

Special-purpose elements to impose Periodic Boundary Conditions for multiscale computational homogenization of composite materials with the explicit Finite Element Method

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Abstract

A novel methodology is presented to introduce Periodic Boundary Conditions (PBC) on periodic Representative Volume Elements (RVE) in Finite Element (FE) solvers based on dynamic explicit time integration. This implementation aims at overcoming the difficulties of the explicit FE method in dealing with standard PBC. The proposed approach is based on the implementation of a user-defined element, named a Periodic Boundary Condition Element (PBCE), that enforces the periodicity between periodic nodes through a spring-mass-dashpot system. The methodology is demonstrated in the multiscale simulation of composite materials. Two showcases are presented: one at the scale of computational micromechanics, and another one at the level of computational mesomechanics. The first case demonstrates that the proposed PBCE allows the homogenization of composite ply properties through the explicit FE method with increased efficiency and similar reliability with respect to the equivalent implicit simulations with traditional PBC. The second case demonstrates that the PBCE coupled with Periodic Laminate Elements (PLE) can effectively be applied to the computational homogenization of elastic and strength properties of entire laminates taking into account highly nonlinear effects. Both cases motivate the application of

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the methodology in multiscale virtual testing in support of the building-block certification of composite materials.

Keywords: Explicit FEM, Periodic Boundary Conditions (PBC), Homogenization, Multiscale computational mechanics, Composite materials

1. Introduction

With the advances in computing power and the growing costs associated to physical experiments for certification of composites, multiscale virtual testing based on the Finite Element Method (FEM) has become a popular approach in the characterization and evaluation of composite materials and structures [1]. This approach often requires homogenization techniques, as the physical response of composite materials at the macroscale is a direct consequence of their microstructural features and architecture. Moreover, the behaviour of the composite might depend on microstructural features other than the properties and topology of the microconstituents (fibres, matrix and interfaces), such as fibre volume fraction, fibre size and shape distributions, distance between neighbouring fibres, voids, among others. Computational homogenization techniques are ideal tools to take all these effects into account.

The elastic, plastic and fracture responses of laminated Fibre Reinforced Polymers (FRP) at the macroscale can be computed efficiently by following a stepwise bottom-up multiscale approach [1–3]. In the first step, *computational micromechanics* is employed to predict the homogenized behaviour of a unidirectional fibre-reinforced yarn or ply, in 2D (e.g. [4–7]) or 3D spaces (e.g. [8–11]), with input properties resulting from the experimental characterization of the composite microconstituents: fibre [12], matrix [7] and fibre/matrix interface [13]. In the case of ply architectures with higher complexity than unidirectional fibres, such as in textile composites, a subsequent homogenization step needs to be performed based on the previously computed behaviour of the unidirectional yarns, the response of the bulk resin matrix and on the topology of the Representative Unit Cell (RUC) of the fabric (e.g. [14–17]). From the orthotropic ply behaviour and lamina orientations within a ply stacking, *computational mesomechanics* can be used to predict the behaviour of the laminate (e.g. [18–20]). At this step the response of the discrete ply interfaces also needs to be taken into account because laminated FRP are prone to delamination. The homogenized behaviour of the

32 laminate can then be applied to the design of composite laminated structures
33 by employing *computational structural mechanics* [1–3]. Some of these mod-
34 elling techniques impose severe non-linearities to the respective numerical
35 problems which become intractable by implicit integration FE solvers due
36 to convergence difficulties. In such cases, explicit numerical schemes become
37 the only viable alternative to achieve meaningful numerical predictions.

38 In the framework of multiscale modelling, the use of Representative Vol-
39 ume Elements (RVE) has become a very popular numerical approach for the
40 purpose of homogenization in highly heterogeneous materials. This tech-
41 nique allows the reproduction of uniform stress states in a domain and thus,
42 the prediction of homogenized thermo-mechanical properties as elasticity and
43 strength. Apart from the selection of the RVE size, which must be sufficient
44 to capture the stress-strain response and failure mechanisms of the compos-
45 ite, the applied boundary conditions play a key role on the assessment of the
46 homogenized properties. There are three common types of boundary con-
47 ditions: uniform boundary displacements or isostrain (Hill-Reuss), uniform
48 boundary tractions or isostress (Hill-Voigt) and Periodic Boundary Con-
49 ditions (PBC). The use of PBC on the RVE boundaries implies that smaller
50 analysis domains are sufficient to obtain reliable homogenized properties [21].
51 Due to this reason, PBCs have been extensively employed in computational
52 homogenization.

53 The classical approach to introduce PBC in a RVE is by means of the
54 definition of strong relations (equations) between periodic nodes, hence im-
55 posing constraints to their allowed displacements. In its essence, this method
56 requires the mesh to be periodic, in such a way that every node on each
57 RVE boundary has its homologous node on the respective opposite (peri-
58 odic) boundary, although enhancements, based on polynomial interpolation
59 [22, 23] and Lagrange multipliers [24], have been proposed in order to avoid
60 the need of matching the mesh topology on opposite RVE boundaries. Either
61 way, the traditional PBC approach is well appropriate for implicit integration
62 numerical schemes but the fulfilment of the periodicity equations in dynamic
63 explicit time integration solvers tends to lead to spurious displacement oscil-
64 lations that compromise the numerical solution. To overcome this issue, this
65 work proposes the imposition of PBC in explicit FE solvers through special-
66 purpose elements, named *Periodic Boundary Condition Elements* (PBCE).
67 The paper demonstrates that this approach is specially well suited for mul-
68 tiscale computational analyses of composite materials and constitutes an en-
69 abling technology for multiscale computational homogenization in composite

70 materials.

71 The formulation of the PBCE for general 3D FE problems and its im-
 72 plementation as a user-defined element in Abaqus/Explicit [25] are detailed
 73 in section 2. The reliability, applicability and efficiency of the approach are
 74 then demonstrated in the framework of multiscale computational analysis
 75 of composites, in section 3. First, the PBCE method in combination with
 76 RVE is applied to micromechanical homogenization of unidirectional FRP
 77 yarns or plies. The results are evaluated through the correlation of numerical
 78 results obtained with traditional PBC and new PBCE. Then, PBCE in
 79 combination with Representative Laminate Elements (RLE) are proposed for
 80 the homogenization of laminate behaviour through computational mesome-
 81 chanics. Finally, the concluding remarks are drawn in section 4.

