

RESEARCH ARTICLE

Gravure printing with a shear rate dependent ink

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Abstract

Gravure printing is a type of printing method that uses metal cylinders with engraved cells that hold ink. The ink is transferred directly to the paper or other material by pressing it against the cylinder. The flow associated with gravure printing includes a flow in a liquid bridge formed in the contact region of the cylinders and a thin film coating flow of the ink. The flow is governed by viscous and capillary forces. In many cases, the flow is unstable, which leads to the formation of instability patterns on the printed surfaces. The analysis of these instabilities is a very challenging problem, especially since industrial inks are usually rheologically complex.

In this experimental and theoretical study, the flow of inks on a rotating cylinder is analyzed, accounting for the shear rate-dependent liquid viscosity. A theoretical solution for the film flow allows us to predict the width of the liquid bridge between two cylinders. Moreover, it is shown that the measured characteristic size of the printed pattern is of the same order as the predicted liquid bridge width. We observe a nearly linear dependence of pattern size and liquid bridge width.

Impact Statement Gravure printing is a high-speed and high-quality printing technique that uses mechanically engraved metal cylinders to transfer ink to the substrate. It is suitable for a wide range of applications, including packaging, magazines, wallpaper, banknotes, printed electronics, and functional coatings. Gravure printing is known for its high production speed and volume. Unlike offset printing, gravure printing transfers ink directly from the cylinder to the substrate without an intermediate cylinder.

The quality of printing can be reduced due to various types of interfacial instabilities that lead to the emergence of patterns on the coated surface. The mechanisms of these instabilities are currently being investigated for Newtonian liquids. However, most real inks are non-Newtonian. In this study, the flow in thin liquid films of inks with shear-rate-dependent viscosity is analyzed theoretically and the pattern in the coated layer is investigated using an industrial gravure printing machine and a gravure printing research platform. The theory allows us to predict the width of the liquid bridge between the cylinders. It is shown that this length correlates very well with the characteristic size of the pattern.

1. Introduction

The processing of structured, thin liquid films on solid surfaces, i. e. printing, is one of the fundamental concepts of industrial production and economics. This is for the simple reason that printed surfaces have a direct impact on the perception and utility value of a product by human beings. At the same time, the expenses for raw materials and energy are small, because they are calculated in essentially two and

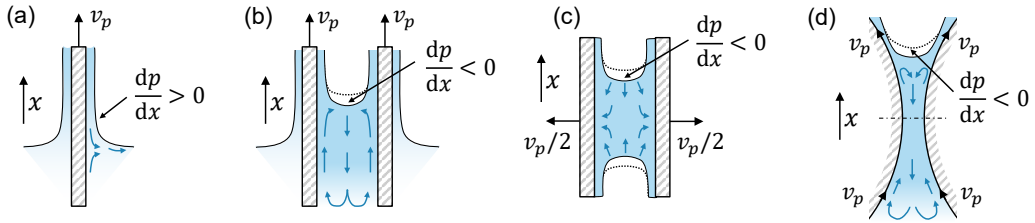


Figure 1. Liquid film formation by lateral plate extraction (a), extraction of two parallel plates (b), plate retraction (c), fluid splitting on rotating cylinders (d).

not in three dimensions. Large quantities of products can be assigned with additional value in a short time. Liquid dosing and distribution on solid surfaces are mechanically simple, by hand as well as by an autonomous machinery, the printing press (Kipphan, 2000). Once in contact with the solid, adhesion forces dominate the printed film, and fix the volatile liquid on its constraining target surface.

Basic configurations of liquid flow and motion of solid plates, leading to the formation of thin liquid coating films, are shown in Fig. 1. The configurations include lateral extraction of the plate from a liquid reservoir, lateral extraction of two parallel plates, normal retraction of two parallel plates, and fluid splitting between two axiparallel rotating cylinders. In all these geometries, the solid surface is wetted with the liquid and dragged out of the reservoir with a specific velocity v_p . The configuration in Fig. 1(d) is associated with the technology of gravure printing, considered in this paper.

