

Article

Numerical Fire Spread Simulation Based on Material Pyrolysis - An Application to the CHRISTIFIRE Phase 1 Horizontal Cable Tray Tests

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Abstract: A general procedure is described, to generate material parameter sets to simulate fire propagation in horizontal cable tray installations. Cone Calorimeter test data is processed in an inverse modelling approach. Here, parameter sets are generated procedurally and serve as input for simulations conducted with the Fire Dynamics Simulator (FDS). The simulation responses are compared with the experimental data and ranked based on their fitness. The best fitness was found for a test condition of 50 kW/m². Low flux conditions 25 kW/m² and less exhibited difficulties to be simulated accurately. As a validation step, the best parameter sets are then utilised to simulate fire propagation within a horizontal cable tray installation and are compared with experimental data. It is important to note, the inverse modelling process is focused on the Cone Calorimeter and not aware of the actual validation step. Despite this handicap, the general features in the fire development can be reproduced, however not exact. The fire in the tray simulation extinguishes earlier and the total energy release is slightly higher as compared to the experiment. The responses of the material parameter sets are briefly compared with a selection of state of the art procedures.

Keywords: CHRISTIFIRE; Fire Dynamics Simulator (FDS); pyrolysis modelling; shuffled complex evolution (SCE); high performance computing (HPC); fire propagation simulation; cone calorimeter simulation; cable tray fire simulation; SPOTPY; PROPTI

1. Introduction

In the fire safety engineering community, design fires are a frequently used tool when conducting fire risk assessments. The rigidity of the prescribed fire developments, e.g. hydrocarbon curve, is an obvious limitation. Ideally, the fire development could be simulated, based on the material of the objects involved, as well as ventilation conditions and energy distribution near the fire's location. One way to achieve this goal is to simulate material pyrolysis. Laboratory tests are utilised to support these simulation efforts, as in general the explicit measurement of the material properties is not feasible. There is an implicit hierarchy assumed, in which micro-scale combustion tests, like Thermo-Gravimetric Analysis (TGA) or Micro-Combustion Calorimetry (MCC), can be used to derive basic parameters to describe the pyrolysis reaction rates, for example parameters for an Arrhenius equation. Bench-scale tests, like the Cone Calorimeter, can be utilised to determine the thermo-physical parameters. Afterwards, these parameter sets can be used to simulate the fire development in a real-scale setup.

The fire safety engineering community has conducted quite some research within this field, e.g. [1–6]. Rogaume provides an overview over some of the challenges when trying to simulate material

32 pyrolysis in [7], also discussing some optimisation strategies for estimating more complex parameter
33 sets. One of the optimisation strategies is the employment of a shuffled complex evolutionary algorithm
34 (SCE) [8], which is relatively common in fire safety engineering [3,4,9–12] due to its performance [13]
35 and is also utilised in the work presented here. For other approaches, as much data is taken from
36 experiments as possible and pyrolysis models are built on it [14].

37 In this contribution, the focus is set on simulating fire propagation in horizontal tray installations,
38 based on pyrolysis of electrical cables. Fire behaviour of cable tray installations has been studied over
39 previous decades. Specifically, the nuclear industry dedicated resources to investigate cable tray fire
40 development on an international level. Some of the more visible projects have been the "Cable Heat
41 Release, Ignition, and Spread in Tray Installations during Fire" (CHRISTIFIRE) Programme [15,16]
42 carried out by the United States Nuclear Regulatory Commission (U.S. NRC), "Propagation d'un
43 incendie pour des scénarios multi-locaux élémentaires" (Fire Propagation in Elementary Multi-room
44 Scenarios – PRISME) [17] carried out by the Nuclear Energy Agency (NEA) of the Organisation
45 for Economic Co-operation and Development (OECD) or the "International Collaborative Project
46 to Evaluate Fire Models for Nuclear Power Plant Applications" (ICFMP) [18] carried out by the
47 Gesellschaft für Anlagen- und Reaktorsicherheit (GRS) gGmbH. Research is also conducted in more
48 general terms, like in the "Fire Performance of Electrical Cables" (FIPEC) project [19] initiated by
49 the European Commission. During these projects, different experiments at various scales have been
50 performed, not solely related to cable fires but also fire and smoke propagation in general.

51 Different strategies have been developed to model the fire development in cable tray installations.
52 Performance of several computational fluid dynamics (CFD) codes, to estimate fire development and
53 propagation in cable tray installations, have been compared in the past [20]. Based on Cone Calorimeter
54 data from the FIPEC project mathematical models for material pyrolysis were created [21]. A relatively
55 simple hand calculation model, "Flame Spread over Horizontal Cable Trays" (FLASH-CAT), was
56 developed primarily from data of the CHRISTIFIRE campaign. The FLASH-CAT model was picked up
57 in the frame of the PRISME programme [22] for trays mounted to a wall, where it was implemented
58 into a CFD code (CALIF3S/ISIS) and some parameters were adjusted, such that the model would
59 better recreate the PRISME data. A very similar setup, also from the PRISME programme, was used
60 where the Cone Calorimeter energy release rate was "painted" on a cable tray model, such that each
61 individual surface cell would follow the development of experimental data [23]. The release starts
62 after a certain material temperature of the cable sample was reached, that could be regarded as an
63 ignition temperature.

64 The principal inspiration for the work presented here, with respect to pyrolysis modelling with
65 the Fire Dynamics Simulator (FDS), is work conducted by Matala et al. [3] and Lautenberger et al. [24].

66 In general, a brute force approach is utilised here, for which it is assumed that either only few
67 parameters are known of the studied material, or cannot be transferred directly to the simulation. Thus,
68 all parameters that define a material in FDS, are worked upon during the optimisation process. This is
69 specifically the case for the residues, of which no information is available.

70 It should be noted that the overall hypothesis for this work is, that parameters are not directly
71 transferable from the experiment to the simulation (yet) and therefore an effective parameter set is to
72 be created, that yields a simulation response close to the data observed in the experiment. The assumed
73 reasons are manifold. An obvious reason is the insufficient spatial resolution of the individual cables,
74 especially in real-scale cases. Additionally, gaps in the understanding of the underlying processes and
75 their complexity may still lead to an incomplete set of models.

76 The presented work aims to describe a procedure, to create material parameter sets that allow
77 the simulation of fire propagation within FDS. It follows the concept that material behaviour can be
78 studied sufficiently well in the laboratory scale, and thus it allows for extrapolation into real-scale
79 scenarios. The foundation for this work is experimental data obtained by CHRISTIFIRE Phase 1 [15].
80 Cone Calorimeter tests are chosen as starting point, with a simplified model of the apparatus being
81 utilised in FDS. Employing a numerical optimisation scheme, material parameters are varied in the

82 simplified Cone Calorimeter simulation setup, such as to find the FDS response that is close to the
83 energy release rate data from the experiment. In an inverse modelling process (IMP) the experimental
84 data serves as target, while the material input parameters are adjusted for a simulation response to fit
85 the target. The best parameter set obtained during this process is then utilised in a real-scale cable tray
86 simulation setup. In a validation step, its performance in estimating the fire development is assessed.
87 Furthermore, the results of the presented procedure are compared to selected state of the art prediction
88 approaches.

89 This work is accompanied by a data repository [25] which contains simulation data like the input
90 files for FDS and PROPTI, data base files with the IMP results and the results of the simulations with
91 the obtained material parameter sets. For a brief description see section 5.

92 2. Materials and Methods

93 2.1. Experimental Data

94 The content of CHRISTIFIRE Phase 1 is briefly summarised below. During that experimental
95 campaign, a relatively large number of different cables has been subjected to fire tests of various scales.
96 Thus, data sets of the same cable, but under different conditions, are available. For the procedure
97 presented here, the focus, of which tests to use, was set on MCC, Cone Calorimetry and horizontal tray
98 installations in the open – the Multiple Tray Tests (MT). As argued below, cable 219 was chosen as the
99 sample, while cables from CHRISTIFIRE Phase 2 have not been considered as of now, however are
100 envisioned to be studied later on.

101 The choice fell on this specific cable, cable 219, because in contrast to the other cables:

- 102 • It showed good repeatability for the different incident heat fluxes during the Cone Calorimeter
103 tests.
- 104 • In the multiple tray tests, the individual trays were completely filled with the cable 219.
- 105 • In the multiple tray tests, the cables were neatly arranged to rows that extended over nearly the
106 whole tray width. This allowed the tray representation as a single solid slab in the simulation.

107 During Phase 1 of the experimental campaign, MCC tests had been conducted on the individual
108 cable components (insulator and jacket), using a Pyrolysis Combustion Flow Calorimeter (PCFC) [26].
109 Samples of about 5 mg from the plastic components were linearly heated up, to 600 °C at a heating rate
110 of 1 °C s⁻¹, within a nitrogen atmosphere. Data was determined, like the specific energy release rate per
111 mass, the mass loss, the amount of solid residue produced, heat of combustion, locations of the maxima
112 of the reaction rates, as well as their respective contributions to the overall decomposition process. This
113 also allows to calculate reaction kinetics parameters, modelled by employing an Arrhenius equation
114 per reaction.

115 Furthermore, the cables had been subjected to Cone Calorimeter tests. Up to three different,
116 constant radiative heat fluxes were imposed on the samples: 25 kW/m², 50 kW/m² and 75 kW/m².
117 The tests at 25 kW/m² and 50 kW/m² were mostly repeated three times, 75 kW/m² was performed
118 just once. Results of these tests are time dependent data series of the energy release rate (ERR) per unit
119 area.

120 Afterwards, real-scale tests in horizontal tray installations were performed. Tray racks were
121 placed on scales in a relatively large room, under an exhaust hood. Thus, they were considered as
122 burning in the open, with little influence from the surroundings. From one up to seven ladder-backed
123 trays were mounted above each other. The trays had a width of 0.45 m, a length of 2.4 m or 3.6 m and
124 were mounted with a vertical distance of 0.3 m. About 0.2 m below the lowest tray, in the centre, a gas
125 burner was positioned that provided an ignition source of 40 kW ± 5 kW. Energy release rates were
126 determined by means of oxygen consumption in the exhaust stream and by the mass loss rate.

127 Results from the Tube Furnace and the Radiant Panel test are neglected during the presented
128 work.

129 2.2. Inverse Modelling Process

130 Inverse modelling is used to obtain material parameter sets to describe its behaviour in the
 131 simulation. Data obtained from Cone Calorimeter tests serves as target for the inverse modelling
 132 process (IMP). An evolutionary, sometimes called genetic, algorithm is utilised to carry out the IMP.
 133 Specifically, the Shuffled Complex Evolutionary Algorithm from the University of Arizona (SCE-UA),
 134 developed by Duan et al. [8] was chosen. It is implemented in the scripting language Python and part
 135 of the framework "Statistical Parameter Optimization Tool for Python" SPOTPY [27]. The SCE-UA is
 136 used as provided via SPOTPY, without adjustments to the algorithm. The optimisation is conducted
 137 over multiple generations. The size of a generation Φ is determined by equation 1

$$\Phi = (2n_{\text{parameter}} + 1) \cdot n_{\text{complex}} \quad (1)$$

138 where $n_{\text{parameter}}$ is the number of parameters to optimise and n_{complex} is the number of complexes
 139 within a generation. The number of complexes was chosen to be equal to the number of parameters,
 140 which is the default setting of the implementation. In general, it is desirable to reduce the amount of
 141 optimisation parameters as the computational complexity, i.e. size of a generation, scales non-linearly
 142 with this value.

143 To assess the fitness of the different parameter sets, the root mean square error (RMSE) is calculated
 144 between the simulation response and the target data. An open-source Python framework, PRPOTI,
 145 serves as a communication interface between a simulation software, here FDS, and an optimisation
 146 algorithm [28–30].

147 As stated above, focus was set on cable 219, in order to streamline the overall process for creating
 148 the material parameter sets. In this text, the conducted IMP runs are labelled by indicating the target
 149 data (T), the fixed parameters (P), as well as the number of times sampling limits have been adjusted
 150 (L). The following labelling options are utilised:

- 151 • Indices for experimental conditions of the target data (T):

- 152 – a: 25 kW/m²
- 153 – b: 50 kW/m²
- 154 – c: 75 kW/m²

- 155 • Indices of fixed parameter (P):

- 156 – A: Arrhenius parameters (taken from the report [15])
- 157 – L1: layer thicknesses 2 mm (insulator and jacket)
- 158 – L2: layer thicknesses 4 mm (insulator and jacket)
- 159 – HT: heat of combustion from toluene (FUEL)
- 160 – HC: heat of combustion from the report [15]

- 161 • Indices of adjusted sampling limits (L):

- 162 – Successively numbered, starting by 0.

163 An example of an IMP run label is provided below:

$$T_{a,b,c}P_{A,L1}L_2, \quad (2)$$

164 where all three irradiance levels are used simultaneously as a joint IMP target, Arrhenius
 165 parameters are set to the data from CHRISTIFIRE, the layer thicknesses are set to 2 mm and it is
 166 the second time the parameter sampling limits are adjusted.

167 The purpose of the different targets is, to determine how well the resulting parameter sets
 168 represent the Cone Calorimeter experiments.

169 It was expected that the IMP runs which take multiple irradiance levels into account would yield
 170 more robust parameter sets over a wider range of external fluxes. In previous simulations [31] it was
 171 realised that specifically the heat flux of 25 kW/m² was difficult to recreate by FDS. Therefore, some
 172 IMP runs contained the 25 kW/m² case, while others were conducted without it.

173 2.3. FDS Modelling

174 The foundation for the FDS simulations is a cable model, consisting of a layered surface (SURF) of
175 different materials (MATL) and released combustible gaseous species (SPEC). All of this information is
176 combined, to form a simulation setup. In general, the simulation setup can be thought of as being the
177 representation of an experimental setup. In this concept, an experimental setup distinguishes not only
178 between individual apparatuses, like Cone Calorimeter or PCFC, but also their settings, e.g. external
179 heat flux or heating rate. In the following, individual aspects of creating the FDS input are discussed.

180 2.3.1. Cable Geometry

181 An electrical cable is an assembly of multiple components, conductors covered by an insulator,
182 wound together and surrounded by a jacket. Each component consists of different materials. In
183 general, the cables themselves cannot be resolved geometrically within the conducted simulations.
184 Therefore, they are represented as a flat obstruction of the flow field (OBST). To account for the cable's
185 composition, a layered SURF was chosen. Top and bottom layer contain the material model for the
186 jacket, while the insulator is embedded in between. Thus, three layers are used in total to represent the
187 cable. This is considered as necessary trade-off, in an effort to reduce the overall computing times.

188 In an earlier study [31], a copper layer divided the insulator, which lead to five layers. However,
189 during the inverse modelling process (IMP) the conductor thickness was repeatedly pushed to its
190 lower limit. This behaviour was also described by Matala and Hostikka [32]. Following that example,
191 the conductor material layer was removed, to speed up the IMP by reduction of parameters.

192 2.3.2. Chemical Reaction and Material Composition

193 As basic concept, pyrolysis is understood as the thermal degradation and consumption of a solid,
194 while gas(es) and solid residue(s) are produced. Based on the material temperature, the Arrhenius
195 equation describes the reaction rate [33]. This is basically the mass release rate of a gas from a solid,
196 which can be converted to the energy release rate, if the gas is combustible.

197 Even if a material appears to be homogeneous on a macroscopic level, microscopically it may
198 consist of a mixture of various components. These components are likely to decompose at different
199 temperature ranges. Micro-scale tests like MCC, allow to observe such a behaviour. As an example,
200 the plastic material of the cable 219 jacket is assumed to be homogeneous. The experimental data
201 shows two peaks for the cable jacket. This is interpreted as decomposition reactions of two different
202 components, which are represented as two materials (MATL) in FDS.