82 2. Periodic Boundary Condition Element

83 2.1. Definition

84 Periodic Boundary Conditions (PBC) guarantee the periodicity of the
 85 mechanical fields and ensure the continuity between neighbouring Represent-
 86 ative Volume Elements (RVE), as in a jigsaw puzzle. The PBC are set by
 87 enforcing that the difference between displacement vectors, \mathbf{u} , of opposite
 88 sides of an RVE of lengths $\ell_1 \times \ell_2 \times \ell_3$ is equal to a relative displacement,
 89 \mathbf{U}_i . In mathematical form:

$$\begin{aligned} \varphi_1(x_2, x_3, \mathbf{U}_1) &= (\mathbf{u}(0, x_2, x_3) - \mathbf{u}(\ell_1, x_2, x_3)) - \mathbf{U}_1 = \mathbf{0} \\ \varphi_2(x_1, x_3, \mathbf{U}_2) &= (\mathbf{u}(x_1, 0, x_3) - \mathbf{u}(x_1, \ell_2, x_3)) - \mathbf{U}_2 = \mathbf{0} \\ \varphi_3(x_1, x_2, \mathbf{U}_3) &= (\mathbf{u}(x_1, x_2, 0) - \mathbf{u}(x_1, x_2, \ell_3)) - \mathbf{U}_3 = \mathbf{0} \end{aligned} \quad (1)$$

90 wherein $\varphi_{i=1,3}$ are the three constraint equations relating relative displace-
 91 ments $\mathbf{U}_{i=1,3}$ of pairs of opposite nodes in the RVE sides, and $x_{i=1,3}$ are
 92 degrees of freedom (DOFs) in three dimensional space. The constraints can
 93 be introduced in the discrete potential energy associated to the weak form
 94 of the elastic equilibrium problem:

$$\Pi^h(\mathbf{u}^h) = \frac{1}{2} \int_{\Omega^h} \boldsymbol{\sigma}(\mathbf{u}^h) \cdot \nabla \mathbf{u}^h d\Omega - \int_{\Omega^h} \mathbf{u}^h \cdot \mathbf{f} d\Omega - \int_{\partial\Omega^h} \mathbf{u}^h \cdot \mathbf{h} d(\partial\Omega) + \sum_{i=1}^3 \Psi_i(\mathbf{u}^h) \quad (2)$$

95 wherein $\boldsymbol{\sigma}(\mathbf{u}^h)$ and $\nabla \mathbf{u}^h$ stand for the stress and strain tensors associated to
 96 the discrete displacement field \mathbf{u}^h , and \mathbf{f} and \mathbf{h} are the body forces and con-
 97 tact stresses at the volume and boundary of the solid, respectively. Finally,
 98 $\Psi_i(\mathbf{u}^h)$ represents the potential energy associated to the introduction of the
 99 periodicity constraints. In the case of explicit time integration, equation 2
 100 can be generalized to the dynamic problem by introducing the inertia and
 101 damping forces in the system.

102 The constraint equations (1) can be rearranged to obtain a more appro-
 103 priate form for the FEM assembly procedure. For the easy imposition of
 104 PBC, the *global* reference nodes (master nodes) M_i and M'_i are defined such
 105 that $\mathbf{U}_i = \mathbf{u}_{M_i} - \mathbf{u}_{M'_i}$, as represented in Figure 1a.

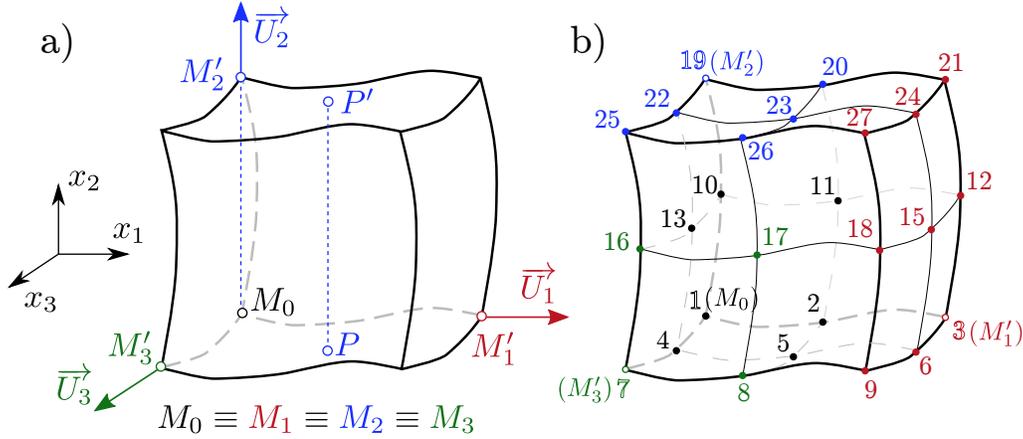


Figure 1: a) Four nodes involved in the PBC of displacement of nodes $P - P'$: M_2, M'_2, P, P' . b) Example FE model with $3 \times 3 \times 3$ nodes illustrating the coupling between periodic nodes.

106 The relative motion between a *local* point P belonging to a given plane of
 107 the RVE and point P' on the parallel plane displaced ℓ_i (length of the RVE
 108 in the direction i) can be expressed as a 4-point condition (Figure 1a),

$$\varphi_i(\mathbf{u}_P, \mathbf{u}_{P'}, \mathbf{u}_{M_i}, \mathbf{u}_{M'_i}) = (\mathbf{u}_P - \mathbf{u}_{P'}) - \mathbf{U}_i = (\mathbf{u}_P - \mathbf{u}_{P'}) - (\mathbf{u}_{M_i} - \mathbf{u}_{M'_i}) = \mathbf{0} \quad (3)$$

109 for *all* pair of opposite nodes P and P' , being $\overline{OP'} = \overline{OP} + \ell_i \mathbf{e}_i$ wherein \mathbf{e}_i
 110 is the unit vector perpendicular to the RVE planes. In this regard, \mathbf{u}_{M_i} and
 111 $\mathbf{u}_{M'_i}$ are selected to reproduce periodic homogeneous strain states through

Table 1: Boundary conditions applied through the master nodes (M'_1, M'_2, M'_3), where '0' represents a fixed DOF, '-' is a free DOF and δ a prescribed displacement.