Gravure printing is a high-quality and high-speed printing method that uses metal cylinders with mechanically engraved cells to transfer ink to the substrate, which is applicable to a long list of applications (Huang & Zhu, 2019; Kapur et al., 2011; Grau et al., 2016; Tiara et al., 2022). It is characterized by high production speed and volume. Industrial devices can achieve speeds of up to 15 m/s. This technology is suitable for long-run printing of packaging, magazines, wallpaper, banknotes, and much more (Kipphan, 2000). Other applications include printed electronics and functional coatings (Grau et al., 2016). In the gravure printing process, the ink is transferred directly from the cylinder to the substrate without an intermediate cylinder, unlike offset printing.

A very detailed theoretical and experimental study of a Newtonian flow in the liquid bridge between two rotating cylinders, associated with the gravure printing process, can be found in Gaskell et al. (1995, 1998). The flow field is obtained using a lubrication approximation. Then the main geometrical parameters of the liquid bridge are predicted. The predictions agree very well with the experimental data. The model includes a matching of the curvature of the flow in a thin liquid film on a cylinder with that of the meniscus of the liquid bridge between the cylinders. The problem is analogous to the Levich & Landau (1942) problem.

The flow is mainly determined by the capillary number Ca

$$Ca \equiv v_p \frac{\mu}{\sigma}, \quad (1)$$

where v_p is the longitudinal velocity of the wall propagation, μ is the liquid viscosity and σ is surface tension. The width L of the liquid bridge, see Fig. 7, formed between two cylinders rotating with the same velocity v_p , is obtained in the form (Gaskell et al., 1995)

$$L \approx 1.24 \sqrt{R h_0} Ca^{-1/3}, \quad (2)$$

where R is the cylinder radius and h_0 is the thickness of the coated film, determined by the total volume per unit width of the transferred ink. This model agrees very well with the experimental data in a wide range of operating parameters.

Recently, a model (Dixit & Homsy, 2013) for dip coating has been developed, which accounts for the elasticity of the surfaces and can also be used for a better understanding of gravure printing.

Printing allows for the consistent production of fine and detailed images with a wide density range. However, as has been pointed out by (Kumar, 2015), liquid relaxation flows can give rise to instabilities under certain operating conditions, which results in spontaneous pattern formation. This is due to the presence of low-energy degrees of freedom at the highly volatile fluid meniscus at the liquid-air interface. In particular, these phenomena apply to the dynamic steady-state environment of a printing press, which keeps up a constant non-equilibrium, and provides plenty of time for an instability to grow in amplitude. Capillary leveling and waves, liquid bridge formation, and interfacial and wetting instabilities determine the behavior and the final state of the film (Sauer et al., 2018). Contrasting to this, phenomena like phase separation or evaporation, which depend on elevated energy scales, can be largely blanked out from understanding the process.

The fingering instability at the meniscus of a retracting liquid-air interface in the Hele-Shaw cell is very similar to the instabilities of the flows in porous media (Saffman & Taylor, 1958). The linear stability of the problem (Carvalho & Scriven, 1999) allows to explain how the particular length scale associated with the fingering instability evolves. Important contributions have been attributed to gravure printing from gravure roll coating research. Pulkrabek & Munter (1983) observed that the coating applied with a hashur gravure roll developed a periodic ribbing pattern with a period that was independent of the hashur period, but was essentially a function of coating layer thickness and velocity. Reverse coating, and the conditions of the ribbing instability, were studied by Coyle et al. (1990). An effort to a principal classification of the possible flow regimes goes back to Benkreira et al. (1994). The particular influence of the gravure pattern of the cylinder has been considered by Benkreira & Cohu (1998), Schwartz (2002), Kapur (2003), Hewson et al. (2006) and Hewson et al. (2009). Recent progress on dealing with one of the fundamental challenges of gravure printing and coating, i.e. the fact that there are interacting ink flows on very different length scales, was contributed by Raske et al. (2017). As a matter of fact, when considering ribbing pattern length scales in printing inks, one cannot neglect the effect of ink rheology, as typical printing inks show significant shear-thinning. This has been studied by Wu et al. (2018) and Wu et al. (2019).