203 Even though, slightly more detailed information on the gaseous species are provided with the
204 report, toluene is used as surrogate fuel. Cables were tested in the Tube Furnace and yields of CO,
205 CO₂, HCl and soot are available. However, in previous simulations [31] heat transfer to the cable
206 surface was identified as a problem in the simulation. This was attributed to the poor spatial resolution
207 of the flame, due to the coarse fluid cells. Therefore, of the implemented species (SPEC) in FDS, one
208 was chosen that offers a high radiative fraction for the flame heat radiation – toluene. Its soot yield,
209 0.178 g g^{-1} , was taken from the SFPE Handbook [34].

210 Both cable components (insulator, jacket) are each allowed to form a solid residue in the simulation.
211 From the experiments, only the mass yield per component for the residues is reported. Parameters like
212 the (bulk) density, emissivity, thermal conductivity or specific heat are not available. Therefore, they
213 have been left to be determined by the optimiser and are effective parameters. Additionally, they act
214 as buffer material. This means, that the optimiser can potentially adjust the residue parameters when
215 it reaches limits for the remaining parameters.

216 2.3.3. Micro-Combustion Calorimetry

217 The MCC data is not directly part of the inverse modelling. Primarily, it gives an indication of
218 how many pyrolysis reactions are to be expected and are thus modelled. Furthermore, they are used to

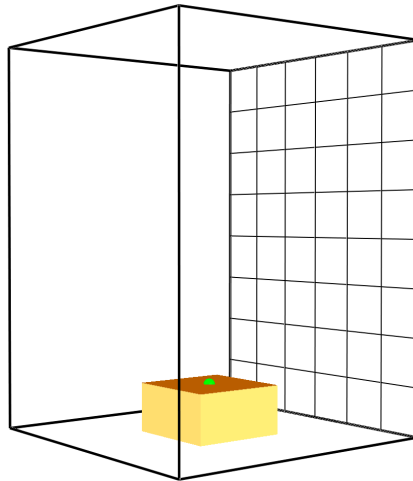


Figure 1. Geometrical representation of the improved Simple Cone Calorimeter (SCC) simulation setup in Smokeview. Darker area represents the model of the cable sample. Was also utilised for the MCC simulations.

219 determine if the Arrhenius parameters, that were found from the experimental data and provided in
220 the report, would emerge out of the IMP. In a further step, the pyrolysis parameters are fixed to the
221 ones obtained from the experiment (called "fixed Arrhenius"), in an attempt to reduce the demand for
222 computational resources.

223 The simulation setup of the MCC test was conducted by utilising the FDS functionality to run
224 only a TGA analysis with no gas phase simulation, i.e. TGA_ANALYSIS= . TRUE . .

225 2.3.4. Simple Cone Calorimeter

226 For a simple Cone Calorimeter model (SCC) the mesh size is set to an edge length of 47 mm
227 (cube-shaped cells), see figure 1. For comparison, in earlier work [3] larger cells were utilised (0.1 m
228 edge length). The smaller edge length provides a couple of benefits: it leads to a higher resolution of
229 the gas phase, it fits the size of the retainer frame window which has an edge length of 94 mm and it
230 is close to the target cell size of 50 mm envisioned for the later cable tray simulation setup. With the
231 increased resolution of the flow field the flame can be resolved more accurately. This in turn leads to
232 a better resolution of the radiative heat flux to the sample surface. Furthermore, sample and flame
233 are surrounded by two cells until the mesh boundary is reached and one fluid cell below the sample
234 surface level. This facilitates the formation of a more stable flow field.

235 2.3.5. Multiple Tray Simulation

236 During CHRISTIFIRE Phase 1, cable fire behaviour has been tested in horizontal cable tray
237 installations in the open. For this work Multiple Tray Test 3 (MT3) was chosen, because trays in this
238 test were solely filled with cable 219.

239 For the geometrical representation of the experimental setup of MT3 a uniform mesh is chosen.
240 The cells are cube-shaped, with an edge length of 50 mm, see figures 2a and 2b. This choice was mainly
241 driven by an attempt to provide a relatively high resolution, while not being too computationally
242 demanding. It is not based on a mesh sensitivity study. The computational domain is divided into six
243 meshes, as indicated by the differently grey-shaded boxes in figures 2a and 2b. The 50 mm cell size is
244 also close to the 47 mm cell size from the SCC simulations, which makes the material parameter sets
245 better transferable to the MT3 simulations. Furthermore, the 50 mm cell size allowed to have five fluid
246 cells between the trays and four between the burner and the lowest tray. In FDS 6.5.3 the principal
247 model for heat transfer within solids is one-dimensional heat conduction in the direction of the surface
248 normal. For this model to take the temperature of the opposite surface into account, when calculating

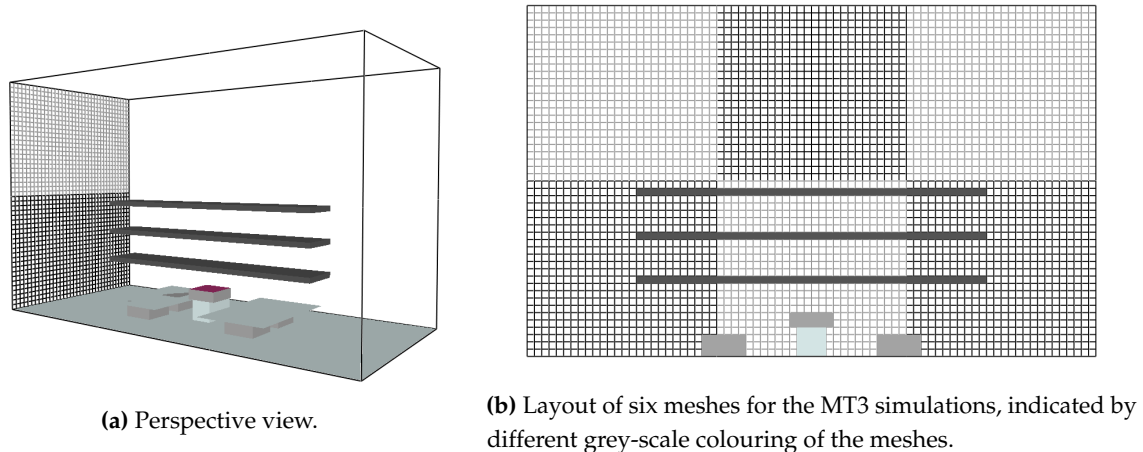


Figure 2. Geometrical representation of the MT3 simulation setup in Smokeview.

249 the material temperature, obstructions need to be one cell thick. Thus, the cable layer was created with
 250 the thickness of one cell.

251 The metal frame of the trays was neglected during the FDS simulations, because it could not be
 252 resolved by the chosen cell size.

253 In the experiment the cables were neatly arranged to rows that extended over nearly the whole
 254 tray width, see figure 3. Therefore, it was assumed, that the representation of the cables as a single, one
 255 cell thick, obstruction would be a reasonable geometrical model. In order to maintain a symmetrical
 256 flow field, the width of the tray models is reduced to 0.4 m, thus the burner could be positioned at the
 257 centre of the tray rack. The length of the slabs is set to 2.4 m, according to the experimental setup.

258 The gas burner is modelled by using a boundary condition where toluene is introduced into the
 259 computing domain (VENT). The same gaseous species, toluene, is used for the gas burner, as well as for
 260 the cable material. The burner's energy release rate per unit area is matched to its top surface (VENT),
 261 such that it releases 40 kW. The burner starts with the beginning of the simulation and is shut
 262 down after 600 s.

263 Also, the SURF that describes the top burner face, is assigned a surface temperature that is changed
 264 by employing the TMP_FRONT parameter and a RAMP. Within the first 100 s of the simulation, after a
 265 delay of 1 s, the surface temperature of the burner is linearly increased to 410 °C. Afterwards, it is
 266 kept constant for 501 s and ramped down linearly for 2599 s. The decrease starts 1 s after the burner is
 267 shut off, to somewhat account for a decaying flame, due to small amounts of remaining fuel in the
 268 piping between the valve and the burner (in the experiment). The prescribed surface temperature is
 269 purely guessed. However, it was deemed necessary to provide some model of the hot burner surface
 270 to support the gas flow field that would form, as well as the radiative interaction with the surrounding
 271 objects, specifically for the bottom face of the lowest cable tray.

272 2.4. Reference Calculations

273 To put the results of the IMP in context, three state of the art approaches are followed. This covers
 274 an alternative method to determine the model parameters, an approach with prescribed energy release
 275 rates, as well as an empirically based model for predicting the fire development within a horizontal
 276 cable tray installation.

277 For the first method the Arrhenius parameters are taken from the MCC results, provided in [15].
 278 This is referred to as "fixed Arrhenius" throughout this text, since they are not touched by the IMP in
 279 these specific cases. They are used in the same IMP setups, as discussed in section 2.2, namely the
 280 pre-exponential factor A and the activation energy E . The reaction order n was set to 1 (FDS default)
 281 and the heat of reaction to 1000 kJ kg^{-1} . The remaining parameters are still utilised for the IMP. A
 282 further adjustment is to set the layer thicknesses to 2 mm for each, jacket and insulator. This adjustment



Figure 3. Photograph of the cable arrangement in MT3, taken from the CHRISTIFIRE report [15].

283 is a general improvement to the original setup, since the layer thickness and the density thereof are
284 related and it leads to a reduction of necessary simulations to be conducted.

285 The second method determines parameters, that allow to map Cone Calorimeter test results to an
286 object's surface, basically a Cone Calorimeter paint. Different approaches for thermally thick samples
287 are summarised and discussed in [35, chapter 7]. In a recently published paper, this concept was
288 utilised in the context of simulating fire spread in cable tray installations [23]. In the work presented
289 here, Janssens' procedure [35] and the "Beji-Merci procedure" [23] are both used to compare the results
290 from the IMP against. Since no ignition times were reported with the Cone Calorimeter tests [15], they
291 are estimated from the energy release rate data, as described in the appendix D.

292 For the third method, the FLASH-CAT model is utilised. It was developed during the
293 CHRISTIFIRE campaign, based on its experimental data [15]. For all multiple tray tests, a calculation
294 was conducted and the respective results provided with the report. Since the model's results for MT3
295 were already available, they are extracted from the respective plot provided with the experimental
296 data.

297 3. Results

298 3.1. IMP Runs

299 3.1.1. Development of the Parameter Sets

300 Due to the large amount of individual simulations performed during the respective IMP runs, in
301 total more than one million, focus was set to the best parameter sets per generation of the SCE-UA.
302 These parameter sets were then used for different simulations, specifically to assess the performance in
303 the tray setup.

304 At first, the development of the fitness value of the best parameter set per generation for each
305 IMP run is shown in figure 4. It is given by the negative root mean square error (RMSE). Each IMP run

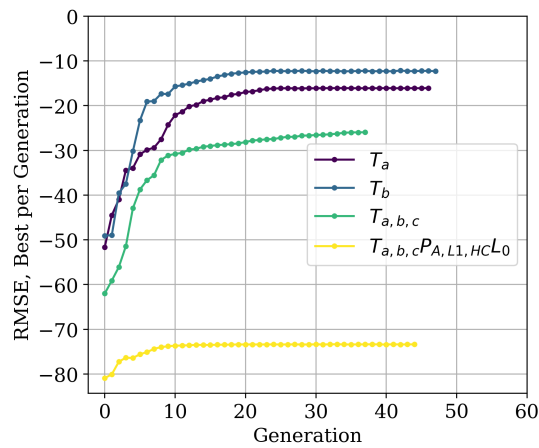


Figure 4. Development of the RMSE values, of the best parameter set per generation, for the IMP runs (T_* and $T_{a,b,c}^{PA,L1,HCL0}$).

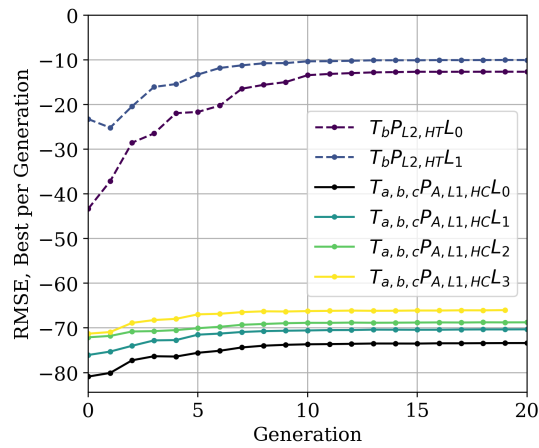


Figure 5. RMSE values for IMP runs with adjusted parameter sampling limits. Successive limit adjustments are marked with L_* .

306 starts out with a relatively large distance to the target. Within the first 10 to 15 generations the fitness
 307 improves notably and afterwards the rate of improvement decreases.

308 Appendix A provides some further information on the development of the individual parameters
 309 during the IMP.

310 3.1.2. Adjusted Parameter Limits

311 During the IMP runs, some parameters got stuck at their sampling limits. This behaviour is
 312 demonstrated in the appendix A.2, where the parameter development is summarised in ribbon plots.
 313 To improve on this, a test series is conducted in which the respective limits are shifted.

314 As general procedure, the sampling limits are adjusted by a percentage of the sampling range. In
 315 cases where the parameter is stuck at the upper limit the percentage is added, otherwise it is subtracted.
 316 The percentage is chosen arbitrarily, with a value of about 30 %. With these adjusted limits a new IMP
 317 run is conducted. The best parameter sets per generation for the adjusted limits are then also run
 318 through the whole stack of simulation setups. In general, it can be observed that the shifted sampling
 319 limits lead to an improvement of the fitness values. This is demonstrated by groups of the IMP runs
 320 T_b^{PL2,HTL_*} and $T_{a,b,c}^{PA,L1,HCL_*}$, see figure 5.

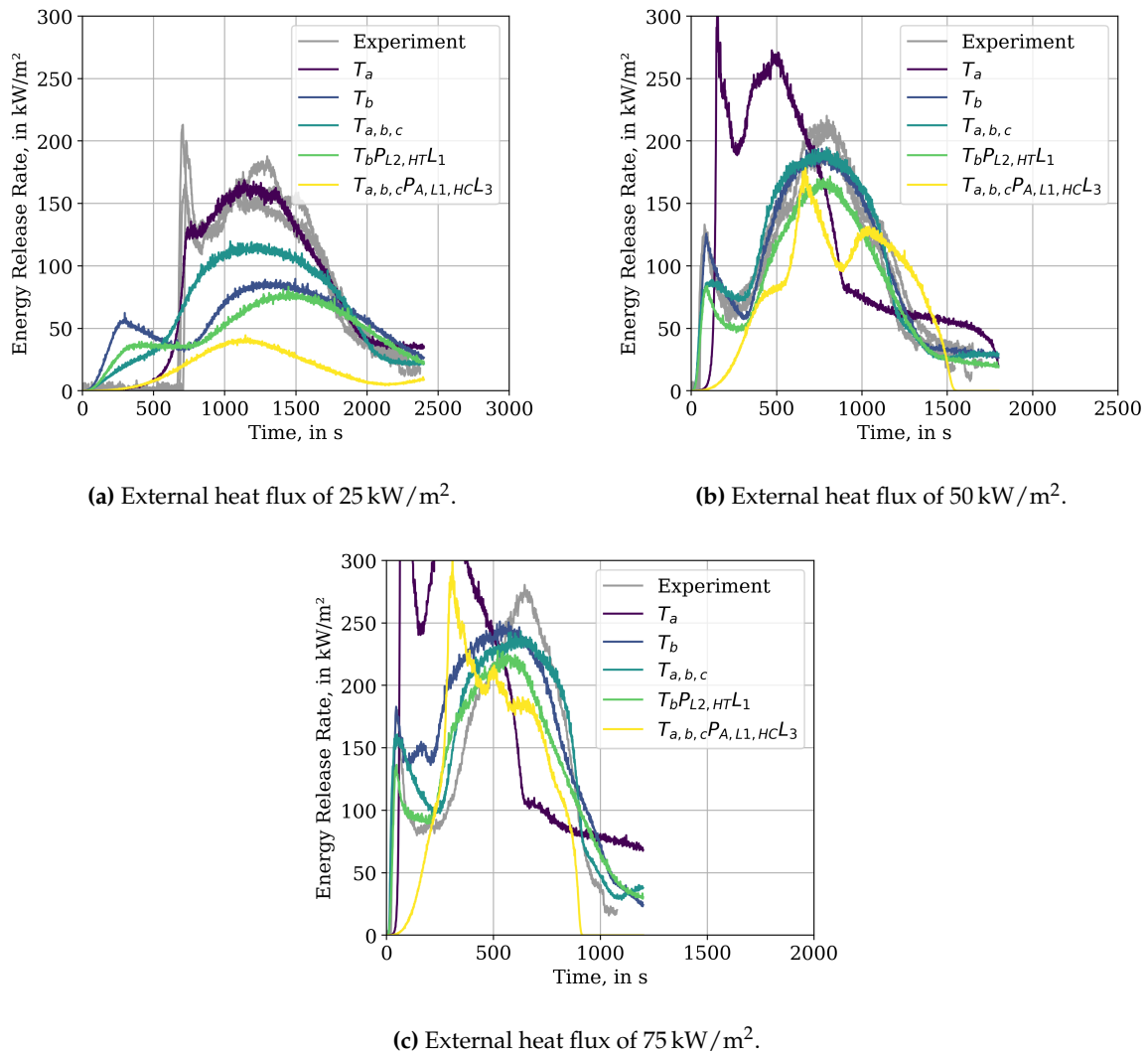


Figure 6. Comparison between energy release rates of Cone Calorimeter simulation, across global best parameter sets.

