarrow d = 327m \tag{22}$$

- ▶ with $T_{arrive} = \frac{327m}{10.8m/s} = 31s$
- ► However the measured time was 26 s



Group and phase velocity

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Issues with the model

$$v_{\rho}(\omega) = \frac{\omega}{k} = \sqrt{\frac{EI}{m}} (1 + i\eta_E) k^2 + \frac{\omega_0^2}{k^2} (1 + i\eta_S)$$
 (23)

$$v_g = \frac{d\omega}{dk} = \frac{2}{k + \frac{\omega_0^2(1+i\eta_s)}{Elm(1+i\eta_E)}}$$
(24)

Phase velocity is the speed of sound. For f = 1249 Hz we get:

$$v_p(2\pi f) = 1613 \text{m/s}$$
 (25)

Which is a lot higher than the speed of sound in air $c_0 = 343$ m/s



Summary

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Summary

- ► The method seems to get a realistic speed and distance
- ▶ Difficult to check the roughness of the tracks/wheel
- Speed of sound a lot higher than the speed of sound in air which explain why the sound arrives in the track before the air.
 - ► Hard to actually hear with the human ear.



Issues with the model

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Summary

- ► Effect of sleepers/other damping types
- ► Two rails/ sound wave interference
- ► Train kinematic (constant speed)
- ► "Correct" values of constants
- Rail junctions
- ► And much more

Thanks for listening! / Any questions?





Appedix A: Wave number

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Damping: $\tilde{E} = E(1 + i\eta_E)$, $\tilde{s} = s(1 + i\eta_s)$ Assume solution of form

$$u(x) = U \exp i\omega t \exp -ikx \tag{26}$$

Insert and solve

$$\tilde{E}Ik^4 + \tilde{s} - m\omega^2 = 0 \Rightarrow k^2 = \pm \sqrt{\frac{m\omega^2 - s}{EI}}$$
 (27)

$$\Rightarrow k(\omega) = k_b(\omega)(1 + i\eta_E)^{-1/4} \left(1 - \frac{\omega_0^2}{\omega^2}(1 + i\eta_s)\right)^{1/4}$$
 (28)

$$k = k_r + ik_i \tag{29}$$

Sources

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Terje Haukaas