The coupling between fluid bulk and meniscus renders the problem a non-linear one, even though the flow profiles can be adequately described by linear Stokes approximation. The instability can be characterized by scaling exponents, which are shifting with certain system parameters, such as surface elasticity or ink rheology. The point is that the finger instability in the Hele-Shaw cell is a limiting case of that which one finds in the printing press, but that the latter is also related to other instabilities such as the Rayleigh-Taylor one, or to even more complex mechanisms, which have not yet been fully elucidated. A recent discussion on the possible instabilities, that have been observed, has been presented by Roisman et al. (2023).

In this study, we characterize the pattern formation that accompanies gravure printing with industrial inks of shear rate-dependent viscosity. The viscosity of such inks can vary by several orders of magnitude, depending on the local instantaneous shear rate. This is why the existing models developed for Newtonian liquids (Gaskell et al., 1995, 1998) cannot be directly applied. The characteristic size of the pattern, produced by instabilities that accompany gravure printing on an industrial printing machine, is measured for a wide range of operating conditions, including the thickness of the coating film and the rotational velocity of the cylinders. An optically accessible printing research platform, which is equipped with a high-speed camera, is used to visualize the temporal evolution of the instability and yields measurements of the liquid bridge width. Additionally, to better understand the development of the instabilities leading to the pattern formation, liquid bridge stretching experiments between two parallel plates are performed.

Next, a theoretical model for the liquid flow is developed, which accounts for the complex rheology of the used printing liquids. The model allows to predict the width of the liquid bridge between the cylinders, formed during gravure printing. It is applicable to liquids of shear rate-dependent viscosity, including yield stress liquids.

Finally, it is shown that the size of the printing pattern is of the same order as the liquid bridge width and depends on it nearly linearly.

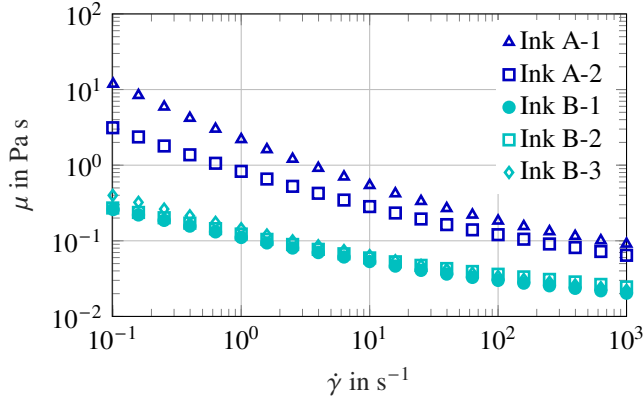


Figure 2. Viscosity μ as a function of the shear rate $\dot{\gamma}$ for inks A and B, as used in the different experimental setups: Industrial printing machine (-1), liquid bridge setup (-2), and printing research platform (-3).

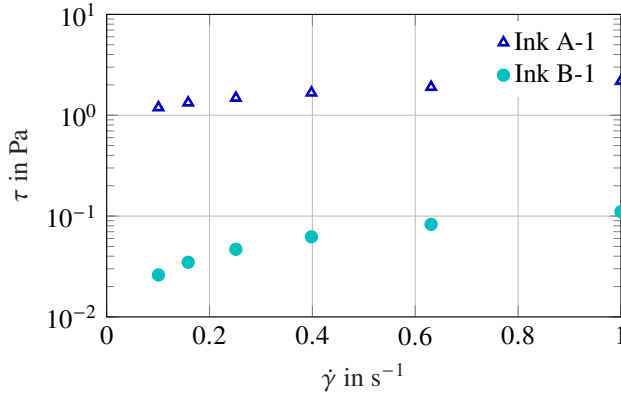


Figure 3. Shear stresses τ of inks A-1 and B-1 at low shear rates $\dot{\gamma} < 1 \text{ s}^{-1}$. The estimated yield shear stress for ink A-1 is $\tau_Y = 0.61 \text{ Pa}$, obtained by extrapolation of the data to $\dot{\gamma} = 0$.