321 3.2. Cone Calorimetry Simulation Results

322 As stated above, not all of the IMP runs were utilising all of the different incident heat flux
 323 conditions as target. Despite this, after the conclusion of the IMP runs, all best parameter sets were put
 324 into simulation setups for all three conditions. This allows for a comparison of the parameter set's
 325 performance under all conditions and specifically compare more rigid (all three tests) to softer (one
 326 test) target setups. The following paragraphs describe the results shown in figure 6.

327 At first, the performance of the IMP results in relation to their respective targets is presented. It
 328 can be observed, that for an external radiative flux of 25 kW/m² only T_a (target 25 kW/m²) is able to
 329 represent the experimental data reasonably well, except for the first peak, see figure 6a. IMP run T_b is
 330 able to reproduce the experimental data well for its target of 50 kW/m², see figure 6b.

331 However, comparing the performance of both parameter sets against data that is not used as IMP
 332 target shows significant deviation. T_a overestimates the energy release for the other two external flux
 333 conditions (50 kW/m², 75 kW/m²) significantly. Even though, the general features of the experimental
 334 data can be reproduced. For T_b , in the 25 kW/m² condition, the energy release is over-predicted right
 335 from the start, the first peak reaches only a quarter of the energy release as the experiment and happens
 336 much earlier. The energy release of the second peak is about a factor of 2 lower as in the experiment.

337 For the 75 kW/m^2 data, the first peak is reproduced, as well as the second, however the intermediate
 338 section is over-predicted by a factor of about 1.5.

339 Visually, $T_{a,b,c}$ shows the best fitness overall three external fluxes, see figure 6. In the 25 kW/m^2
 340 case it is not able to capture the first peak. However, the energy release in the first about 700 seconds is
 341 lower, compared to T_b and the second peak is represented better. Under the 50 kW/m^2 condition, the
 342 first peak and valley are reproduced, but appear smoothed. The second peak is represented about as
 343 well as in T_b . With an external flux of 75 kW/m^2 , both peaks are reproduced and the valley is captured
 344 better as compared to T_b , see figure 6c.

345 The parameter set of $T_b P_{L2,HT} L_1$ shows slightly better performance in the Cone Calorimeter
 346 simulations compared to T_b . Under an external flux of 25 kW/m^2 the first peak is less pronounced than
 347 that of T_b , however the long delay visible in the experiment is also not reproduced. For the 50 kW/m^2
 348 case the performance is slightly worse as T_b , specifically the first peak is not captured well. In the
 349 75 kW/m^2 case, the first peak is still not resolved well, yet the valley is captured more closely than
 350 by T_b . In all three conditions the peak energy release is less than the the values reported from the
 351 experiments.

352 The state-of-the-art approach, with pre-determined Arrhenius parameters (here based on MCC
 353 data), layer thickness and heat of combustion, is represented by the $T_{a,b,c} P_{A,L1,HC} L_3$ IMP run. Here,
 354 all three experiments were used as target. The experimental data of the 25 kW/m^2 case could not be
 355 reproduced. It shows none of the distinct features and the peak energy release is a factor of about
 356 4 lower than observed in the experiment. A slightly better reproduction of the experiments could
 357 be achieved for 50 kW/m^2 and 75 kW/m^2 , however the energy release rate development diverges
 358 significantly.

359 All IMP runs show difficulties to reproduce the 25 kW/m^2 condition. While some show a
 360 two-peak-structure, none are able to reproduce the long delay to ignition, that can be seen in the
 361 experimental data.

362 For an overview of the performance of the best parameter sets per generation the reader is directed
 363 to the appendix C, where T_b is provided as an example.

364 3.3. Multiple Tray Simulation Results

365 For each best parameter set per generation of the IMP runs, a simulation in a MT3 setup was
 366 performed. The respective simulation results of the energy release rate are plotted and compared to
 367 the experimental data provided by the report [15].

368 Of the IMP runs, only T_b is able to reproduce the features of the ERR development, see figure 7.
 369 The first peak, around the time where the burner is switched off at 600 s, is over-predicted by a factor
 370 of about 2. After the burner is switched off, the ERR decreases by about 40 kW, which is similar to
 371 what is observable in the experimental data. In the simulation, the decrease is followed by a peak that
 372 overshoots the first peak by about 80 kW, which is again similar to the experimental data, however
 373 less pronounced there with about 60 kW. The last peak in the simulation response is a bit lower than
 374 the previous peak, while in the experiment the final peak is again about 50 kW larger than the one
 375 before. The last two peaks from the simulation overestimate the ERR of the experiment by a third, i.e.
 376 about 90 kW. The progression in the simulation is faster compared to the experiment. At the time the
 377 experiment reaches the second peak, the simulation has reached the third peak and starts to decrease.
 378 The different peaks are associated with the propagation of the flame to the next cable layer within the
 379 tray installation.

380 The parameter set of IMP run T_a leads to a massive over-prediction of the ERR and the features
 381 could not be reproduced. In contrast to that, IMP run $T_{a,b,c}$ is not able to cause any significant fire
 382 development and does not recover after the burner is switched off. Similar behaviour could be
 383 observed for the $T_{a,b,c} P_{A,L1,HC} L^*$ set, here only the last run, $*L_3$ is shown as example. With the given
 384 limits, the parameter sets are not able to achieve fire propagation after the gas burner is shut off.

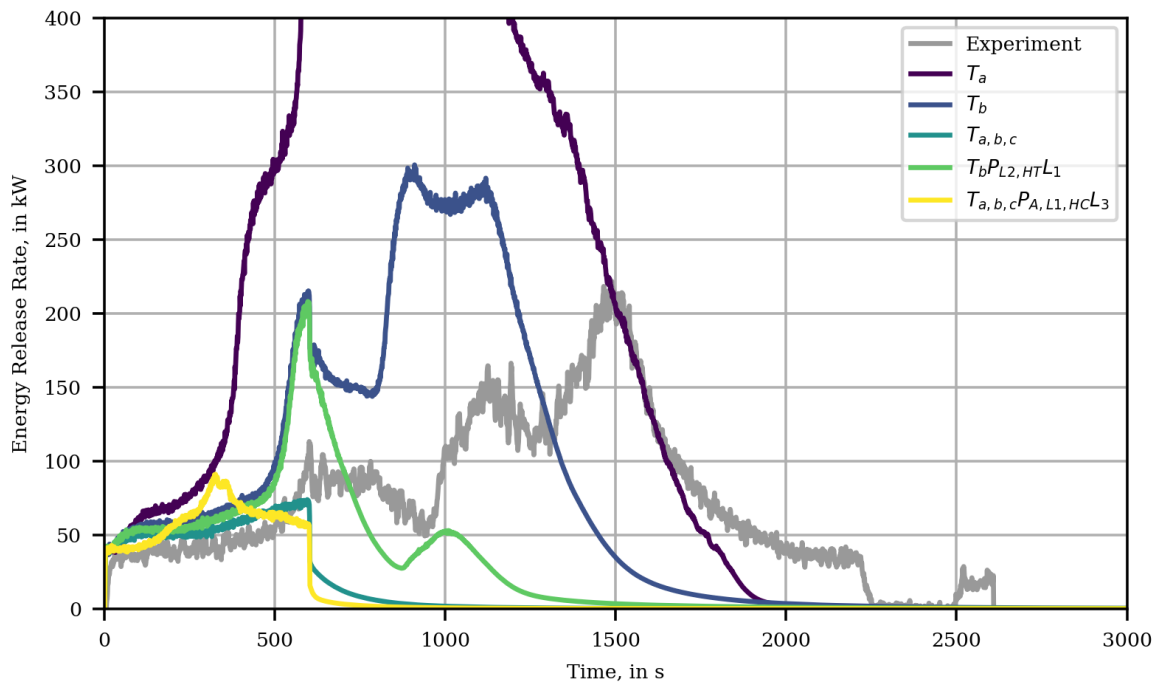


Figure 7. MT3 simulation results for a selection of the best parameter sets per IMP run, compared to experimental data. Peak for T_a at about (829.0 s, 856.2 kW)

385 $T_b P_{L2,HTL1}$ is able to recover briefly after the burner is shut off, but does not show meaningful
 386 fire development. It is notable that $T_b P_{L2,HTL1}$ starts from the same trajectory as T_b , but loses a lot of
 387 its ERR after the burner is cut. Interestingly, an early parameter set of this run is able to reproduce the
 388 experimental data in the MT3 setup better, as the best parameter set of the IMP (not shown here).

389 For the parameter sets that lead to a propagation it seems that an energy release of about 200 kW
 390 needs to be reached to get a sustainable position, able to cope with the burner shut-off. Yet, it may not
 391 be sufficient in all cases ($T_b P_{L2,HTL1}$).

392 3.4. Reference Calculation

393 At first, the results of the FLASH-CAT model are briefly outlined. The model results follow the
 394 experimental data, however it over-predicts the energy release rate, see figure 8. FLASH-CAT is also
 395 not able to resolve the features (peaks) of the experimental data. In contrast to the experiments, where
 396 the combustion is sustained up to about 2500 s, the FLASH-CAT model shows a duration of 5400 s.
 397 This leads to a higher total energy release than observed during the experiment, a factor of about 2.8.
 398 More details on this are provided in section 3.5.

399 Simulation responses for the different approaches of the Cone Calorimeter paint methodology are
 400 provided in figure 8. Across the different variations to the procedures, two clusters of results were
 401 generated one with high ERR and one with a low ERR. Due to their similarities, the figure shows only
 402 one representative for each cluster. For the low ERR cluster, after the burner is switched off the fire
 403 decays until extinction near 2200 s. Only the high ERR cluster shows a significant fire propagation and
 404 energy release, where the plot resembles the development of the 50 kW/m² Cone Calorimeter test that
 405 served as input for the RAMP. See appendix D for more details.

406 3.5. Total Energy Release in MT3 Setup

407 The total energy release (TER) for each best parameter set per generation was determined and
 408 compared to the experimental value, see figure 9. Furthermore, data from the FLASH-CAT model
 409 (extracted from [15]) and the reference calculations for the MT3 setup are added to the plot. The TER

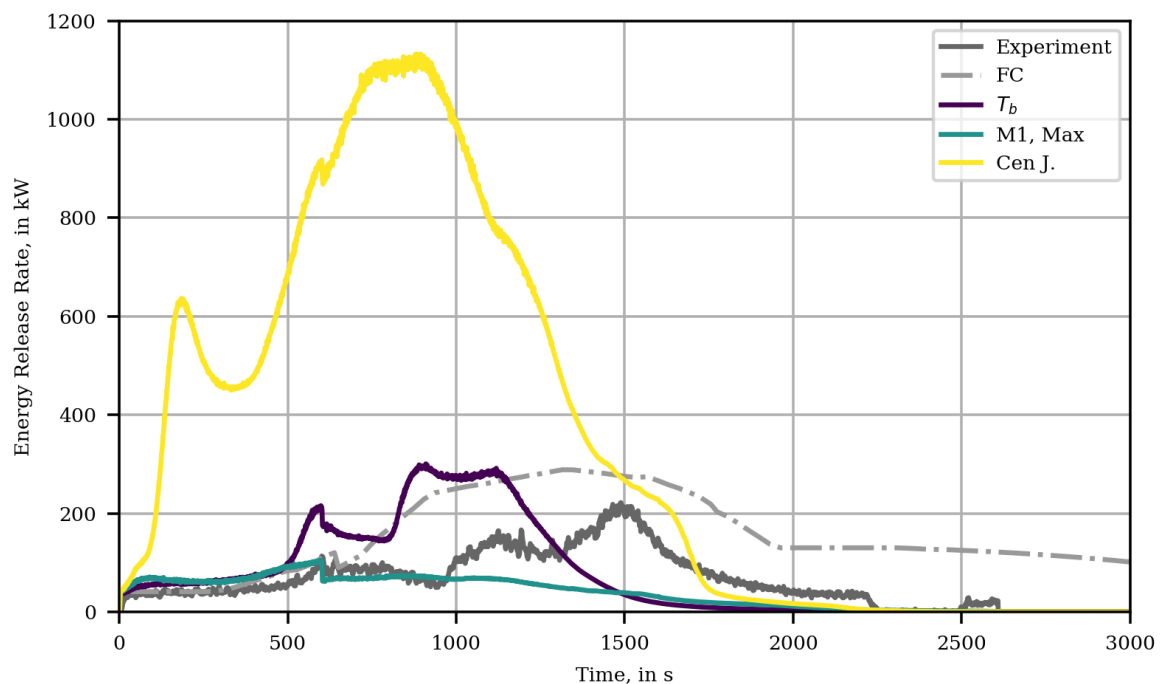


Figure 8. MT3 simulation results for different Cone Calorimeter paint methods and T_b results are provided for comparison. Label "M1" refers to a specific method of the "Beji-Merci" procedure [23], "J." refers to the Janssens' procedure [35].

410 for the experiment is calculated to be about 0.2 GJ, i.e. by integrating over the experimental data series.
 411 The FLASH-CAT model shows a significantly higher TER of about 0.5 GJ. Both values are provided as
 412 constant dashed lines in figure 9, to allow a comparison for the different model responses.

413 For the best parameter set of T_a the TER is slightly higher than the value from the FLASH-CAT
 414 model. T_b shows a TER which is slightly higher than the value from the experiment. The remaining
 415 IMP runs did not perform well during the MT3 simulation, which is indicated by TER values between
 416 0.05 GJ up to 1.2 GJ. For the reference calculations the lowest TER is about 1.0 GJ for the "Beji-Merci"
 417 procedure and the highest over 1.2 GJ for Janssens' procedure, overpredicting the TER by a factor of 2
 418 with respect to FLASH-CAT.