Load case	$\mathbf{u}_{M'_1} = \vec{U}_1$	$\mathbf{u}_{M'_2} = \vec{U}_2$	$\mathbf{u}_{M'_3} = \vec{U}_3$
Uniaxial (1-direction)	$(\delta, 0, 0)$	$(0, -, 0)$	$(0, 0, -)$
Uniaxial (2-direction)	$(-, 0, 0)$	$(0, \delta, 0)$	$(0, 0, -)$
Pure shear (12-direction)	$(-, \delta, 0)$	$(0, -, 0)$	$(0, 0, -)$

112 the model. For simplicity, the displacement of the reference nodes, M_i , is
 113 set to zero to prevent rigid body motion of the whole model whereas the
 114 displacement of the master nodes, M'_i , depends on the loading case selected
 115 (see Table 1). Combined loading can be applied by superposition of boundary
 116 conditions for uniaxial and pure shear loading cases.

117 Due to compatibility reasons, periodic boundary conditions cannot be
 118 applied to every pair of periodic nodes. This is the case for node pairs
 119 (P, P') that belong to more than one PBC (edges and vertices). As a general
 120 rule, a P' node can only be part of one PBC. This is illustrated in Figure 1b
 121 for an example model with $3 \times 3 \times 3$ nodes in which each group of P' nodes
 122 is shown in a different color (red, blue and green), each color corresponding
 123 to the coupling with a reference master node (M'_1, M'_2, M'_3 , respectively).

124 The linear constraint [26] between the displacements of these four points
 125 in equation 3 can be defined as:

$$\varphi^e(\mathbf{u}_P, \mathbf{u}_{P'}, \mathbf{u}_M, \mathbf{u}_{M'}) = \mathbf{L}\mathbf{u}^e =$$

$$= \begin{bmatrix} 1 & 0 & 0 & -1 & 0 & 0 & -1 & 0 & 0 & 1 & 0 & 0 \\ 0 & 1 & 0 & 0 & -1 & 0 & 0 & -1 & 0 & 0 & 1 & 0 \\ 0 & 0 & 1 & 0 & 0 & -1 & 0 & 0 & -1 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} u_P^1 \\ u_P^2 \\ u_P^3 \\ u_{P'}^1 \\ u_{P'}^2 \\ u_{P'}^3 \\ u_M^1 \\ u_M^2 \\ u_M^3 \\ u_{M'}^1 \\ u_{M'}^2 \\ u_{M'}^3 \end{bmatrix} = \mathbf{0} \quad (4)$$

126 where u_p^i corresponds to the i -component of the displacement vector of node
 127 P .

128 Instead of satisfying the constraint exactly, a penalty approach is used
 129 such that the deviation from the exact fulfilment of the constraint penalizes
 130 the potential energy. The value selected for the penalty stiffness, k , should
 131 be high enough to make periodicity as accurate as possible. If the constraint
 132 $\boldsymbol{\varphi}^e = \mathbf{0}$ is verified, the element-wise elastic potential is minimum,

$$\Psi^e(\mathbf{u}^e) = \frac{1}{2}k \boldsymbol{\varphi}^e(\mathbf{u}^e) \cdot \boldsymbol{\varphi}^e(\mathbf{u}^e) = \frac{1}{2}k \mathbf{L}\mathbf{u}^e \cdot \mathbf{L}\mathbf{u}^e \quad (5)$$

133 and the internal forces necessary to obtain a good approximation of the
 134 constraints are calculated from the gradient of the potential, according to:

$$\left(\frac{\partial \Psi^e}{\partial \mathbf{u}^e} \right) = k \mathbf{L}^T \mathbf{L}\mathbf{u}^e = \mathbf{F}_k^e \quad (6)$$

135 This approach can be seen as a generalized spring network between nodes
 136 belonging to the boundaries, pulling the system back to the periodic con-
 137 straint. Hence, its natural implementation in an explicit time integration
 138 FE solver such as Abaqus/Explicit [25] is by means of 4-node user-defined
 139 non-volumetric elements, henceforth named Periodic Boundary Condition
 140 Elements (PBCE), defined by means of a subroutine VUEL. The application
 141 of these user element implies the replacement of each set of PBC constraint
 142 equations (1) by a PBCE. Each PBCE enforces a local “penalty” constraint
 143 between opposite nodes P , P' , and master nodes M_i , M'_i (Figure 1a) similar
 144 to the classic PBC (equations 1). The points $\{M_i, M'_i\}$ are assembled to
 145 be the same for each pair of opposite surfaces so that the globally-imposed
 146 displacement difference \mathbf{U}_i , is the same for all pairs of opposite nodes P , P' .

147 The *global* constraint $\Psi_i(\mathbf{u}^h)$ and the external forces \mathbf{F}_{ext} appear natu-
 148 rally when the elements associated with the nodes belonging to the domain
 149 boundaries are assembled, and the displacements/forces are imposed to the
 150 master nodes. The constraint is satisfied approximately for each pair of oppo-
 151 site nodes. With the PBCE, the displacements of nodes M_i are constrained,
 152 whereas the displacements of nodes M'_i are imposed (Figure 1a).

153 It should be noted that either relative displacements \mathbf{U}_i or forces \mathbf{F}_i can be
 154 externally imposed through the master nodes. For instance, recalling Table 1,
 155 a uniaxial test in the direction 3 is imposed by means of $\mathbf{U}_3 = (0, 0, \bar{\epsilon}_3 \ell_3)$
 156 and $\mathbf{U}_1 = (u_1, 0, 0)$ and $\mathbf{U}_2 = (0, u_2, 0)$, being $\bar{\epsilon}_3$ the average strain imposed

157 to the RVE along direction 3. In this case, u_1 and u_2 stand for the output
 158 lateral Poisson contraction resulting from the FEM computation.