2. Experimental method

Two different printing machines are used for the gravure printing experiments within this work: An industrial printing machine §2.1 and a printing research platform §2.2. The industrial printing machine serves as a tool for the creation of large-area printouts that show fingering patterns of different scales. The printouts are digitized and the distance between the fingers is measured. Since an industrial printing machine does not offer the possibility to measure the meniscus length, we additionally use our optically accessible printing research platform. There, measurements of the meniscus length are possible from *in situ* high-speed video recordings of the printing nip. The engraving of the printing form as well as the used printing inks are chosen the same on both printing machines, thus the experimental results from both machines are complementary. However, there also exist some major differences between the industrial printing machine and the printing research platform. First, the nip in the industrial printing machine has a cylinder-cylinder-geometry, whereas the printing research platform uses a cylinder-plate-geometry. Second, the industrial printing machine is fed with a paper substrate, which is pressed onto the rigid printing cylinder by an elastic, rubber-coated impression roller (*soft-on-hard* setup), whereas the printing research platform uses a rigid glass substrate, which is pressed to the rigid printing cylinder (*hard-on-hard* setup).

Table 1. Surface tension σ of all used fluids.

Fluid	σ in mN/m
Ink A-1	29.1 ± 0.9
Ink A-2	27.5 ± 0.1
Ink B-1	30.7 ± 0.2
Ink B-2	27.4 ± 0.2
Ink B-3	29.8 ± 0.1

Two different printing fluids are used for this work. The first fluid is called *ink A* here. It is a commercial, bluish-red, water-based gravure ink R0001235761 (Sun Chemical Group GmbH, Niedernhausen, Germany) in its delivery viscosity of around 30 seconds efflux time in a 4 mm DIN flow cup (DIN 53211). The second fluid is called *ink B*, which is the same bluish-red gravure ink but diluted with water to an efflux time of 14 seconds. Measurements of viscosity over the shear rate at 20 °C are performed on a rotational rheometer Lab+ (NETZSCH-Gerätebau GmbH, Selb, Germany) for both fluids after formulation. A cone-plate-geometry with a cone diameter of 60 mm and an opening angle of 1° is used.

Since the shelf-life of inks A and B is limited (ink A: several months, ink B: a few weeks), we have to use different batches and reformulations of the ink for the industrial printing experiments (§2.1), the experiments on the printing research platform (§2.2) and the liquid bridge stretching experiments (§2.3.2). This leads to slightly different viscosities of the inks among the different experimental setups. Measurements of viscosity over shear rate for all fluids used can be found in Fig. 2. The inks for the industrial printing machine are called inks A-1 and B-1, the inks for the liquid bridge setup are called inks A-2 and B-2 and the ink for the printing research platform is called ink B-3. Each data point in Fig. 2 is the mean of three measurements. The standard deviation of the three measurements is not plotted since it is around 1 to 3 % for all data points. For this study, we only use the viscosity measurements of inks A-1 and B-1 from the industrial printing experiments (Fig. 2), since these experiments are the focus of this work and are used to determine the characteristic size of the printed patterns ℓ .

The shear stresses $\tau = \mu\dot{\gamma}$ for relatively small shear rates $\dot{\gamma} < 1 \text{ s}^{-1}$ are shown in Fig. 3. Ink A-1 demonstrates the properties of a yield stress fluid. The estimated yield stress is $\tau_Y = 0.61 \text{ Pa}$. The yield stress for ink B-1 is smaller than 0.1 Pa and its exact value cannot be determined by the extrapolation of the existing data. The yield shear stress for ink B-1 is neglected in the further analysis.

The surface tension of all used fluids is measured using a drop shape analyzer DSA 100 (Krüss, Hamburg, Germany) in pendant drop mode at temperatures between 22 and 26 °C. 20 droplets are measured per fluid. The results are shown in Tab. 1.

2.1. Industrial gravure printing machine

Gravure printing experiments are performed on an industrial printing web-press Gallus RCS 330-HD (Gallus Ferd. Rüesch AG, St. Gallen, Switzerland). Printing velocities up to 2.667 m/s with a maximum printing width of 330 mm can be achieved with the machine. For this study, experiments with two different water-based gravure printing inks (inks A-1 and B-1) at printing velocities of 0.083, 0.167, 0.25, 0.5, 1, 1.5, and 2 m/s are conducted. The values in the unit m/s were converted from integer values in the unit m/min, which is a common unit in the printing industry, and then rounded to three decimal places. In printing form, an electromechanically engraved, chromium-plated copper printing cylinder with a radius 72.75 mm on a steel body is employed. The engraving has a raster angle of $\alpha_r = 59.35^\circ$, which corresponds to HELL engraving angle #2, and four different raster frequencies $f_r = [60, 70, 80, 100] \text{ lines/cm}$. The engraved cells are pyramid-shaped and have an opening angle of 120° , which equals the diamond stylus angle used for engraving.