419 3.6. Micro-Combustion Calorimetry Simulations

420 Even though, the results of the MCC experiments were not directly used during the optimisation
 421 process, the obtained parameter sets are compared to this data. For each best parameter set two FDS
 422 input files are generated, which contained either the jacket or the insulator material. Utilising the
 423 TGA_ANALYSIS=.TRUE. functionality of FDS, a MCC simulation is conducted for the best parameter
 424 sets per IMP run. The results of the jacket material are presented in figure 10a, the data of the insulator
 425 in figure 10b. The simulation results are compared to the experimental and model data provided by
 426 the report [15]. For the given parameter sampling limits, none of the IMP runs is able to reproduce the
 427 experimental data as an emergent phenomenon. Simulations with fixed Arrhenius parameters are not
 428 shown here, since they, by construction, produce nearly the same result as the model parameters from
 429 the report [15].

430 Appendix C provides an exemplary overview over the performance of the best parameter sets
 431 per generation for IMP run T_b in the MCC simulation setup.

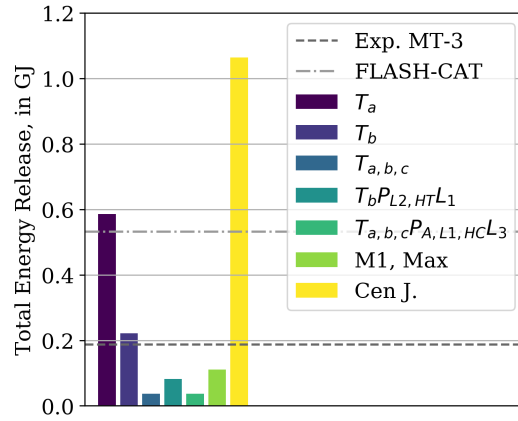
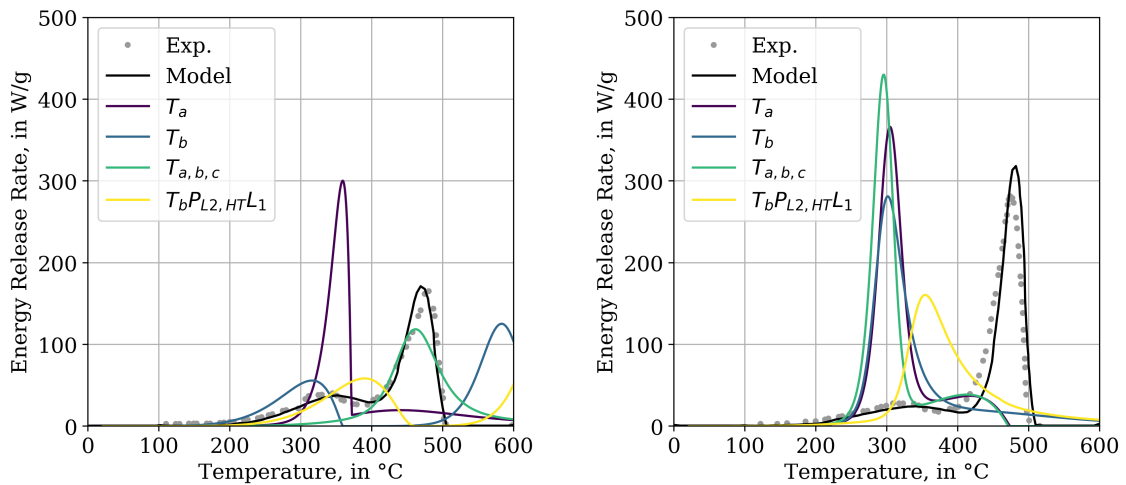


Figure 9. Total energy release of the MT3 simulations, compared with the experimental values and results from the FLASH-CAT (FC) model.



(a) MCC simulation response of the jacket material. **(b)** MCC simulation response of the insulator material.

Figure 10. Comparison of energy release rates of MCC simulation (TGA_ANALYSIS=.TRUE.) of cable 219, across best parameter sets of different IMP runs, against experimental (Exp.) and model data [15].

4. Discussion

It is obvious, that the IMP results presented here do not fit perfectly to the experimental data. Other models, which are strongly based on empirical data, can achieve a better fit, eventually by construction. Yet, their ability to forecast other scenarios is potentially limited. The work presented here is intended to provide a reference to a methodology for finding parameters that reasonably represent the fire behaviour of cable materials. In future work, the parameter transferability and limits of the methodology have to be investigated further.

Still, to the authors knowledge, the here presented work is the first time a successful material parameter based fire propagation simulation in a cable tray arrangement was achieved.

An important observation is the performance of IMP run $T_{a,b,c}$. Qualitatively, it performs best in the Cone Calorimeter simulations across all external flux conditions, see figure 6. Though, its fitness values show notably worse performance as for example T_b , see figure 4. On the other hand, the fitness assessment is different due to different target functions. Thus, it is not reasonable to compare fitness values across different IMP. In the cable tray simulations, however, no sustained fire development could be achieved after the burner was switched off. As of yet, we have no explanation for this behaviour. Especially, since the jacket material response in the MCC setup, it is relatively close to the experimental data.

It should be pointed out, that the here generated parameter sets are subject to different dependencies. For one, it can be shown that the performance is dependent on the fluid cell size and the solution only converges for higher resolutions, as discussed below. Furthermore, parameter set performance is also sensitive to computer architecture, software versions and operating systems, see appendix F.

It is curious that the Cone Calorimeter experiments can be reproduced relatively well, after optimisation. However, the extrapolation to the trays seems difficult. Similar behaviour can be observed when trying to extrapolate from the micro scale to the Cone Calorimeter.

Essential new aspects that have been considered for the overall process, i.e. parameter generation and cable tray simulations, are summarised below:

- The mesh resolution was increased in the optimisation step significantly, when compared with Matala's work [12].
- A wider array of material parameters was taken into account during the optimisation step, including the residues' parameters, and not only the reaction kinetics.
- With toluene a gas species was chosen, that produces more soot as compared to propane, which leads to a higher radiative fraction of the flame.
- The gas burner was assigned a (highly speculative) surface temperature profile with a slow decay to account for the heat up and feedback during the experiments.

As mentioned above, in order to become a reference for further investigations, an extensive appendix is provided. It contains all the considered approaches, and summarises the data sets and analysis methods provided in an online repository [25].

4.1. IMP

As a general conclusion, it can be demonstrated that the presented approach, using the SCE algorithm and the here formulated constrains, is able to find material parameter sets that are able to reproduce the Cone Calorimeter test responses within FDS 6.5.3, see figure 6. This is in agreement with findings reported by other researchers, e.g. [4,10,32]. The resulting parameter sets are shown to produce simulation responses that match their individual targets relatively well.

This outcome is not a general statement, as it is the result of the constrains set by the used model (FDS) and the chosen parameter sampling limits. Brief tests conducted to adjust the parameter limits seem to be beneficial to improve the overall fitness, as demonstrated in figure 5. Despite changes to the sampling ranges, not all parameters could be directed away from the limit they were stuck at, see

480 appendix A.2. This may be improved by more aggressive changes to the sampling ranges, but also by
481 taking the improved FDS input (e.g. layer thickness of 4 mm, no HEAT_OF_COMBUSTION from the cable
482 material) into account.

483 The generated parameter sets are to be regarded as effective parameters, in that they are not
484 necessarily realistic values. However, the sampling limits have been chosen to not allow values that
485 are too far away from what could be regarded realistic. Since no information was available for the
486 thermo-physical parameters of the residues of the cable components, they were basically used as buffer
487 material. Due to their parameters being part of the optimisation parameters, the algorithm is able to
488 indirectly influence the material decomposition, by changing e.g. the thermal inertia and the emissivity
489 of the sample.

490 One could imagine to follow a similar concept with the gaseous species, by introducing a "gaseous
491 buffer". On a simple level it could mean to mix an inert SPEC to the FUEL, like nitrogen, and have the
492 algorithm be able to adjust the fraction. However, to cover the initial delay for low flux conditions
493 better, it may be useful to introduce more gas mixtures, e.g. for each cable component, that are
494 associated to the pyrolysis reactions, as discussed by Matala [3, publication 4]. This requires that more
495 detailed information on the composition of the released gas mixture is available. Otherwise this model
496 would only be as arbitrary as any other. The CHRISTIFIRE Phase 1 report [15] provides information on
497 yields of selected gaseous components from tube furnace tests. Even though this information was not
498 used in the study presented here, to maintain consistency with the selected surrogate fuel of toluene,
499 the argument can certainly be made that the yields alone are not sufficient. Primarily, because they
500 represent the average value during steady-state conditions. This makes it difficult to connect them
501 with the changing temperature profiles present in the other setups, like MCC or Cone Calorimeter.
502 More detailed data-time series would be specifically necessary, when it is to be attempted to connect
503 the mass loss of the sample to the release of gaseous species and the resulting formation of a flame.
504 This is in contrast to the approach followed here, where only the energy release of the flame was
505 considered and the path to the formation of the flame was mostly ignored.

506 As stated above, the IMP yields good parameter sets for reproducing the Cone Calorimeter results.
507 Thus, it seems that the interaction/relationship of the heat transfer with the pyrolysis processes can be
508 reproduced sufficiently well. In order to individually check the validity of the pyrolysis process, the
509 MCC test data is utilised. Yet, the Arrhenius parameters gained deviate from the ones reported from
510 the experiment, despite expected otherwise. To represent the pyrolysis better, one could attempt to use
511 the Arrhenius parameters directly, as obtained from the experiments, i.e. not considered during the
512 optimisation. However, with the fixed Arrhenius parameters the simulations of the Cone Calorimeter
513 and the multiple tray tests do not yield a reasonable response. This could only be achieved when
514 the IMP was given access to the whole ensemble of the material parameters, therefore controlling
515 the interaction globally. Additionally, it is known, that the oxygen concentration in the gas phase
516 around the sample can not be neglected and might significantly influence the material decomposition
517 behaviour [36]. Thus, MCC tests where the sample is heated in a nitrogen atmosphere might not be
518 representative to the conditions during Cone Calorimeter tests. This could also be a cause as to why
519 the generated material parameters are not transferable from the Cone Calorimeter simulations to the
520 MCC simulations.

521 This leads to the conclusion that these processes and their interactions are not sufficiently well
522 reproduced. It might be caused by incomplete modelling approaches, e.g. formation of bubbles or
523 cracks, the crude geometrical representation of the cables, i.e. layered, or a combination thereof. This
524 drawback is compensated by producing effective parameters, including the Arrhenius parameters.

525 Having said the above, it should be pointed out that the HEAT_OF_COMBUSTION parameter for the
526 MATL was misunderstood, thus it ended up being part of the pool of optimised parameters, originally.
527 The project had progressed too far when this was realised and due to the computational demand it was
528 decided to not re-run all IMPs. Thus, the mass fluxes, leaving the solid and entering the gas phase, are
529 not consistent. Yet, the energy release is consistent. Since the optimisation target has been the ERR, this

530 misunderstanding has no direct consequences downstream, i.e. Cone Calorimeter and tray setup, as
531 long as gained parameter values for the Arrhenius parameter and the HEAT_OF_COMBUSTION are used
532 together. This may potentially be the reason why the individual values of the Arrhenius parameters
533 alone are not representing the MCC results.

534 4.2. Gas Phase in Cone Calorimeter Simulations

535 Compared to previous work [3,31], where the sample surface was resolved with only a single
536 fluid cell, the resolution is increased here. The simplified Cone Calorimeter model has four fluid cells,
537 two by two, to resolve the sample surface instead. This leads to smoother energy release profile, as
538 well as a higher resolution of the flame. Even though the energy release rates seem to converge for
539 higher resolutions, significant grid-dependence can be observed, specifically for the determined heat
540 fluxes towards the sample. The heat flux profiles exhibit a more pronounced development, which
541 follow the profile of the energy release. This is attributed to a higher resolution of the flame and thus
542 the improved calculation of the radiative and convective heat fluxes. It highlights the need, to take
543 the flame already into account during the IMP as an emergent phenomenon, as opposed to a static
544 prescribed radiative flux. Further information on the grid sensitivity is available in appendix B and
545 appendix E.1.

546 4.3. Multiple Tray Setup

547 In general, the observed grid-dependence in the SCC simulations is not expected to have a
548 significantly negative impact on this study. Mainly, because the cell size during the optimisation step
549 is close to the cell size in the MT3 validation setup.

550 It is further interesting to note, that the highest heat fluxes in the tray simulation setups are
551 occasionally about a factor of 2 higher than the imposed conditions in the Cone Calorimeter tests, see
552 figure A11b. It is not quite clear if this is an "artefact" out of the simulation, or actually observable
553 in real tests, since no data was available to compare this observation to. It also raises the question, if
554 higher external fluxes in the Cone Calorimeter tests might be necessary to be added to the existing
555 stack of tests. Future (cable) tests could look into this kind of behaviour.

556 4.4. Low Heat Flux Condition

557 This section discusses the impact of the low heat flux condition, i.e. about 25 kW/m^2 and less,
558 in the Cone Calorimeter and cable tray simulation setups. With the given FDS simulation setup, it
559 is very difficult to reproduce the 25 kW/m^2 Cone Calorimeter tests, see figure 6 for example. This is
560 due to the ignition delay observed during the experiments. In IMP run T_a (25 kW/m^2 target), the fast
561 increase in the energy release at ignition cannot be reproduced. It is smoothed, i.e. the energy release
562 starts long before the ignition and the first peak is not present.

563 This primarily seems to be related to how FDS (here version 6.5.3) handles the gas phase
564 combustion. For any given cell, combustion is allowed to occur even if the fuel concentration is
565 very low. In reality however, the concentration of fuel gas might not be sufficient for combustion to
566 ensue and just leave the sample surface. This difference leads to a non-zero energy release very early
567 on in the simulation. Additionally, the radiative heat feedback is increased, which in turn leads to
568 faster increase of sample temperatures and therefore quicker release of more combustible gas. As a
569 result, a smoother transition from the pre- to the post-ignition phase is observed, in contrast to the
570 rapid step-like increase observed in the experiments.

571 In this simulation campaign, the released species (SPEC) was only combustible gas, while it is
572 likely, specifically for cables designed to be fire retardant, that inert gaseous species are released first.
573 Thus, it may make it difficult for FDS to deal with the delay visible in the 25 kW/m^2 test responses.
574 The approach of Matala, publication 4 in [3], of providing a more detailed decomposition model, which
575 may also release inert gaseous components, could be a solution.

576 Keeping this shortcoming in the modelling in mind, only the parameters of IMP T_a are able to
577 reproduce the low flux case in the Cone Calorimeter simulation reasonably well. However, applying
578 this parameter set to other, higher, flux conditions in the Cone Calorimeter leads to a significant
579 over-prediction of the energy release rate and fire development. Similar behaviour can be observed in
580 the cable tray simulation setup, where the fire development is significantly more severe, with respect
581 to a faster progression and a higher peak energy release rate that is about 4 times higher as in the
582 experiment, see figure 7.

583 IMP runs with the remaining experiments as target show difficulties to reproduce the low flux
584 condition. Even though, the parameters of the 50 kW/m^2 are able to reproduce the 75 kW/m^2
585 behaviour relatively well and vice versa. Under an incident flux of 25 kW/m^2 the energy release
586 starts notably earlier with a larger magnitude, as compared to the 25 kW/m^2 IMP parameters and the
587 experiment.

588 The importance of the 25 kW/m^2 case is not completely clear. On one hand it seems that it does
589 not matter too much, when the higher fluxes can be represented well, i.e. by using the higher fluxes as
590 IMP targets. Even though, the energy release rate in the MT3 simulations is notably overestimated
591 around the time of the burner cut-off. On the other hand, assuming a fire propagating along a
592 horizontal fuel bed, every surface element in front of the flame needs to "pass through" a low flux
593 regime to be ignited and differences here should influence the overall speed of the fire propagation.
594 The latter part is demonstrated in figure A11b in appendix E.2, where it can be seen that large areas of
595 the cable tray experience low heat flux conditions over the whole course of the simulation.