159 As it is presented, this method originates undamped oscillations in dy-
 160 namic analyses, as verified in preliminary simulations. Hence, damping mech-
 161 anisms are implemented in the PBCE while preventing that its valid motions
 162 are affected. Viscous Rayleigh damping gives a force proportional to the neg-
 163 ative rate of change of $\mathbf{L}\dot{\mathbf{u}}^e$ and parallel to the elastic force:

$$\mathbf{F}_c^e = c \mathbf{L}^T \mathbf{L} \dot{\mathbf{u}}^e \quad (7)$$

164 where c is a damping coefficient. For low loading rates, for which the effect of
 165 inertial forces is negligible, an additional mass m can be added to the system
 166 in the same way:

$$\mathbf{F}_m^e = m \mathbf{L}^T \mathbf{L} \ddot{\mathbf{u}}^e \quad (8)$$

167 Finally, the resulting equation of motion of the element, taking into account
 168 the external forces, becomes

$$\mathbf{0} = \mathbf{F}_k^e + \mathbf{F}_c^e + \mathbf{F}_m^e - \mathbf{F}_{ext}^e = \mathbf{L}^T \mathbf{L} (k \mathbf{u}^e + c \dot{\mathbf{u}}^e + m \ddot{\mathbf{u}}^e) - \mathbf{F}_{ext}^e \quad (9)$$

169 *2.2. Stiffness, damping and mass parameters*

170 The selection of the parameters k , c and m of the PBCE must be done
 171 according to a compromise between the accuracy of the results and the com-
 172 putational cost. For instance, high values of k would increase the accuracy
 173 of the periodic condition, nevertheless, the critical stable time increment of
 174 the PBCE (Δt_{stab}^{PBCE}) would be reduced, increasing the number of increments
 175 to complete the simulation.

176 Based on equation 9, the Courant-Friedrichs-Lewy (CFL) condition [27]
 177 can be analysed for the derived PBCE to obtain the stable time increment
 178 of the user element as,

$$\Delta t_{stab}^{PBCE} \approx \frac{2}{\omega} \cdot \left(\sqrt{1 + \xi^2} - \xi \right) \quad (10)$$

179 where the angular frequency of the user element is $\omega = 2\sqrt{k/m}$, and $\xi =$
 180 c/\sqrt{km} .

181 Two examples on how to select these parameters are described in Sec-
 182 tions 3.1 and 3.2.

183 **3. Multiscale computational applications**

184 The traditional approach to implement PBC is by means of constraint
185 equations (`*EQUATION` in Abaqus [25]). This method has strong founda-
186 tions for implicit solvers based on static equilibrium, but exhibits several
187 drawbacks when explicit dynamic time integration (i.e. central differences)
188 is used. It is observed that the relationships between master and slave dis-
189 placements is translated into equations that introduce intense high-frequency
190 oscillations in the system. It has not been possible to identify the exact cause
191 for this behaviour which is possibly related to implementation difficulties in
192 explicit algorithms. Related difficulties might be in the origin of the lim-
193 itation in the number of supported constraint equations ¹. Moreover, the
194 method with traditional PBC is computationally expensive. The Periodic
195 Boundary Condition Element (PBCE) approach proposed in this paper is
196 more efficient under similar conditions and it is not limited in the number of
197 periodic DOF's.

198 In the following, the PBCE method is applied and validated under two
199 computational homogenization scenarios in composite materials: microme-
200 chanical and mesomechanical homogenization.

201 *3.1. Micromechanical homogenization*

202 Micromechanical homogenization in composite materials is generally used
203 to compute the elastic and strength properties of an orthotropic lamina and
204 predict ply failure envelopes, e.g. [5–7, 11]. The behaviour of the ply trans-
205 verse to the fibres direction can be analysed with two-dimensional or quasi-2D
206 RVE, as shown in Figure 2. Herewith, a 2D version of the PBCE presented
207 above is used in the computation of transverse tensile properties of the uni-
208 directional Carbon-Fibre Reinforced Polymer (CFRP) material AS4/8552.

209 The microstructure of the RVE of an unidirectional composite is idealized
210 as a dispersion of parallel and circular fibres randomly distributed in the
211 polymer matrix. A minimum of 50 fibres is generally enough to capture
212 adequately the essential features of the microstructure of the material while
213 maintaining reasonable computing efforts, as demonstrated by González and
214 LLorca [28]. Synthetic fibre distributions statistically equivalent to the real

¹In Abaqus, this limit has been increased from version to version, being around 90000 for v6.14 [25]

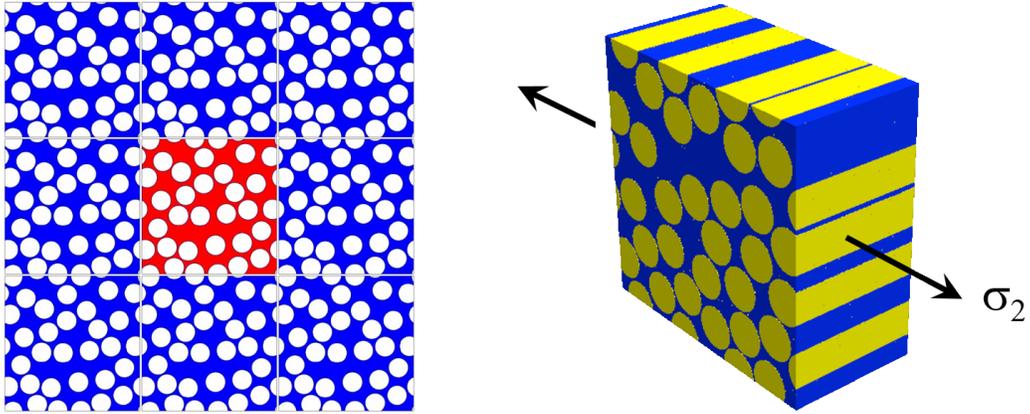


Figure 2: The composite mechanical behaviour is determined by solving numerically the boundary value problem for a RVE of the composite which is much larger than the heterogeneities in the microstructure.