The thickness of the printed film h_0 strongly depends on the volume of the engraved cells on the printing cylinder per area, the so-called dip volume V_{dip} . First, the mean volume \bar{V} of the engraved cells on the printing cylinder used in the industrial experiments is measured using a 3D optical profilometer

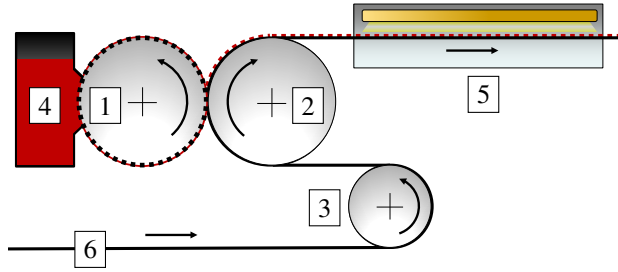


Figure 4. Sketch of the gravure printing unit within the industrial printing machine Gallus RCS 330-HD. 1) Printing cylinder, 2) pressure roller, 3) deflector roller, 4) doctor blade chamber filled with printing ink, 5) hot air and infrared drying unit, 6) substrate.

Table 2. Thicknesses of the printed film h_0 , estimated with equation (4).

f_r in lines/cm	h_0 in μm
60	7.4
70	6.2
80	5.2
100	3.5

PLu neox (Sensofar DACH, Langen, Germany) in confocal mode. For each of the four raster frequencies $f_r = [60, 70, 80, 100]$ lines/cm, three cells are measured and the average is used as the mean volume \bar{V} . The 3D surface data is analyzed in MATLAB 2022b, using the algorithm from Schäfer (2019). Second, the dip volume V_{dip} is calculated using equation (3) (Schäfer, 2020)

$$V_{\text{dip}} = \frac{\bar{V} f_r^2}{\sin 2\alpha_r}, \quad (3)$$

where V_{dip} is the dip volume, \bar{V} is the mean volume of three cells, f_r is the raster frequency and α_r the raster angle.

To estimate the thickness of the (wet) printed film h_0 , we use equation (4)

$$h_0 \approx 0.5V_{\text{dip}}. \quad (4)$$

We would like to emphasize that this is just a rough approximation of h_0 , since h_0 depends on the transfer volume. The transfer volume also is a fraction of V_{dip} , but it is dependent on Ca , see Grau et al. (2016). All determined values for h_0 are shown in Tab. 2.

Since the focus of this study is fingering instabilities, printed patches of size 64 mm x 180 mm with a tonal value of 100 % are used. This tonal value is equivalent to the highest possible transfer volume. At high transfer volumes, a continuous meniscus is formed in the printing nip, and the ink splitting regime of *lamella splitting* is observed, which comes along with fingering instabilities. These instabilities lead to finger patterns on the printed substrate. In contrast, at lower tonal values, we observe the regime of *point splitting*, which leads to dot patterns on the substrate. The printed dot patterns correspond to the engraved raster pattern on the printing cylinder (Brumm et al., 2023, 2021).

A sketch of the gravure unit of the industrial printing machine is shown in Fig. 4. The engraved cells on the printing cylinder (1) are filled with ink from the doctor blade chamber (4). The excess ink is wiped away by the doctor blade (with a doctor blade angle of -37°), which leaves only a thin lubrication film on the walls between the cells. The substrate (6), onto which the ink shall be deposited, first passes a deflector roll (3) and is then guided through the nip between the printing cylinder and pressure roller (2). The pressure roller applies a high pressure onto the substrate so that the ink from