596 One way to provide clarification, could be to investigate the actual heat flux levels to be expected
597 in large scale configurations during fire experiments. For one to determine if the observed surface flux
598 levels in FDS are sensible and also to determine if Cone Calorimeter experiments with higher fluxes
599 are necessary in the optimisation process.

600 4.5. Geometrical Representation of Cables

601 The one-dimensional heat conduction model utilised in the presented simulations could influence
602 the fire development. Specifically, when considering the absence/presence of the copper conductor.
603 This conductor may serve as a heat sink near the fire seat and pre-heating the insulation material further
604 away from the fire [21]. In the Cone Calorimeter simulation setups this might not be of too much
605 importance, due to the small cable pieces and the relatively uniform heat up of the exposed surface. In
606 that case, it may mainly behave as a heat sink and should be covered by the effective parameter set
607 derived from the IMP. It should be more influential for the cable tray simulations. However, in the
608 given setups it was not possible to resolve the necessary length scales. Investigations of this behaviour
609 may become possible in the near future, due to the three-dimensional heat conduction model added to
610 newer FDS versions, as well as new functionalities allowing for unstructured solids (i.e. GEOM name
611 list group).

612 It has been pointed out, e.g. [19,23], that cables are not necessarily put in a tray such that they
613 form a continuous slab. Often, they are somewhat loosely packed or combined into bundles. These
614 structures may however be in the sub-grid scale. Matala investigated if individual cables could be
615 modelled in the sub-grid scale, by utilising cylindrical particles [23]. It would be interesting to see
616 how the new unstructured solid method (GEOM) method is able to actually resolve the cable models
617 geometrically.

618 4.6. Reference Calculations

619 As described earlier, the application of both reference methods, Cone Calorimeter paint and
620 FLASH-CAT, did not yield satisfying results in the here investigated setup. In the following paragraphs
621 possible reasons are discussed.

622 For the Cone Calorimeter paint approach not only the energy release rate data is needed, but
623 also the thermal parameters ($k\rho c_p$), to determine an effective surface ignition temperature. These

624 parameters were not available for the plastic material, thus density and heat capacity were guessed,
625 based on material properties found on a web page for material properties. This may hamper the
626 comparability between this work and [23], but should be representative for an approach a practicing
627 fire safety engineer might pursue. In the work presented here, the cables have been modelled as
628 continuous slabs, instead of "poles", which may be one reason why Beji and Merci could obtain more
629 convincing results in some of the *a posteriori* simulations in [23].

630 The predicted duration of the fire in the FLASH-CAT approach is based on the combustible mass
631 per tray and a flame front propagation speed [15]. Ignition of the individual trays is controlled as a
632 timed sequence, based on the experimental findings. In contrast to the observation in the experiments
633 and also the simulations presented here (T_b), eventually all trays get involved and are consumed
634 completely. This seems to be the main cause for the much longer duration (figure 8), as well as the
635 larger magnitude of the total energy release (figures 9).

636 4.7. Robustness of the Model Parameter Sets

637 Cable 219 was deliberately chosen to be investigated, because it (a) showed the best reproducibility
638 of the repeated Cone Calorimeter tests, (b) the cable was used in trays containing the same cable and
639 (c) the individual cables were arranged in rows which made it easier to represent in the simulation.

640 Other cable tray tests in the campaign showed a more severe fire development, but contained a
641 mixture of various cables per tray and between trays. Having gained confidence that the proposed
642 procedure can generate useful material parameter sets, more cables are to be investigated in future
643 work, e.g. cables 220 and 701.

644 These two show similar behaviour in the Cone Calorimeter. Furthermore, they were used in
645 tray experiments with mixed cables, where individual trays were filled with a single cable type of
646 220 or 701, but no mixture of cables within a tray. Thus, it might be easier to reproduce, than cable
647 mixtures within trays. It would also make MT8 and MT11 accessible, that showed a much severe fire
648 development, with peak energy release rates of about 800 kW. In the long run, larger scale simulations
649 need to be performed, like the corridors and the vertical shaft setups, from phase 2 of the experimental
650 campaign [16].

651 With the investigation of further cables, the robustness of the method presented here can be
652 evaluated. More robust material parameter sets allow investigation of the influence of parameters
653 like distances, number of trays and burner energy release rates and times, cable tray arrangements in
654 corridors can be investigated. This would also set goals for future cable testing campaigns, to validate
655 the simulation results.

656 4.8. Design Proposals for Future Experiments

657 For further work on parameter optimisation, to simulate fire propagation, it is important to
658 have access to data from bench-scale, as well as well documented large scale fire experiments. The
659 former is needed during the optimisation, while the latter is necessary to validate the parameter
660 set's performance, which is of specific importance. Simulations focusing on micro- and bench-scale
661 alone, as well as neglecting the gas phase reactions seem not to be sufficient to replicate the large
662 scale fire behaviour. To fill gaps within the existing body of experimental data, future test campaigns
663 should start from the CHRISTIFIRE campaign design as a base line. It is suggested by the authors to
664 focus on one single cable to perform all tests with. Tests in the open offer good cases for simulation
665 software, while presenting a rather modest need for computing resources. However, real world
666 installations are often found in confined spaces, close to walls and ceilings. It is therefore necessary to
667 perform similar experiments as the corridor setup presented in phase 2 [16]. Experimental setups with
668 smaller wall sections connected to the trays in the open, similar to FIPEC [19] or like the ones used by
669 Zavaleta et al. [37], could serve as an intermediate step.

670 Information of peripheral conditions, like material data of surrounding materials, distances to
671 walls (laboratory size/footprint), ventilation conditions, surface temperatures of burner, floor and

672 other surfaces around the test should be recorded as well, to be able to create more comprehensive
673 models.

674 5. Data Repositories

675 During the course of the IMP runs, and the following analysis of the results, an extensive amount
676 of data was produced. Aiming to allow other researchers to gain a better insight into to whole work,
677 we provide public access to most of the data. Thus, this paper is accompanied by publicly available
678 online data repositories. A summary repository is hosted via Zenodo [25]. It contains only the data
679 necessary to reproduce the figures shown here, like the `propt_i_db.csv` or `*_hrr.csv` files, but not the
680 full simulation results. Also, all input files for FDS and PROPTI are provided, as well as the target data.

681 Furthermore, Jupyter notebooks are provided with the respective repositories. These notebooks
682 are used to process the results from the IMP's and provide an overview by creating various plots.
683 Some are used to guide investigations on the parameter sets, by allowing to create new FDS input files
684 from within the notebooks, as well as presenting the new results within the same notebook afterwards.

685 From the summary repository at Zenodo, a link will lead to a more comprehensive repository,
686 hosted by the Forschungszentrum Jülich. It contains the full FDS simulations that were created during
687 analysing the IMP results, such that they can be loaded into SmokeView for further study.

688 **Author Contributions:** Tristan Hehnen [1], Lukas Arnold [2], Saverio La Mendola [3]

- 689 ● conceptualization: [1], [2] and [3];
- 690 ● methodology: [1], [2] and [3];
- 691 ● software: [1] and [2];
- 692 ● validation: [1];
- 693 ● formal analysis: [1];
- 694 ● investigation: [1];
- 695 ● resources: [1], [2] and [3];
- 696 ● data curation: [1];
- 697 ● writing—original draft preparation: [1], [2] and [3];
- 698 ● writing—review and editing: [1], [2] and [3];
- 699 ● visualization: [1];
- 700 ● supervision: [2] and [3];
- 701 ● project administration: [1], [2] and [3];
- 702 ● funding acquisition: [1]

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713 study; in the collection, analyses, or interpretation of data; in the writing of the manuscript, or in the decision to
714 publish the results.

715 Abbreviations

716 The following abbreviations are used in this manuscript:

717

FDS	Fire Dynamics Simulator
SCE	Shuffled complex evolution algorithm
TGA	Thermo-Gravimetric Analysis
MCC	Micro-Combustion Calorimetry
CHRISTIFIRE	Cable Heat Release, Ignition, and Spread in Tray Installations during Fire experiment campaign
U.S.NRC	United States Nuclear Regulatory Commission
PRISME	"Propagation d'un incendie pour des scénarios multi-locaux élémentaires" (Fire Propagation in Elementary Multi-room Scenarios)
NEA	Nuclear Energy Agency
OECD	Organisation for Economic Co-operation and Development
ICFMP	International Collaborative Project to Evaluate Fire Models for Nuclear Power Plant Applications
GRS	Gesellschaft für Anlagen- und Reaktorsicherheit gGmbH
FIPEC	Fire Performance of Electrical Cables
CFD	Computational fluid dynamics
718 FLASH-CAT	Flame Spread over Horizontal Cable Trays
CALIF3S/ISIS	Computational fluid dynamics software
IMP	Inverse modelling process
MT	Multiple Tray Tests
PCFC	Pyrolysis Combustion Flow Calorimeter
ERR	Energy release rate
SPOTPY	Statistical Parameter Optimization Tool for Python
PRPOTI	Open-source Python framework that serves as a communication interface between a simulation software and an optimisation algorithm
RMSE	root mean square error
SCC	Simple Cone Calorimeter simulation setup
MT3	Multiple Tray Test 3
TER	Total energy release
FZJ	Forschungszentrum Jülich
JURECA	Jülich Research on Exascale Cluster Architectures (supercomputer)

719 Appendix A. Parameter Limits

720 An overview over the utilised parameters is provided in tables [A1](#) and [A2](#). These tables also
721 contain experimental data provided, which is limited to the MCC results [[15](#)], apart from some guessed
722 values for the reaction orders, heats of reaction, as well as layer thicknesses.

723 Appendix A.1. Ribbon Plots

724 In order to summarise the development of each parameter, over the course of the IMP, ribbon plots
725 were created, see figure [A1](#). As an example, the jacket layer thickness development of T_b was chosen
726 to illustrate how the ribbon plots are produced. On the left side of figure [A1](#), a scatter plot provides
727 an overview of each individual parameter value for each repetition (x-axis), within its sampling
728 range (y-axis). The points are plotted with a slight transparency, to indicate where most of them
729 are accumulated. In the centre plot, a histogram is presented that contains the information over the
730 sampling range. This is further compressed, by creating a heat map ribbon of the histogram, shown on
731 the right hand side. Due to the binning necessary for the histogram, all parameter sampling ranges are
732 immediately normalised. Thus, all parameter ribbon plots of a single IMP run can be stacked together
733 horizontally. Furthermore, $y=0$ then shows the lower limit of the respective sampling range, while $y=1$
734 shows the upper limit, as can be seen in the subsequent plots in figure [A2](#).

735 With the ribbon plots, the effective development of the parameters during the IMP can be observed.
736 Within the given simulation setups and parameter ranges, some parameters are forced to the limits of
737 the respective sampling ranges. As an example, the ribbon plot of IMP run T_b is shown in figure [A2b](#).

Table A1. Overview of the optimisation parameters of the IMP runs for the insulator. Data from the experiment is provided, if available. Layer thickness has been projected from a circular to a rectangular cross section to account for the layered representation in FDS. From the best parameter sets, over all IMP runs, the minimum (IMP Min.) and maximum (IMP Max.) values are provided. The parameter sequence is the same as for the ribbon plots, simply skipping parameters that are not used. (Note: values labelled with "*" are guessed and/or FDS default.)

"Physical" Parameter	Experiment	IMP Min.	IMP Max.	Unit
Insulator Thickness	1.97e-03 *	3.46e-03	4.99e-03	m
Emissivity	unknown	3.24e-01	6.83e-01	-
Density	1.18e+03 *	1.02e+03	1.15e+03	kg/m ³
Conductivity	unknown	2.05e-01	2.30e-01	W/(m K)
Specific Heat	unknown	1.49e+00	1.72e+00	kJ/(kg K)
Heat of Combustion	3.26e+04	3.89e+04	4.15e+04	kJ/kg
Insulator Reaction A				
Pre-exponential factor	3.99e+01	3.62e+01	4.19e+01	1/s
Activation Energy	4.10e+04	4.24e+04	5.00e+04	kJ/kmol
Reaction Order	1.00e+00 *	5.02e-01	2.08e+00	-
Heat of Reaction	1.00e+03 *	2.54e+02	6.08e+02	kJ/kg
Insulator Reaction B				
Pre-exponential factor	4.50e+20	4.26e+20	5.26e+20	1/s
Activation Energy	3.15e+05	2.36e+05	2.60e+05	kJ/kmol
Reaction Order	1.00e+00 *	5.07e-01	3.65e+00	-
Heat of Reaction	1.00e+03 *	4.84e+02	7.62e+02	kJ/kg
Insulator Residue				
Density	unknown	4.46e+02	6.12e+02	kg/m ³
Conductivity	unknown	1.93e-01	2.00e-01	W/(m K)
Specific Heat	unknown	6.90e-02	8.60e-01	kJ/(kg K)
Emissivity	unknown	2.98e-01	6.50e-01	-

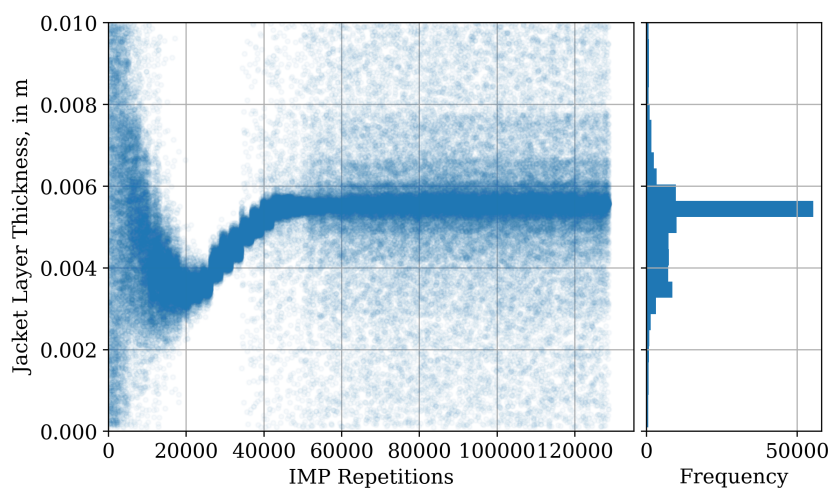


Figure A1. Demonstration of how parameter information is condensed. Left side shows the parameter development for the jacket layer thickness during T_b . The centre plot shows the frequency of the sampled parameters, distributed over 25 bins. The colour bar at the right side is a heat map ribbon of the histogram in the centre, with yellow being the highest frequency and blue the lowest.

Table A2. Overview of the optimisation parameters of the IMP runs for the jacket. Data from the experiment is provided, if available. Layer thickness has been projected from a circular to a rectangular cross section to account for the layered representation in FDS. From the best parameter sets, over all IMP runs, the minimum (IMP Min.) and maximum (IMP Max.) values are provided. The parameter sequence is the same as for the ribbon plots, simply skipping parameters that are not used. (Note: values labelled with "*" are guessed and/or FDS default.)