215 ones are generated with a modified Random Sequential Adsorption (RSA)
 216 algorithm [8].

217 The RVE is discretized in Abaqus/Explicit [25] in the following way: the
 218 matrix and the fibres are modelled with 4-node fully integrated quadrilat-
 219 eral isoparametric elements under the assumption of plane strain (CPE4),
 220 while the fibre-matrix interface debonding is simulated with 4-node cohesive
 221 isoparametric elements (COH2D4) inserted at the interfaces between fibres
 222 and matrix. Perfect and homogeneous contact between fibres and matrix is
 223 assumed. The carbon fibres are assumed to behave as linear elastic trans-
 224 versely isotropic solids. The matrix is modelled as an isotropic elastic-plastic
 225 solid according to a modified Drucker-Prager plasticity yield surface includ-
 226 ing damage [25, 29]. The fibre-matrix interface behaviour follows a mixed-
 227 mode bilinear traction-separation law [25]. Detailed information about the
 228 constitutive models and material properties can be found in [4, 7].

229 A reference analysis was carried out with Abaqus/Standard [25] within
 230 the framework of the finite deformations theory. In addition, explicit dy-
 231 namic analyses employing the default Abaqus/Explicit [25] PBC scheme,
 232 by means of constraint equations, were also run for comparison with the
 233 developed PBCE approach. In each analysis, an initial thermo-mechanical
 234 loading step simulates the cooling-down process from curing to service tem-
 235 peratures, given the significant influence of the respective residual stresses

236 on the homogenized properties. This stage is followed by the application of
 237 mechanical load up to failure. Two typical load-cases were analysed herein:
 238 uniaxial transverse tension and transverse compression.

239 A careful selection of the mechanical parameters of the PBCE was done
 240 in advance to maximize the accuracy of the simulation without penalizing its
 241 computational cost. To this end, m was selected as the average nodal mass
 242 of the model ($m = 3.73 \cdot 10^{-7}$ g) such that no remarkable mass concentration
 243 would take place along the boundaries. In order for the PBCE to provide a
 244 good approximation of the periodicity condition, k must be high compared
 245 to the overall stiffness of the model (on each direction),

$$k \cdot N_e \gg k_{\text{model}} \quad (11)$$

246 where the stiffness of the model in the transverse direction is $k_{\text{model}} = E_2 \cdot$
 247 $A/L \approx 10^4$ N/m, and N_e is the number of user elements in the transverse
 248 direction ($N_e \approx 100$). A value of $k = 10^5$ N/m was found to be enough
 249 for the current analyses. Based on preliminary simulations, a value of $c =$
 250 0.001 N s/m for the damping coefficient was sufficient to remove spurious
 251 oscillations. This combination of parameters did not penalize the stable
 252 time increment of the simulation, i.e. $\Delta t_{\text{stab}} < \Delta t_{\text{stab}}^{\text{PBCE}}$ (see equation 10).

253 Load was applied by means of a velocity-controlled profile following a
 254 smooth step to minimize shock waves that would introduce high inertial
 255 effects. For both load cases, the steady-state loading rate selected was $5 \cdot$
 256 10^{-4} m/s with a peak acceleration of 0.6 m/s².

257 The stress-strain curves resulting of the different analyses, as well as stress
 258 fields for the tensile cases and strain fields for the compression cases, are
 259 shown in Figure 3. For transverse tension, it is observed that the mechan-
 260 ical fields are equivalent between implicit and explicit analyses, and that
 261 ultimate failure is triggered by the same cracking mechanisms at similar ap-
 262 plied stress level (≈ 51.5 MPa) in both schemes. However, the explicit FE
 263 results using constraint equations (Explicit-PBC) are highly oscillatory and
 264 under-predict the transverse tensile strength of the material. For transverse
 265 compression, the match between implicit and explicit analyses with PBCE is
 266 again remarkable in terms of strain fields and load at failure (≈ 205 MPa).
 267 The explicit analysis with constraint equations also shows an oscillatory re-
 268 sponse although the transverse compression strength obtained matches the
 269 one predicted by the two other schemes.