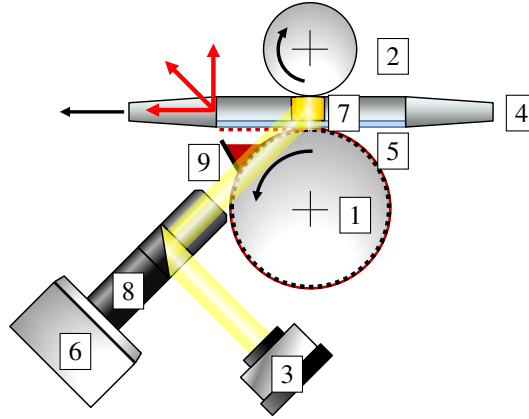


Figure 5. Sketch of the gravure printing research platform in front view. 1) Printing cylinder, 2) impression roller, 3) high power LED, 4) optically accessible substrate carrier, 5) glass substrate (light blue), 6) high-speed camera, 7) gold-plated prism (orange), 8) Navitar zoom objective, 9) doctor blade.

the cells is transferred to the substrate. In this work, we use a coated paper substrate Sigmakraft RTC Green (Mosaico Specialty Papers, Altavilla Vicentina, Italy) with a grammage of 70 g/m^2 . The printed layer of ink is subsequently dried in a hot air and infrared dryer (5).

The printed substrate is digitized using a color-calibrated flatbed scanner Epson Perfection V850 Pro (Seiko Epson Corporation, Suwa, Japan) at high resolution of 2400 dpi ($10.58 \text{ } \mu\text{m/px}$). The scans reveal spontaneous finger pattern formation phenomena created during fluid transfer in the printing nip. The distance ℓ between the fingers is determined with the method from Brumm et al. (2019), which uses one-dimensional fast Fourier transform (FFT) perpendicular to the printing direction and subsequent peak-finding within the averaged frequency spectrum. The algorithm is implemented in MATLAB 2022b using the `fft` function. For both printing fluids (inks A-1 and B-1), experiments with 28 parameter combinations of raster frequency f_r and printing velocity v_p are conducted. For each parameter combination, eight printed patches are analyzed.

2.2. Gravure printing research platform

The gravure printing research platform is the second experimental setup used for gravure printing experiments within this work. It complements the industrial gravure printing machine since it offers optical accessibility to the printing nip and *in situ* observation of the meniscus shape. Measurements of the meniscus length are possible as well. The development of our research platform is inspired by the experimental setup of Schäfer et al. (2019); Schäfer (2020).

A sketch of the gravure printing research platform in front view is found in Fig. 5 and a side view is shown in Fig. 6. The engraved cells of the gravure printing cylinder with a radius 110 mm (1) are filled using a doctor blade (9), where printing fluid has been pipetted onto. The key component of the research platform is the custom-built, optically accessible substrate carrier (4), onto which a Gorilla glass substrate (Corning Incorporated, Corning, NY, USA) of size 207 mm x 90 mm x 1.1 mm is mounted (5). Inside the substrate carrier, there is a gold-plated prism (7), which reflects the white light from the high-power, cold-light light-emitting diode (LED) LED8 (StarLight Opto-Electronics GmbH & Co. KG, Nuremberg, Germany) (3). In the setup, a Navitar zoom objective 12X (Navitar, Rochester, NY, USA) (8) with coaxial illumination and aperture is used in combination with a high-speed camera Fastcam Nova S12 (Photron Deutschland GmbH, Reutlingen, Germany) (6). The optically accessible substrate carrier is pressed onto the printing cylinder with a pressure roller while it passes the nip with

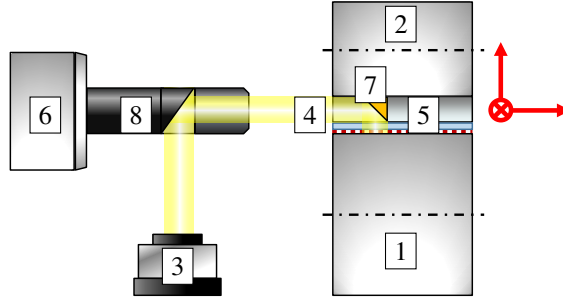


Figure 6. Sketch of the gravure printing research platform in side view. 1) Printing cylinder, 2) impression roller, 3) high power LED, 4) optically-accessible substrate carrier, 5) glass substrate (light blue), 6) high-speed camera, 7) gold-plated prism (orange), 8) Navitar zoom objective.

the printing velocity v_p . Printing pressures between 2 and 16 N/mm and printing velocities between 0.1 and 3 m/s can be realized.