"Physical" Parameter	Experiment	IMP Min.	IMP Max.	Unit
Jacket Thickness	3.12e-03 *	3.80e-03	6.00e-03	m
Emissivity	unknown	3.35e-01	9.80e-01	-
Density	1.32e+03 *	8.55e+02	1.11e+03	kg/m ³
Conductivity	unknown	1.73e-01	2.06e-01	W/(m K)
Specific Heat	unknown	1.29e+00	1.71e+00	kJ/(kg K)
Heat of Combustion	2.53e+04	2.31e+04	2.74e+04	kJ/kg
Jacket Reaction A				
Pre-exponential factor	1.51e+03	1.62e+03	2.27e+03	1/s
Activation Energy	5.86e+04	5.50e+04	6.40e+04	kJ/kmol
Reaction Order	1.00e+00 *	5.16e-01	2.99e+00	-
Heat of Reaction	1.00e+03 *	2.52e+02	7.28e+02	kJ/kg
Jacket Reaction B				
Pre-exponential factor	4.92e+14	3.70e+14	4.79e+14	1/s
Activation Energy	2.28e+05	1.94e+05	2.64e+05	kJ/kmol
Reaction Order	1.00e+00 *	5.15e-01	2.20e+00	-
Heat of Reaction	1.00e+03 *	9.20e+02	1.71e+03	kJ/kg
Jacket Residue				
Density	unknown	4.81e+02	5.99e+02	kg/m ³
Conductivity	unknown	1.69e-01	2.29e-01	W/(m K)
Specific Heat	unknown	6.95e-01	1.03e+00	kJ/(kg K)
Emissivity	unknown	3.65e-01	9.89e-01	-

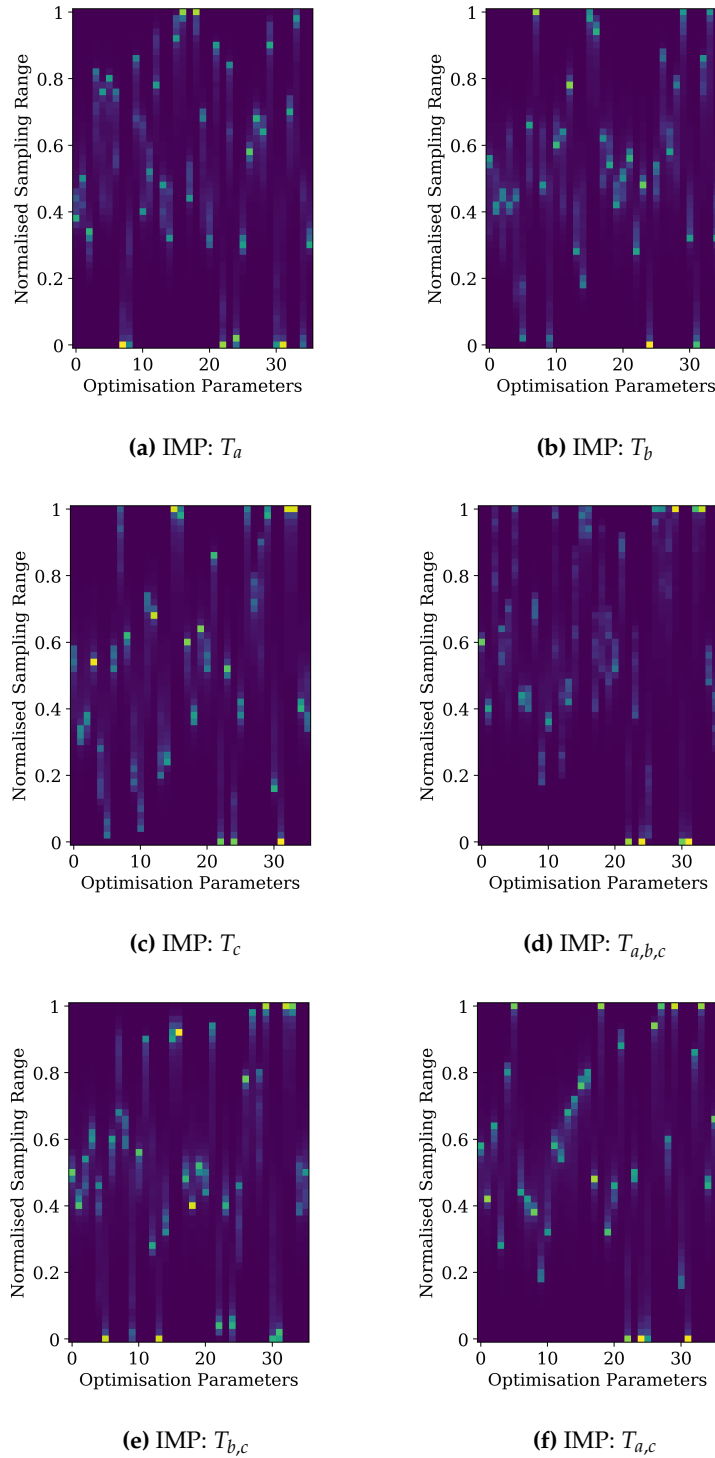


Figure A2. Frequency distribution of the optimisation parameters of the different IMP runs (T_*). The normalised sampling range is distributed over 51 bins.

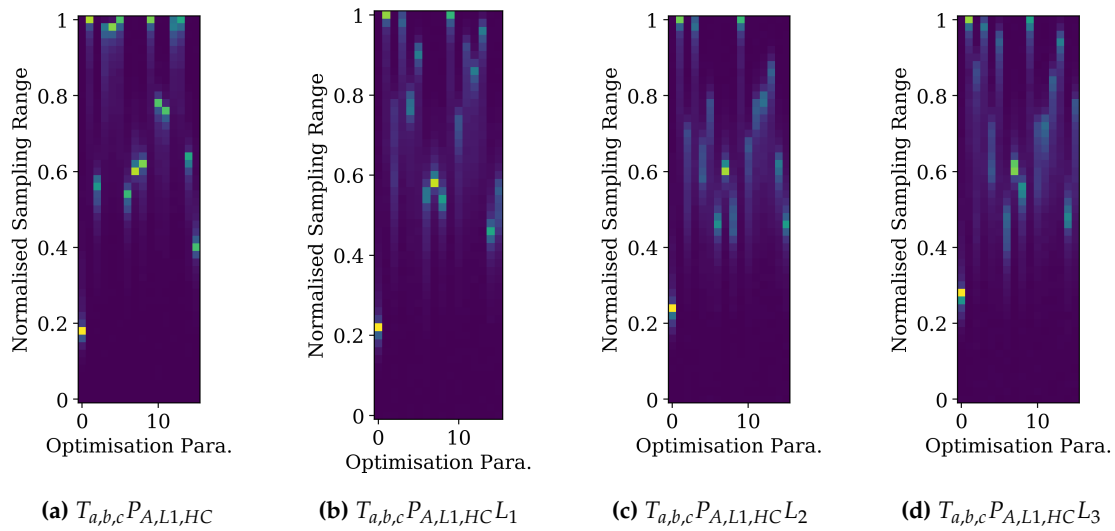


Figure A3. Comparison of the parameter distributions for adjusted parameter limits, with fixed Arrhenius parameters. Sampling ranges have individually been shifted upwards for the stuck parameters, in an effort to improve the overall fitness. Original sampling limits are shown in the left most plot, with successive shifts shown to the right.

738 Appendix A.2. Shifted Parameter Sampling Ranges

739 During the time where the primary IMP runs (T_*) were performed, the layer thickness and the
 740 density were independently varied. Later on this was changed, because both are related, thus one
 741 of them was set to a fixed value. Similarly, the HEAT_OF_COMBUSTION was removed because it is
 742 connected to the surrogate fuel concept. For the use case demonstrated in this paper the amount of
 743 released fuel is controlled via the Arrhenius model. Therefore, the released energy should correspond
 744 to this released fuel and not be scaled by a HEAT_OF_COMBUSTION value of a different material.

745 A more comprehensive assessment on the outcome of shifting the sampling limits is based on
 746 IMP $T_{a,b,c}P_A$. The decision, to use IMP runs with fixed Arrhenius parameters for this assessment,
 747 was simply made to reduce the computational demand. It is still sufficient to demonstrate the fitness
 748 improvement by shifting the sampling limits, see figure 5.

749 Figure A3 provides four ribbon plots that demonstrate how the parameter development changes
 750 for the adjusted parameter limits. On the left side, figure A3a, the original IMP run with the fixed
 751 Arrhenius parameters is shown. To its right are three successively adjusted runs ($T_{a,b,c}P_{A}L_1$ to
 752 $T_{a,b,c}P_{A}L_3$). It can be seen that for the given sampling ranges some improvement could be achieved,
 753 see also figure 5. However three parameters, stuck at the upper limit, were not sufficiently influenced.

754 The best parameter sets per generation with the adjusted limits are also tested in the SCC setup.
 755 The overall behaviour is quite similar to primary IMP runs, T_b and $T_{a,b,c}P_{A,L1,HC}$. The T_b derivatives
 756 (T_b^*) fit the 50 kW/m² quite well, as it is the target, while diverge in similar fashion for the other
 757 tests. Responses for the fixed Arrhenius derivatives show similar behaviour as IMP $T_{a,b,c}P_{A,L1,HC}$.
 758 Comparisons of the simulation responses with the Cone Calorimeter test data is provided in figure A4.

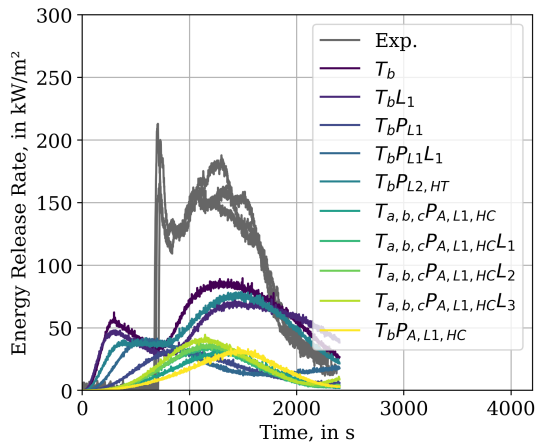
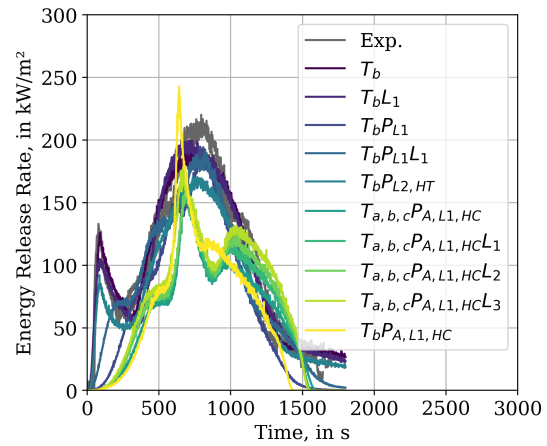
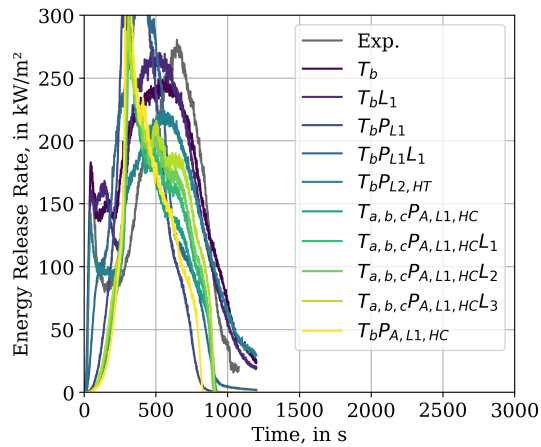
(a) Incident heat flux of 25 kW/m^2 .(b) Incident heat flux of 50 kW/m^2 .(c) Incident heat flux of 75 kW/m^2 .

Figure A4. Energy release rates of the best parameter set for IMPs with adjusted parameter limits (SCC).

759 **Appendix B. Grid Sensitivity**

760 *Appendix B.1. Cone Calorimeter*

761 To investigate the behaviour of the parameter sets for different cell resolutions, the best parameter
762 set of T_b was chosen. The simplified Cone Calorimeter simulation setup SCC was used as base mesh.
763 Its cell resolution was reduced by a factor of 0.5, thus the sample surface was covered by a single cell.
764 To achieve higher resolutions, the cell size of the base mesh was divided by factors 2 to 5. The results
765 are presented in figure A5 and compared to the experimental data (dashed line). It can be seen that
766 the parameter set does not provide a resolution-independent solution for the energy release rate. For
767 the higher external fluxes it seems to be achievable for a factor of two and higher resolutions. In the
768 25 kW/m² setup the convergence is slower. It can also be noted, that a resolution reduction leads to a
769 much more noisy response.

770 *Appendix B.2. MT3*

771 The best parameter set for IMP run T_b was also utilised in a MT3 simulation setup with 25 mm
772 cells. The results further highlight the strong grid-dependence of the parameter sets, see figure A6.
773 Note that the fire developed so strongly that the flame was cut at the upper boundary of the MESH and
774 thus the energy release rate is likely higher than the observed maximum value of (958.0 s, 975.3 kW).
775 In the higher resolution case 25 mm, the fire also propagated over nearly the whole tray length, in
776 contrast to the lower resolution. The fire in the lower trays decayed earlier than the fire on the top tray.
777 This lead to the extinguishing of the fire in the top tray before reaching the end.

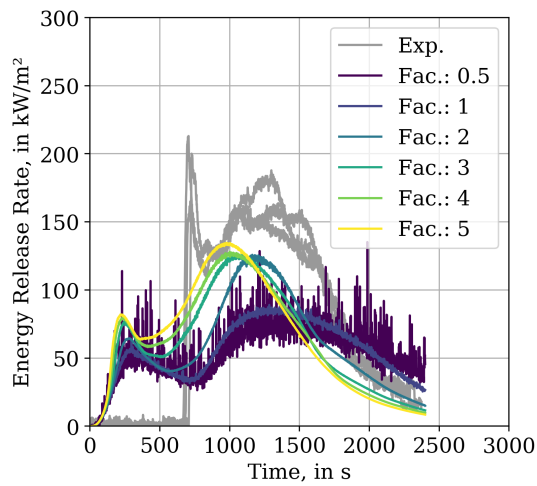
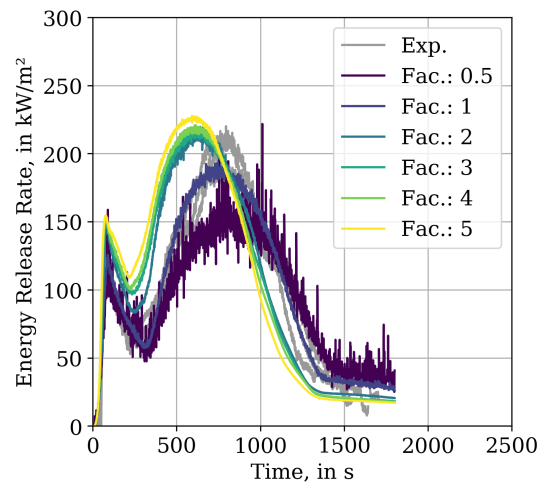
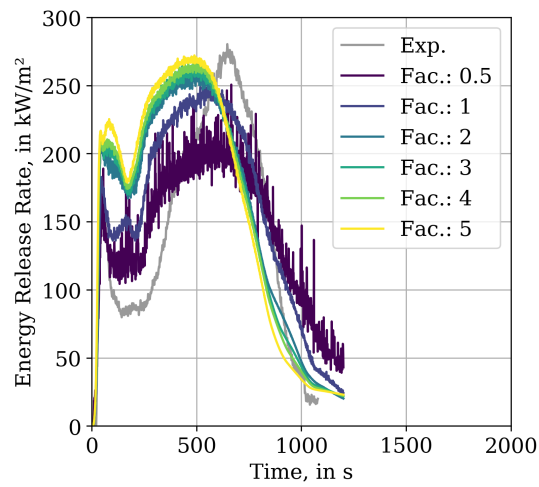
(a) Incident heat flux of 25 kW/m^2 .(b) Incident heat flux of 50 kW/m^2 .(c) Incident heat flux of 75 kW/m^2 .

Figure A5. Comparison between energy release rates of Cone Calorimeter data, from simulation and experiment (Exp.). Fluid cell sizes were changed by the noted factor, w.r.t SCC (47 mm). The used material parameter set is taken from IMP T_b .