270 A summary of the computational cost associated to the solution of the

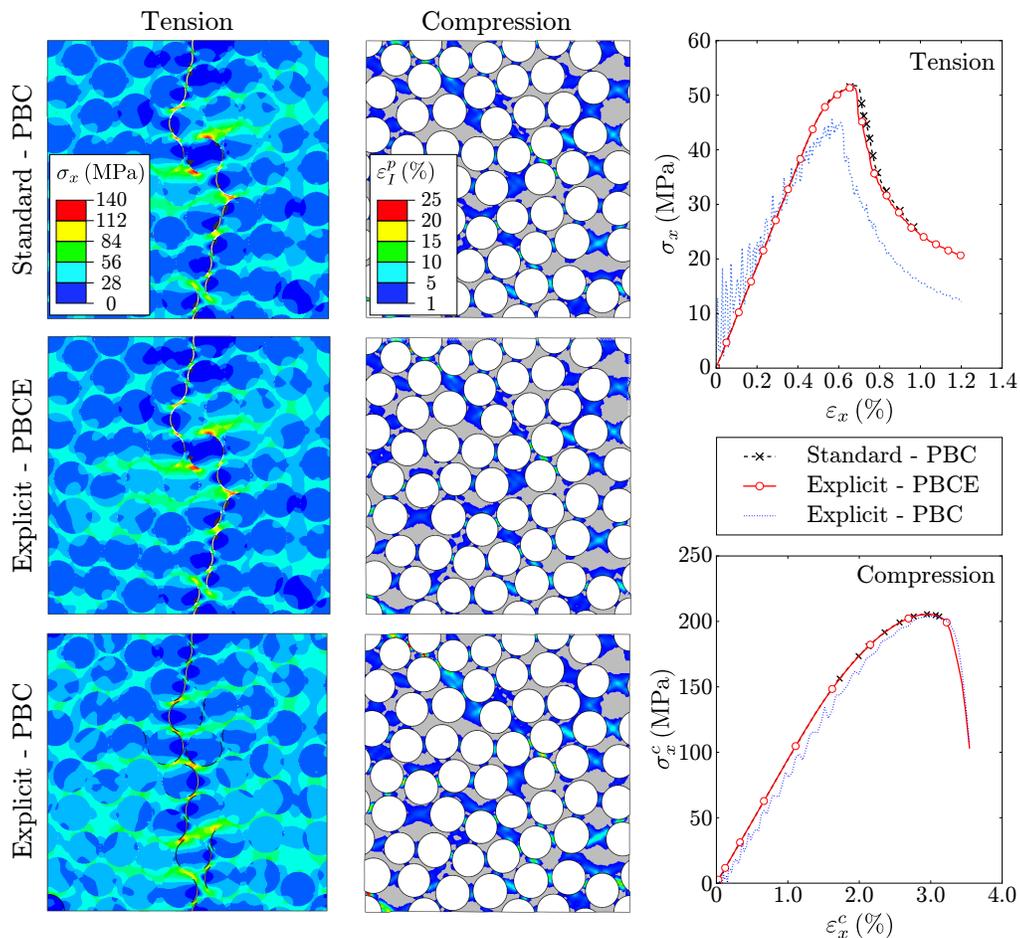


Figure 3: Comparison of the results obtained with Periodic Boundary Conditions Elements (PBCE) in Explicit against the Periodic Boundary Conditions (PBC) in Standard and Explicit by means of constraint equations. Transverse tension (left column) and compression (middle column) load cases are shown. The resulting stress-strain curves for each load case (tension and compression) for the three different schemes are shown in the right column.

271 tensile loading case by each of the numerical schemes is given in Table 2.
 272 For a fair comparison, all calculations were performed in a single CPU
 273 (Intel[®] Xeon[®] E5-2680 processor). In overview, the time required by the
 274 implicit solver (Abaqus/Standard) to complete the simulation is remark-
 275 ably higher than for the explicit approaches. Nevertheless, the computation

276 time required to reach the peak load point (value in parenthesis) is consider-
 277 ably lower for the implicit solver which takes advantage of the initial quasi-
 278 linearity of the problem by allowing large load increments at this stage. The
 279 explicit schemes are remarkably advantageous in the softening regime where
 280 the simulation becomes highly non-linear due the appearance of plastic defor-
 281 mation and damage. By comparing both explicit approaches, it is observed
 282 that the use of PBCE results in a $\approx 35\%$ reduction in computation time with
 283 respect to the traditional PBC scheme. The amount of memory required by
 284 the solver is also slightly reduced with the PBCE method ($\approx 10\%$).

Table 2: Comparison of the computational efficiency achieved with the PBCE scheme compared to the traditional solving schemes for the tensile load case (see Figure 3). In parenthesis, the computation time required to reach the peak load point.

Scheme	Computation time [s]	Memory required [Mb]
Standard - PBC	1766 (186)	32.0
Explicit - PBCE	800 (592)	16.9
Explicit - PBC	1055 (781)	19.3

285 3.2. Mesomechanical homogenization

286 The use of PBC at the mesoscale allows for the definition of a Represent-
 287 ative Laminate Element (RLE), in essence a RVE of a laminate [18, 19], as
 288 represented in Figure 4. The use of PBC aims at introducing an uniform far-
 289 field stress to a small portion of the laminated material structure, assuming
 290 that the RLE behaviour is statistically representative of the whole specimen
 291 [30]. In this way, this approach allows the computation of the homogenized
 292 elastic and strength properties for a given laminate configuration in all or-
 293 thotropic directions, and the prediction of a laminate failure envelope.

294 The traditional way to determine laminate properties and qualify com-
 295 posite materials for structural applications is done through costly and time
 296 consuming experimental testing following carefully devised test standards.
 297 In the recent years, numerical simulation arose as a promising alternative to-
 298 wards efficient material certification by virtual testing, with the added advan-
 299 tage that a much larger range of configurations can be considered [1, 2, 31].
 300 The standard test methods can be modelled with high-fidelity and accu-
 301 rate predictions of laminate behaviour and relevant properties achieved, as
 302 demonstrated by Falcó et al. [20]. Both physical and virtual approaches aim

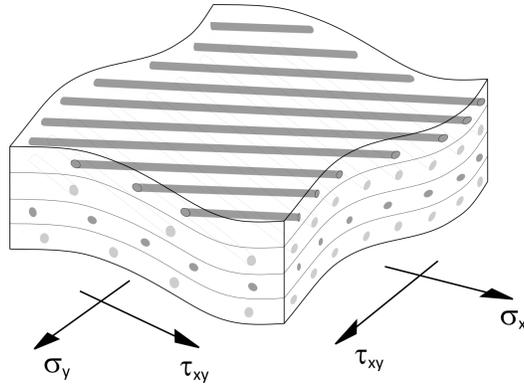


Figure 4: Representative Laminate Element (RLE).

303 at reproducing a macroscopically homogeneous stress state such that the re-
 304 sultant behaviour can be considered intrinsic to the laminate configuration.
 305 However, because of the finite width of the coupons and the three dimensional
 306 stress states at their edges [32, 33], the behaviour is significantly affected by
 307 edge cracking and delamination. By means of the RLE approach proposed
 308 in this paper, edge effects are removed from the boundaries of the numerical
 309 model and replaced by PBC, so that the analyses address only the material
 310 response. Moreover, the computational requirements are remarkably reduced
 311 since the RLE can be much smaller than the virtual coupon.

312 To capture the relevant mechanisms of laminate behaviour, the RLE do-
 313 main is discretized in plies and ply interfaces. While interlaminar damage is
 314 assumed to occur in the form of delaminations along predefined and discrete
 315 crack planes, ply damage might occur in the form of fibre breakage, fibre
 316 pull-out, kink-banding and matrix cracking at any location within the plies.
 317 Hence, the appropriate description of the ply interface behaviour is achieved
 318 by means of cohesive and frictional relations between discrete fracture planes
 319 whilst the ply deformation mechanisms can be adequately tackled by means
 320 of a Continuum Damage Model (CDM) [20]. This modelling approach im-
 321 poses severe instabilities, such as snap-back due to brittle cracking, to the
 322 numerical problem which typically result in convergence issues in implicit
 323 solvers. Therefore, the explicit numerical integration of the RLE, coupled
 324 with the PBCE proposed in this paper, constitutes the enabler of the com-
 325 putational homogenization of laminate behaviour.