The high-speed camera is operated with a frame rate of 30000 fps, an aperture time of 1/60000 s, and a resolution of 640 px x 480 px. Thus, a field of view of around 22 mm x 16 mm with a spatial resolution of 34 $\mu\text{m}/\text{px}$ is obtained. As printing fluid, ink B-3 is used. Printing velocities of 0.25, 0.5, 1, and 2 m/s are used at a printing pressure of 2 N/m and a doctor blade pressure of 0.3 N/mm. The doctor blade angle is 64°. For the research platform, the same engraving as in the industrial printing experiments is applied to the printing cylinder (electromechanical engraving, HELL engraving angle #2, stylus angle 120°). Printing experiments are performed on patches of the printing cylinder, which are engraved with a raster frequency f_r of 60, 80, and 100 lines/cm and a tonal value of 100 %. For this study, only the meniscus length L is of interest, although it would also be possible to determine the distance ℓ between the fingers from the high-speed videos.

2.3. Observations of instabilities during printing

2.3.1. Pattern of instabilities observed after gravure printing

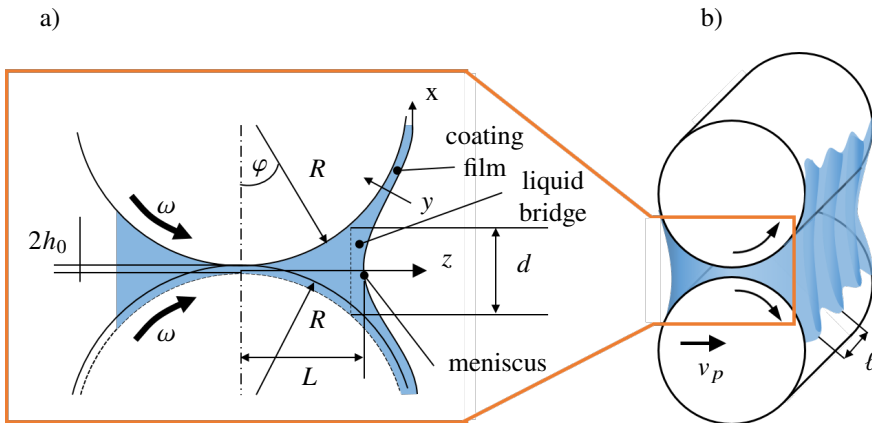


Figure 7. Flow configuration associated with gravure printing. Sketch of the rotating cylinders, liquid bridge of ink, and formation of a coating film. a) Close-up sketch showing variables, b) Three-dimensional sketch showing fingering at the meniscus between rolls.

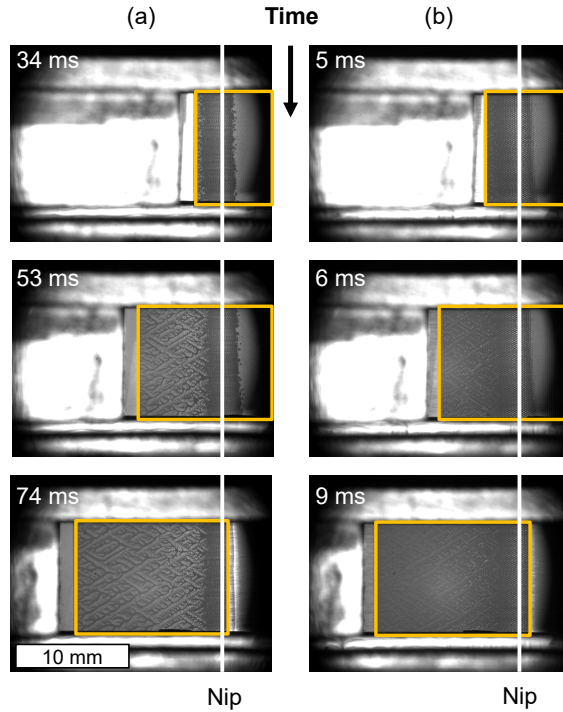


Figure 8. Exemplary high-speed video frames recorded with the printing research platform. Time evolves from top to bottom, as indicated by the time stamps in ms. The printing velocities v_p and raster frequencies f_r are: (a) $v_p = 0.25$ m/s, $f_r = 100$ lines/cm, and (b) $v_p = 2$ m/s, $f_r = 60$ lines/cm. The region of interest (ROI) is indicated in each frame with a yellow rectangle, which moves from right to left with v_p . The ROI shows finger patterns, which are clearly visible in (a) and smaller, with less contrast and more blurred in (b). The printing nip between the gravure cylinder and substrate is marked as a white line.