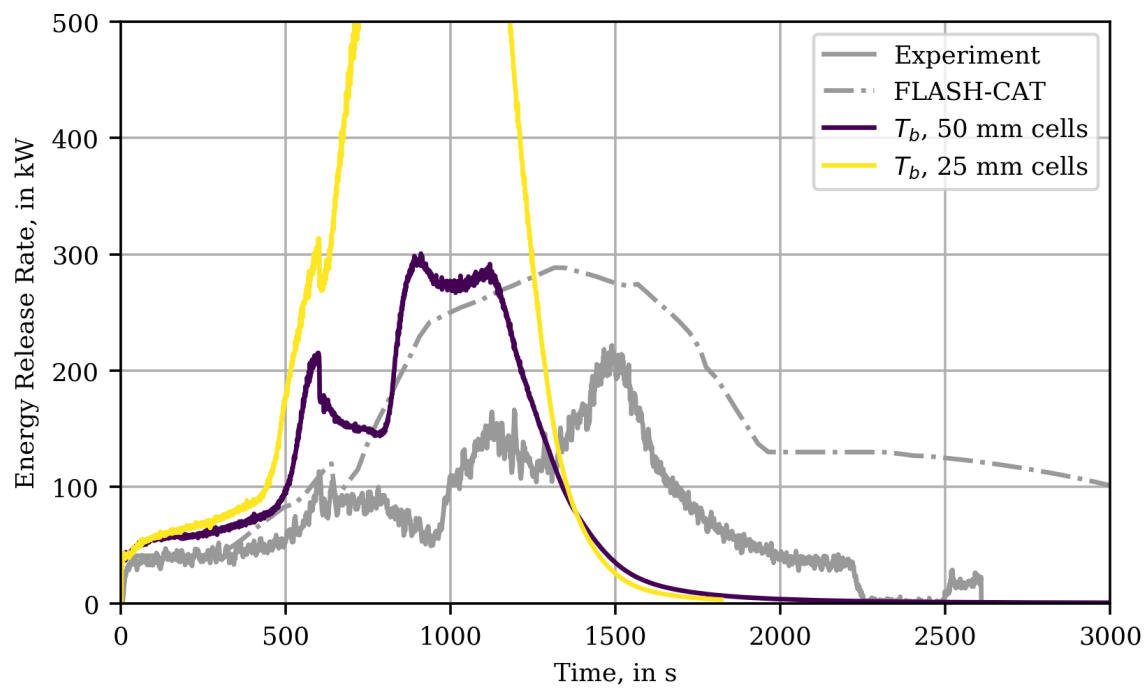
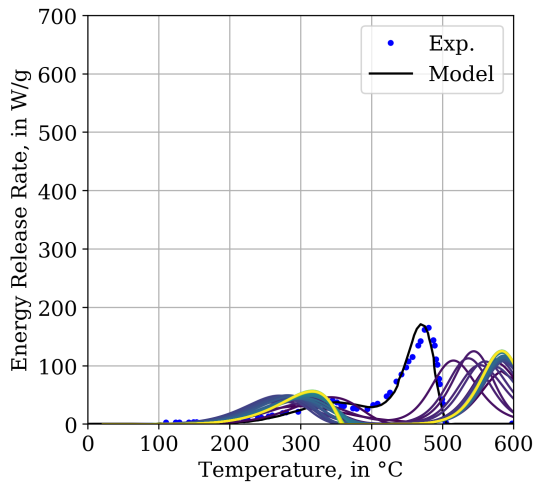


Figure A6. MT3 simulation results for the best parameter set of IMP run T_b for with cell sizes of 50 mm (original setup) and 25 mm. The peak of the simulation with 25 mm cells is located at about (958.0 s, 975.3 kW).

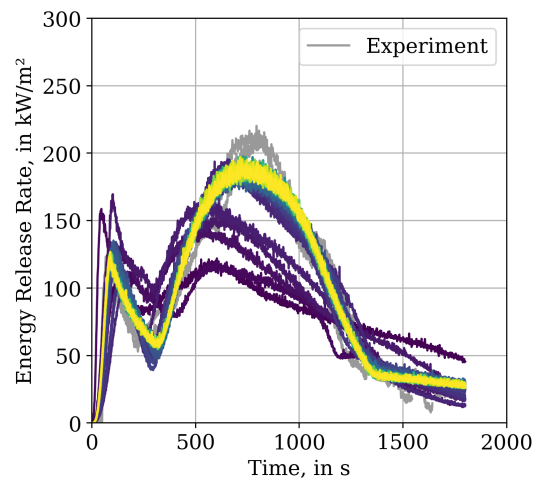
778 **Appendix C. Best Parameter Set per Generation Development**

779 To provide an impression on how the simulation response for the parameter sets change during
780 their development, three plots are provided. They contain the best parameter sets per generation
781 of IMP run T_b , applied in the three simulation setups of the stack: MCC in figure [A7a](#), simple Cone
782 Calorimeter in figure [A7b](#) and MT3 in figure [A7c](#). For all three plots, in dark blue the results of the first
783 generation are drawn, while the most recent generation is drawn in yellow. For MCC and SCC, the
784 first ten or so generations show some variation and converge afterwards.

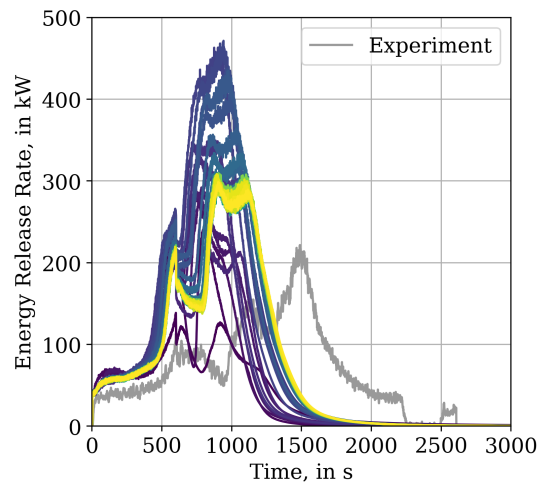
785 Interestingly, despite using the same sequence of parameter sets, the MT3 simulations show much
786 more variation during the parameter evolution, as compared to the SCC simulations.



(a) MCC simulation results of the jacket material.

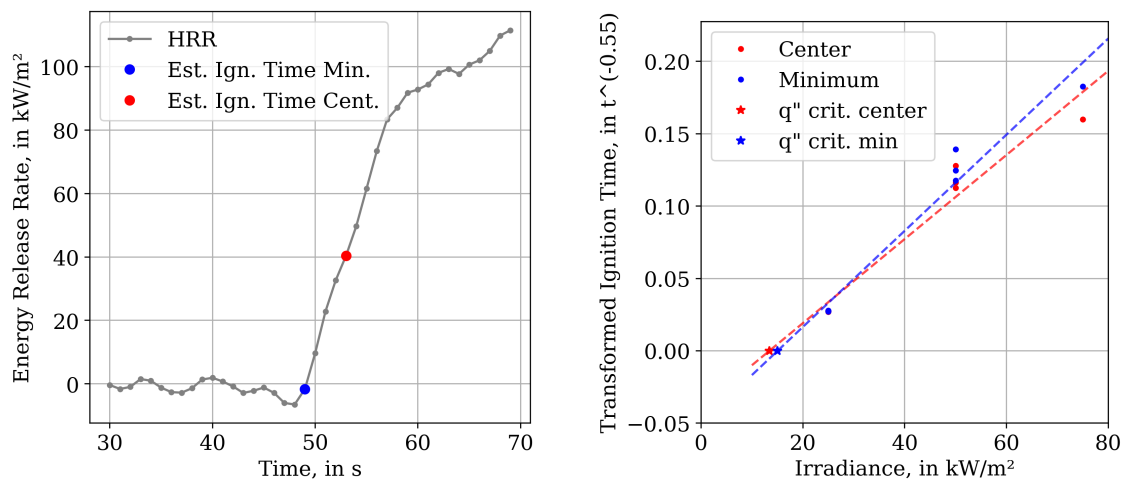


(b) Cone Calorimeter simulation results.



(c) MT3 simulation results.

Figure A7. Development of the response from the best parameter set per generation, for IMP T_b (50 kW/m^2), compared to experimental data [15]. Dark blue represents generation 0, yellow represents generation 47.



(a) Illustration of how the ignition times were estimated, based on a repetition of 50 kW/m² Cone Calorimeter test. The blue dot indicates the minimum ignition time, while the centre is marked in red. (b) Estimation of the critical heat fluxes, based on the "minimum" and "centre ignition times". Note: At 25 kW/m² six data points are overlapping, three red, three blue.

Figure A8. Illustration of the procedure to estimate ignition times in the Cone Calorimeter experiments.

787 Appendix D. Cone Calorimeter Paint Methods

788 Since no ignition times were reported with the Cone Calorimeter tests, they are estimated from the
 789 energy release rate data. For each experiment, one ignition time is taken at the beginning of the peak
 790 and one roughly in the centre. These times are taken visually from the respective plots. As example
 791 see figure A8a of a repetition of the 50 kW/m² tests. The blue dot marks the minimum ignition time,
 792 while the red dot marks the centre. Furthermore, thermal conductivity (0.1165 W m⁻¹ K⁻¹) and density
 793 (1175 kg m⁻³) for chlorosulfonated polyethylene, the jacket material, were unknown and taken from a
 794 web page providing material data for designers and engineers [40]. To steer the combustible gas release
 795 in FDS, a control function (RAMP) is created from the first repetition of the 50 kW/m² data series. The
 796 beginning of the data, until reaching the estimated ignition time, is neglected in this control function,
 797 with respect to the minimum ignition time. Note: The 25 kW/m² showed a smoother increase, thus
 798 the chosen ignition times are debatable.

799 For all seven data sets of the experiments, both ignition times, minimum and centre, are
 800 determined and are provided in figure A8b. Each group, red and blue, is used to determine the
 801 critical heat flux for Janssens' procedure, by finding the intersection point of a linear fit, through the
 802 respective group, with the x-axis. The critical fluxes are marked by a star.

803 In the paper [23] three data points were utilised, one for a higher, and two for a lower radiative
 804 flux. This lead to two different linear fits drawn into a plot, of which the x-axis intersections were
 805 determined. It is not quite clear if, for more available data points for the lower flux, an average value
 806 would be desirable or if the lowest and highest values are to be taken into account regardless. It
 807 was decided to mimic the previously described procedure. For the irradiance levels of 25 kW/m²
 808 and 50 kW/m², three data points are available. From each of these clusters the highest and lowest
 809 transformed time are taken. From the 75 kW/m² data point multiple lines are drawn one through
 810 each of the highest/lowest points described before. Due to two guessed ignition times from the
 811 experimental data, this process is performed for each group resulting in eight linear fits. Some of the
 812 lines produce intersection points on the negative side of the x-axis, leading to negative critical heat
 813 fluxes. The negative values are ignored and from the remaining positive values the lowest and highest
 814 are chosen, mixing both guessed ignition time groups together. Based on these remaining critical

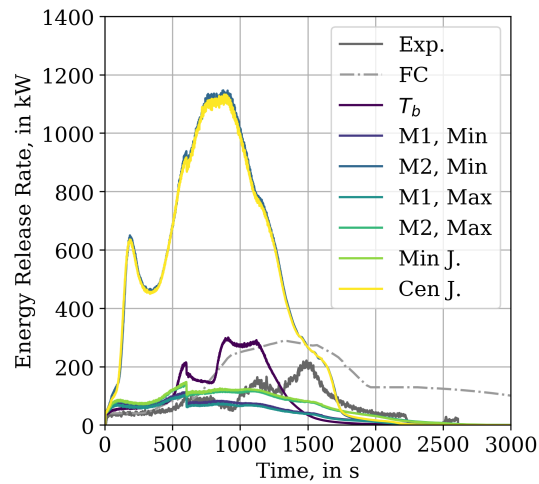


Figure A9. MT3 simulation results for different Cone Calorimeter paint methods. Labels "M1" and "M2" refer to different methods of the Beji-Merci procedure [23], "J." refers to the Janssens' procedure [35].

815 fluxes, thermal inertia parameters are determined following the two methods discussed by Beji and
 816 Merci [23]. Thus, four data sets are obtained and MT3 simulations performed.

817 For the "Beji-Merci procedure", the two methods discussed in their paper [23] followed the basic
 818 methodology of Janssens's procedure. A linear fit is created, taking two data points at different
 819 irradiance levels into account. The intersection with the x -axis is used here as critical heat flux as well.
 820 Four critical heat fluxes have been determined, one for each estimated ignition temperature and one
 821 for heat fluxes. Thus, four data sets are obtained and MT3 simulations performed.

822 Better values for Janssens' procedure could be produced when using the minimum heat flux
 823 where ignition occurs (q_{min}), instead of the critical heat flux (q_{crit}), as highlighted in the description
 824 of said procedure in the Ignition Handbook [35]. However, information about q_{min} was not available
 825 in the report [15] and thus only q_{crit} was utilised. A possible approach may be to calculate the mean
 826 point between q_{crit} and the lowest irradiance level used in the tests, however this was not attempted
 827 here. For more details on the overall procedure, see `ReportConeCalorimeterPaint.ipynb` in [25].

828 Appendix E. Heat Flux Assessment

829 Appendix E.1. Cone Calorimeter Radiative Flux Assessment

830 The fluid mesh resolution study is also used to assess the INCIDENT_HEAT_FLUX to the sample
831 surface. The results are presented in figure A10. A device is employed that integrates the heat flux
832 over the sample surface (DEVC with STATISTICS='SURFACE INTEGRAL'). The data is smoothed before
833 plotting, by employing a Savitzky-Golay algorithm. A 2nd order polynomial is utilised, with a window
834 length of about 10 % of the data points, per data series. Each of the three sub-plots contains a dashed
835 horizontal line to indicate the external flux level that represents the test condition. These values were
836 also defined by the EXTERNAL_FLUX parameter in the surface lines (SURF) during the IMP runs. It can
837 be seen, as a general observation, that the heat flux at the sample surface in the simulation is always
838 larger than the prescribed external flux value. The heat feedback decreases with lower and increases
839 with higher cell resolution. For the base SCC simulation setup (factor 1), the radiative heat flux from
840 the flame is about 3 kW/m² higher in the 25 kW/m² case to up to 10 kW/m² for the 75 kW/m² case.
841 For resolutions higher than the base SCC setup, the incident heat flux is significantly higher than the
842 prescribed external flux.

843 It is interesting to note, that the two peak structure, is mostly not reproduced in the
844 INCIDENT_HEAT_FLUX plots for a factor of 0.5. It seems also to smooth out for higher fluxes, see
845 figure A10c.

846 Appendix E.2. Multiple Tray Simulation Radiative Flux Assessment

847 Within the MT simulations, the distribution of the radiative heat flux on the tray surface
848 was tracked, per time step and surface cell. The tracked value in FDS was, among others, the
849 INCIDENT_HEAT_FLUX. As an example the INCIDENT_HEAT_FLUX distribution for T_b is shown in
850 figure A11a. Shown are the unfolded surfaces of the three obstructions representing the respective
851 trays. The larger areas of the three groups are the top and bottom faces, with the lower one being at
852 the bottom. Blue colours show a INCIDENT_HEAT_FLUX of 0 kW/m² up to 160 kW/m² in yellow.

853 To assess the general development of the INCIDENT_HEAT_FLUX over the whole simulation time
854 and tray surface, a histogram heat map is provided for the best parameter set of T_b see figure A11b. For
855 each individual time step, a histogram was created for the heat fluxes between 0 kW/m² to 160 kW/m².
856 Three dashed lines show the external heat fluxes that were used during the Cone Calorimeter tests.
857 The heat map colour encodes the surface area receiving a certain radiative flux. During the whole
858 simulation most of the area of the trays only receives low levels of heat radiation, up to about 15 kW/m².
859 At about 400 s, the area increases that receives higher radiative fluxes, which coincides with a growing
860 fire and its propagation from the bottom of the lowest tray into the space between the lowest and the
861 middle tray, where the bottom of the tray in the middle gets involved. At around 700 s some decay,
862 initialised by the burner cut-off 100 s before, is superimposed by the propagation of the fire into the
863 space between the middle tray and the tray at the top, thereafter reaching the top of the top tray. At
864 about 1000 s the fire starts to extinguish.