326 For the purpose of demonstration, the In-Plane Shear (IPS) test on an
 327 AS4-8552 laminate is addressed herein. This experiment is used to charac-

328 terize the in-plane shear response of a ± 45 laminate, and is defined according
 329 to the ASTM D3518 test standard [34]. It consists of a rectangular coupon
 330 of $[\pm 45]_s$ configuration, 25 mm in width by up to 250 mm in length, loaded
 331 under quasi-static tension up to failure. To define an appropriate RLE, it is
 332 sufficient to consider an area of $10 \times 10 \text{ mm}^2$ of the laminate to capture a
 333 representative number of intralaminar cracks for the element width employed
 334 (0.2 mm), as shown in Figure 5. Since the laminate at any point is statisti-
 335 cally representative of the laminated structure, the only constraint on the
 336 dimensions of the RLE is that it should be much larger than the characteris-
 337 tic dimensions of the physical mechanisms that are to be simulated. In this
 338 case, the relevant phenomena are matrix cracking and delamination, which
 339 are associated to fracture process zones of the order of less than a millimetre
 340 [35]. Moreover, due to the out-of-plane symmetry of the $[\pm 45]_s$ configuration,
 341 only two plies (± 45) need to be modelled with properly imposed symmetry
 342 boundary conditions.

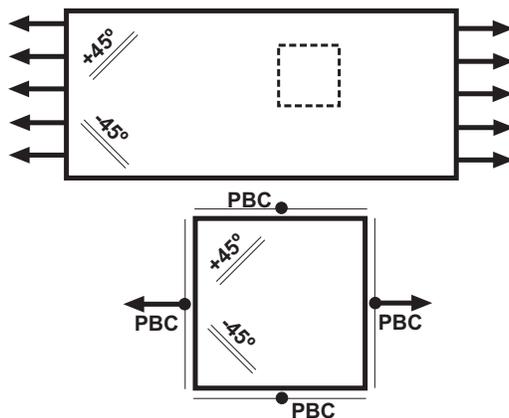


Figure 5: Illustrations of the In-Plane Shear (IPS) test (top) and the corresponding RLE (bottom) with applied PBCs and loads.

343 The laminate modelling approach follows the work of Falcó et al. [20]. Ac-
 344 cordingly, the ply interface response is modelled by means of a general mixed-
 345 mode cohesive zone method coupled with frictional behaviour. The coupled
 346 cohesive-frictional approach is adopted to include the possible effects of ply
 347 friction during and after delamination, and is implemented in the kinematics
 348 of surface contact interaction algorithms available in Abaqus/Explicit [25].
 349 The unidirectional FRP plies are modelled by means of a thermodynamically-

350 consistent CDM that takes into account the relevant ply deformation mech-
351 anisms [20]. The nonlinear elastic-plastic shear behaviour of the material
352 is modelled by a Ramberg-Osgood law [36]. The possibility of elastic un-
353 loading is tackled by means of a general elastic predictor - plastic corrector
354 algorithm. The relevant ply and interface properties required by these models
355 are given in [20]. Similar properties for the same material (different batches)
356 are available in [37]. A regularized meshing approach is used, with material-
357 alignment and directional biasing, as described in [20]. Each ply (0.184 mm
358 in thickness) is discretized with a single through-the-thickness plane of reg-
359 ular 8-noded hexahedral isoparametric elements of $0.6 \times 0.2 \times 0.184 \text{ mm}^3$
360 in volume with reduced integration (C3D8R), except around the RLE edges
361 wherein tetrahedral elements (C3D6R) are used.

362 As in the computational micromechanics case above, a judicious selection
363 of the mechanical parameters of the PBCE was performed to ensure both
364 the accuracy and the efficiency of the simulation. To this end, the PBCE
365 damping and stiffness coefficients were set to $c = 0.1 \text{ N s/mm}$ and $k =$
366 $2 \cdot 10^5 \text{ N/mm}$, respectively. The nodal mass of the PBCE was taken as the
367 average nodal mass of the RLE.

368 Quasi-static tensile displacements were imposed to the RLE, as repre-
369 sented in Figure 5, until collapse was produced by the accumulation of matrix
370 cracks and delamination between the $+45^\circ$ and -45° layers. For the purpose
371 of qualitative correlation (Figure 6), the simulated accumulation of matrix
372 cracks is compared with equivalent experimental results of an IPS test on a
373 similar carbon/epoxy material which have been obtained by means of X-ray
374 computed tomography (XCT) [38, 39].

375 In the experiments (Figure 6, left), cracks develop similarly in the $+45^\circ$
376 and -45° layers, starting from the edges of the specimen, following directions
377 parallel to the fibres due to the kinematic constraints imposed by the mi-
378 crostructure. The crack density is always higher around the edges than in
379 the specimen central sections and it increases with the applied load until
380 saturation. Delamination also grows from the specimen edges. Finally, the
381 accumulation of matrix cracking and delamination leads to instability and
382 specimen collapse. The simulations on the smaller size RLE (Figure 6, right)
383 capture this damage pattern while discarding the undesirable effects caused
384 by the edges. It should be mentioned that, whilst the XCT is able to capture
385 critical and sub-critical damage mechanisms, the simulations only predict the
386 first, i.e. cracks completely developed through the thickness of the plies. Al-
387 though the CDM does not contain information of the kinematic constraints

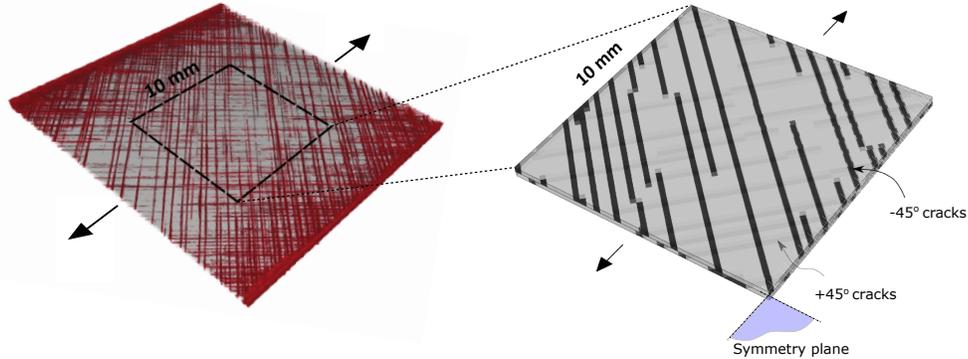


Figure 6: Qualitative correlation between experimentally-obtained (left) and simulated (right) development of matrix cracking in a plain stress $[\pm 45]_s$ laminate (experimental results adapted from [38]). Note: both experiments and simulations performed in similar carbon/epoxy $[\pm 45]_s$ coupons, although not exactly the same material.

388 imposed by the ply microstructure (the shear parallel and perpendicular to
 389 the fibre are represented with the same deformation tensor), this effect is ob-
 390 tained with the regularized meshing with material-alignment and directional
 391 biasing [20], leading to the correct simulation of crack directions. Hence, the
 392 RLE can be considered approximately representative of the central sections
 393 of the finite-width IPS coupon.

394 The results of the simulation in terms of the stress-strain behaviour are
 395 shown in Figure 7. The response of the RLE is nonlinear in a very similar
 396 way to the Ramberg-Osgood law [36] implemented at the constitutive level
 397 to describe the pure shear stress vs. shear strain relation of the ply, although
 398 not exactly since the IPS test configuration does not create pure shear on the
 399 ply but a mixed-mode loading situation, with a small fraction of transverse
 400 tension. For this same reason, the ultimate IPS load, $IPSS = 99.7$ MPa, also
 401 diverges from the ply shear strength, $S_L = 110.4$ MPa [37]. This demonstrates
 402 that this property is not adequately characterized by the IPS experiment [34],
 403 and a better alternative for that purpose is the Short Beam test standard
 404 ASTM D2344M [40] that measures the Interlaminar Shear Strength (ILSS)
 405 in a laminate.

406 Through-the-thickness matrix cracking, as shown in Figure 6, initiates at
 407 the highest load and deformation stages, rapidly growing and interacting with
 408 interface delamination to produce the collapse of the RLE. The simulated
 409 cracking is, however, not influenced by coupon edge effects as in the IPS

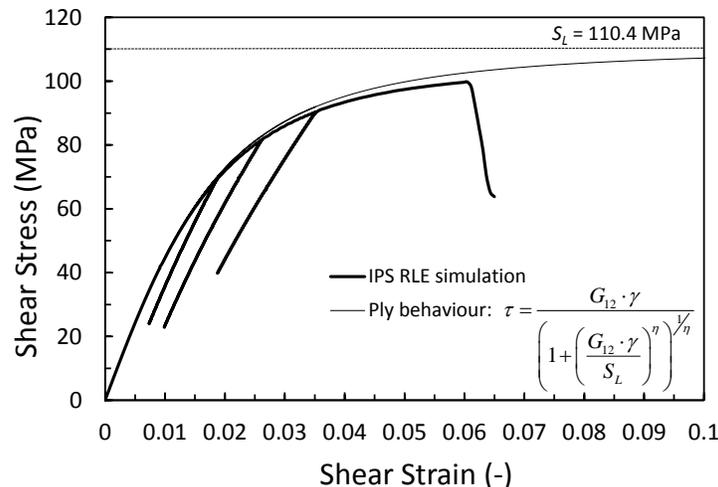


Figure 7: Stress-strain curves for the plain tension test. The appearance of the relevant damage events are marked with arrows in the figure. Ply in-plane shear strength, $S_L = 110.4$ MPa, measured by means of the Short Beam Test [37]. Numerically-obtained laminate In-Plane Shear Strength, IPSS = 99.7 MPa (at $\gamma_{pl} = 0.04\%$). Experimentally-obtained $[\pm 45]_s$ specimen IPSS = 91.6 MPa (SD = 2.51 MPa) corresponding to $\gamma_{pl} = 0.05$ [37]. Ply in-plane shear modulus $G_{12} = 4.9$ GPa. Ramberg-Osgood exponential, $\eta = 1.9$.

410 experiment. As result, the numerically obtained In-Plane Shear Strength,
 411 IPSS = 99.7 MPa is higher than the average value obtained experimentally
 412 with the IPS experiment, IPSS = 91.56 MPa [37].

413 The numerically-obtained unloading-reloading behaviour of the RLE is
 414 also represented in Figure 7 to demonstrate that the PBCE, and the consti-
 415 tutive ply model, work well under these circumstances.

416 4. Conclusion

417 Special-purpose Periodic Boundary Condition Elements (PBCE) were
 418 proposed to impose Periodic Boundary Conditions (PBC) to general Rep-
 419 resentative Volume Elements (RVE) in FE solvers based on dynamic explicit
 420 time integration. This approach solves the issue of spurious oscillations re-
 421 sulting from the application of the traditional PBC approach in the explicit
 422 FEM, overcomes limitations in the number of constraint relations and al-
 423 lows gains in computational efficiency. The PBCE formulation was imple-
 424 mented by means of a user-defined element through a VUEL subroutine in
 425 Abaqus/Explicit [25]. The reliability and applicability of the approach were

426 demonstrated in the framework of multiscale computational analysis of com-
427 posites. First, the PBCE method in combination with RVE were applied to
428 micromechanical homogenization of unidirectional FRP yarns or plies. The
429 correlation between traditional PBC in implicit integration and PBCE in the
430 explicit FEM was remarkable. Then, PBCE in combination with Representa-
431 tive Laminate Elements (RLE) were proposed and validated for the purpose
432 of homogenization of laminate behaviour through computational mesome-
433 chanics to expedite the virtual testing of composite materials and eliminate
434 undesired effects of coupon-based experiments.

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443 **Data availability**

444 The raw/processed data required to reproduce these findings cannot be
445 shared at this time as the data also forms part of an ongoing study.

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