865 The INCIDENT_HEAT_FLUX values are demonstrated to reach much higher flux values, as
866 experimental data was available during the IMP, within the simulation. During the full MT3 simulation,
867 individual cells reached significantly higher flux levels as observed in the Cone Calorimeter simulations.
868 They are nearly twice as high, for the resolution of the IMP simulations under the most severe external
869 flux of 75 kW/m², see figure A10. For Cone Calorimeter simulations with higher resolutions the peak
870 heat flux of about 100 kW/m² gets closer to maximum in the tray but is still about 40 kW/m² short.
871 For now it is not quite clear, if the high flux levels in the trays are an artefact from the simulation
872 or realistic, since no experimental data was available to check it against. The relatively high flux
873 levels in the higher resolutions for the Cone calorimeter seem to point towards higher fluxes are to be

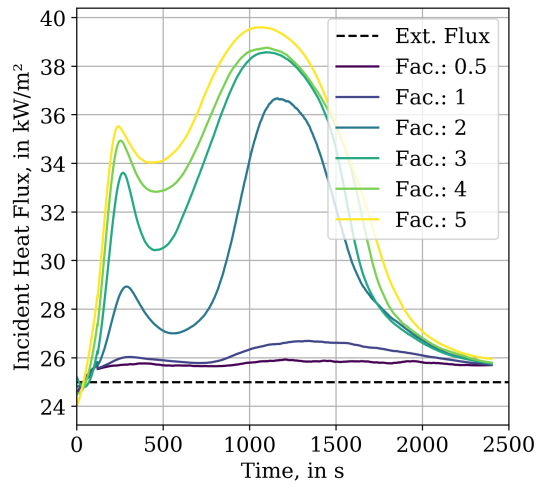
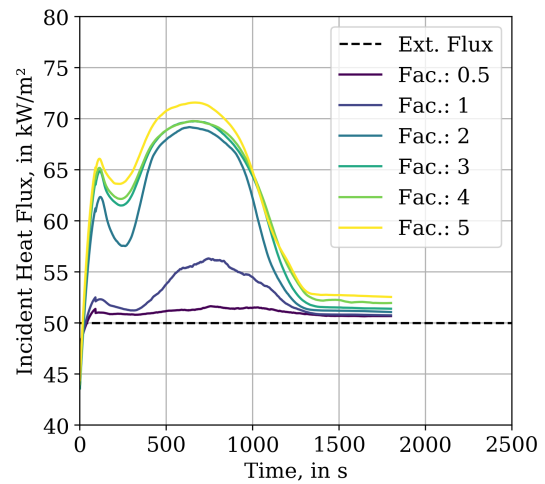
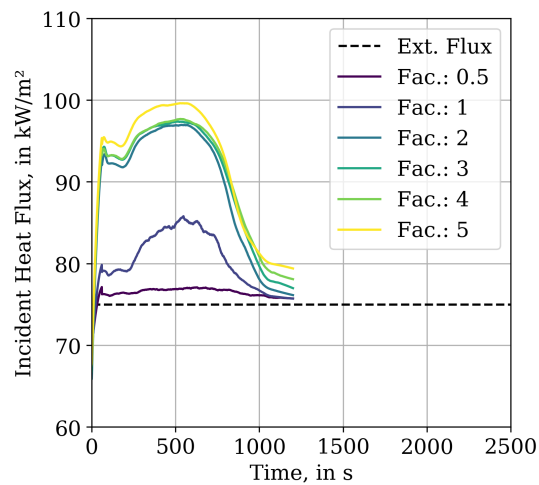
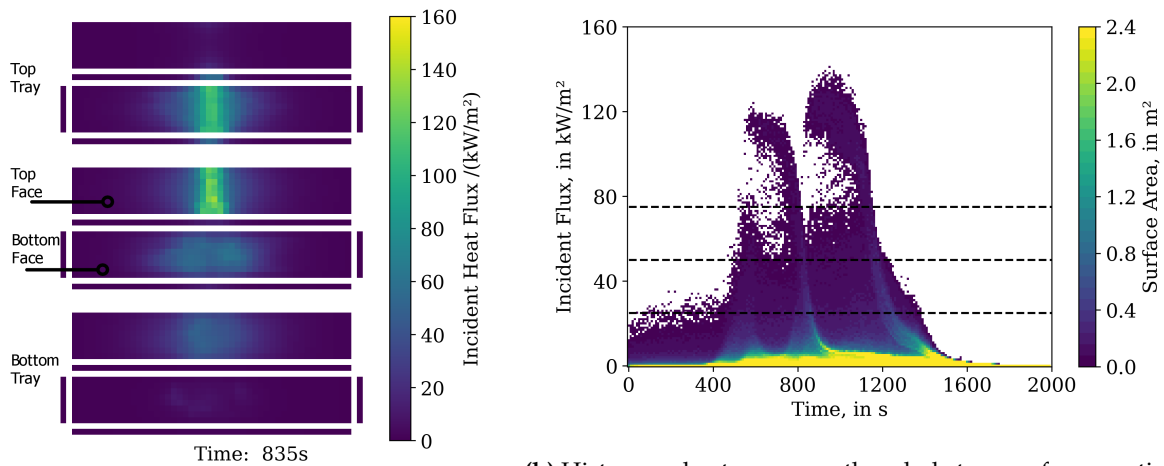
(a) Experimental condition: 25 kW/m².(b) Experimental condition: 50 kW/m².(c) Experimental condition: 75 kW/m².

Figure A10. Different INCIDENT_HEAT_FLUX responses for material parameter set of T_b in Cone Calorimeter simulations with different cell sizes. Cell sizes based on SCC (47 mm) and are changed by dividing through the noted factor. Noise reduction by Savitzky-Golay, 2nd order polynomial, window length about 10 % of the amount of data points.



(a) Unfolded tray surface for a single time step, see also figure 2a.

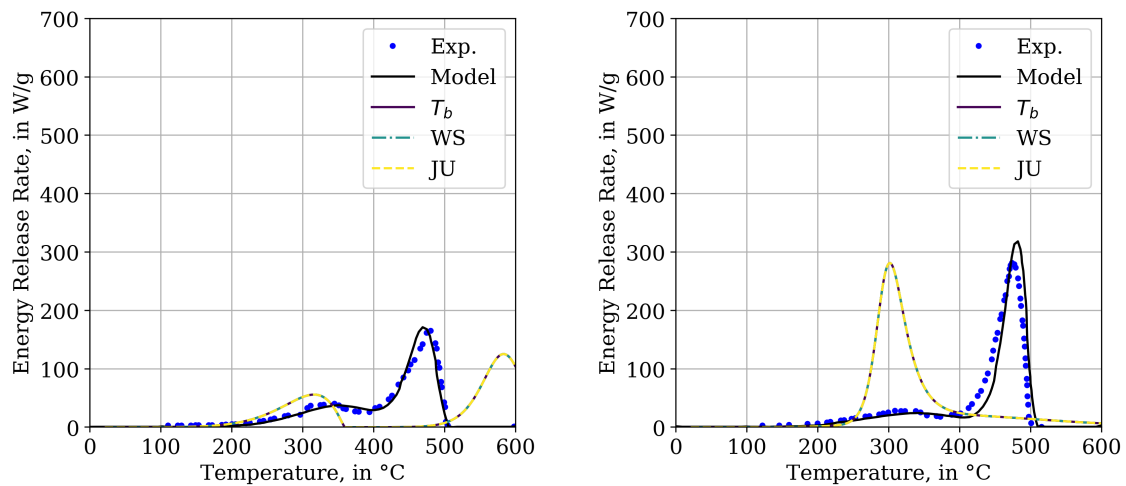
(b) Histogram heat map over the whole tray surface, per time step. External flux for Cone Calorimeter experiments shown as dashed lines. Cells of zero heat flux are omitted (white).

Figure A11. Simulation results showing the development of the INCIDENT_HEAT_FLUX on the cable tray surface for the best material parameter set of IMP T_b .

874 expected. They might simply not be able to be reproduced correctly with the low resolution during the
 875 optimisation.

876 Furthermore, as expected, it can be observed that the amount of cells with lower flux levels is
 877 relatively large. To visually distinguish areas with trivial heat flux, i.e. zero, the data points are omitted.

878 An animation of the very similar GAUGE_HEAT_FLUX development, as a side-by-side comparison
 879 between figures A11b and A11a, can be found in the Videos/MT3_GaugeHeatFlux directory within the
 880 data repository [25].



(a) MCC simulation response of the jacket material. (b) MCC simulation response of the insulator material.

Figure A12. Comparison between energy release rates of MCC simulation of the best parameter set of IMP T_b (TGA_ANALYSIS=.TRUE.). T_b : JURECA, Linux, FDS 6.5.3; WS: Workstation, Windows 10, FDS 6.5.3; JU: JURECA, Linux, FDS 6.7.0.

881 Appendix F. Computer and Software Versions

882 A brief assessment of the transfer-ability of the generated parameter sets to different FDS versions,
883 and operating systems, was conducted. Compared are here the FDS versions 6.5.3 [33] and 6.7.0.

884 Simulations necessary during the IMP were performed on the supercomputer JURECA at
885 the Forschungszentrum Jülich (FZJ) in Germany [38], which utilises an Linux-based operating
886 system. A self-compiled FDS 6.5.3, revision FDS6.5.3-0-gbac6600, was utilised for the IMP, as well
887 as simulations of the best parameter sets per generation after the completion of the respective
888 IMP. This comprised different simulation setups, the MCC, simplified Cone Calorimeter and MT.
889 Thus, consistency between the different setups was ensured. Also on JURECA, FDS 6.7.0, with the
890 revision FDS6.7.0-0-g5ccea76-HEAD, was used for comparison with FDS 6.5.3. Both FDS versions are
891 self-compiled against the software libraries available on JURECA. A pre-compiled version of FDS 6.5.3,
892 revision FDS6.5.3-598-geb56ed1 as provided by NIST via the respective web page, was used on a
893 desktop workstation with a Windows 10 operating system. The FDS input files were the same for all
894 cases, Cone Calorimeter setup and MT3 setup.

895 The best parameter set of T_b was used as an example for the transferability assessment. Thus, in
896 the subsequent plots FDS 6.5.3 on JURECA is marked with T_b . FDS 6.7.0 on JURECA is marked with
897 JU and FDS 6.5.3 on the workstation is marked with WS.

898 Results of the MCC simulations across the different FDS versions and operating systems show no
899 difference, see figure A12. For the Cone Calorimeter simulations it can be seen that the peak energy
900 release rates are slightly higher with FDS 6.7.0 (JU) than for both FDS 6.5.3 versions (WS, T_b), see
901 figure A13. The same procedure was followed with the MT3 simulation setup, see figure A14. In this
902 plot, differences are visible between all FDS versions. Both FDS versions on JURECA show a relatively
903 similar behaviour, with FDS 6.5.3 showing a bit higher peak energy release rate, while the version of
904 the workstation over-predicts the peak energy release by a factor of nearly 2.

905 This adds a further aspect of dependencies for the model, and highlights that parameter set
906 performance is also sensitive to computer architecture, software versions and operating systems.

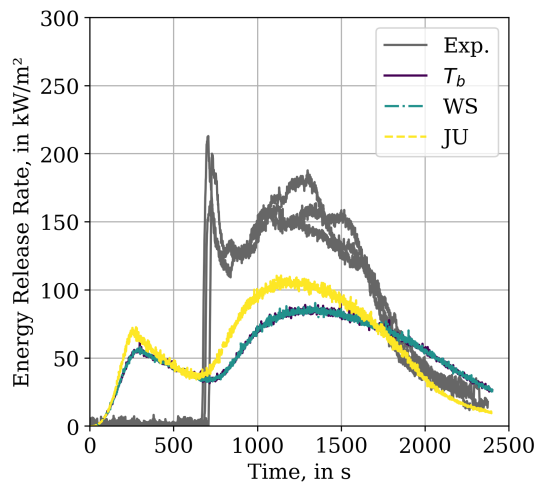
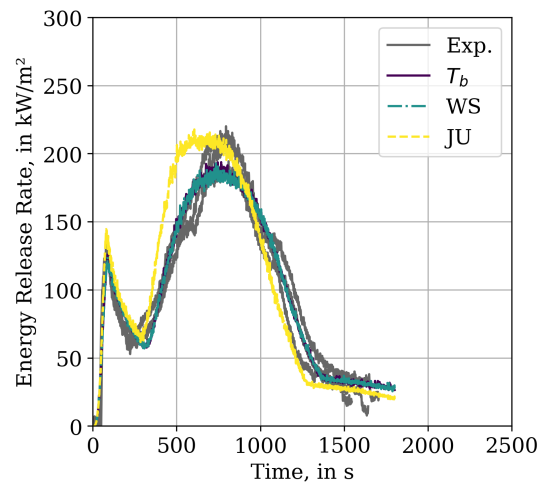
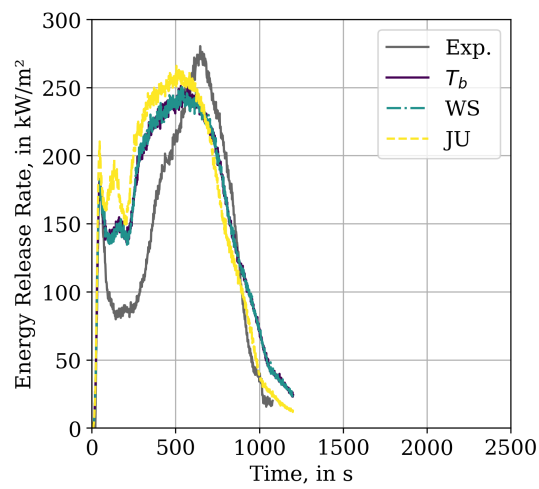
(a) Experimental condition: 25 kW/m².(b) Experimental condition: 50 kW/m².(c) Experimental condition: 75 kW/m².

Figure A13. Comparison between energy release rates of Cone Calorimeter simulation of the best parameter set of IMP T_b . T_b : JURECA, Linux, FDS 6.5.3; WS: Workstation, Windows 10, FDS 6.5.3; JU: JURECA, Linux, FDS 6.7.0.

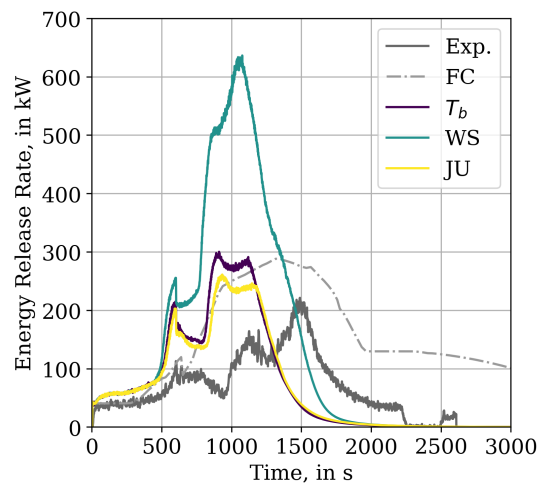


Figure A14. Comparison between energy release rates of MT3 simulation of the best parameter set of IMP T_b . T_b : JURECA, Linux, FDS 6.5.3; WS: Workstation, Windows 10, FDS 6.5.3; JU: JURECA, Linux, FDS 6.7.0.

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