The flow configuration associated with the gravure printing is shown schematically in Fig. 7. The flow includes a liquid bridge formed between two rotating cylinders. The width of the liquid bridge L is determined by the total volumetric flux of the ink, by tangential speed v_p of rotation of the cylinders, and by the ink's rheological and interfacial properties. The main geometrical and kinematic parameters determining the problem are defined in Fig. 7(a). The shape of the meniscus is influenced by the appearance of the interfacial instabilities. These instabilities are shown schematically in Fig. 7(b). The thin deposited liquid films on the cylinders are not shown in Fig. 7(b) for better clarity.

A set of high-speed videos of the flow in the liquid bridge at various operational conditions are shown in Fig. 8. The images in Fig. 8 also show the emergence of geometric finger patterns on the printed surfaces. A zoomed-in image of the region of interest can be found in Fig. 9.

The location of the nip is determined using extra high-speed videos without printing fluid. There, the nip can be located due to light refraction at the contact region of the glass substrate and gravure cylinder.

One example of a typical scan of a printed sample from the industrial gravure printing machine is shown in Fig. 10. The zoomed-in images show the appearance of fingering patterns, which are oriented roughly along the printing direction. The pattern depends on the raster frequency f_r , which determines also the volumetric flux of the ink.

The appearance of the finger patterns is also strongly influenced by printing velocity and choice of printing ink, see Fig. 11. The higher the printing velocity, the lower is the distance between the fingers

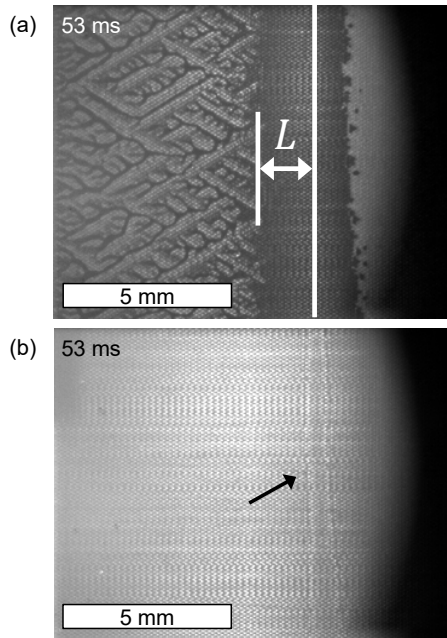


Figure 9. High-speed video frames cropped to the ROI. (a) Zoomed-in frame from Fig. 8(a) at time 53 ms. The image shows the meniscus width L , which is the distance between the nip and unstable meniscus on the left side of the nip. The meniscus at the right side of the nip is stable. Printing direction from left to right. (b) High-speed recording without printing fluid with $v_p = 0.25$ m/s and $f_r = 100$ lines/cm. It is used to determine the position of the nip, which shows characteristic light refraction at the contact area between the glass substrate and the printing cylinder (see arrow).

and the pattern appears more blurred. In comparison to ink A-1, ink B-1 produces finger patterns with a larger finger distance. This is attributed to the lower viscosity of ink B-1.

2.3.2. Finger formation during fast liquid bridge stretching

The main subject of this paper is the study of finger formation during gravure printing. However, this technology does not always allow direct observation of the formation and development of the instabilities leading to the formation of finger-like structures on the printed surfaces. This problem becomes even more challenging since the rheology of the inks is non-Newtonian since its viscosity depends on the strain rate.

The development of the fingering instability can be best observed in experiments with a much simpler geometry and well-investigated flows in a so-called lifting of a Hele-Shaw cell. This configuration is shown schematically in Fig. 1(c). A liquid volume in a thin initial gap between two parallel solid plates is stretched since one of the plates starts to move with a constant acceleration. The flow of the liquid in the gap is observed using a high-speed video system through a fixed transparent plate. The details of the experimental setup and the description of the fingering instability for Newtonian liquids can be found in the literature (Brulin et al., 2020).

Two inks, ink A-2, and ink B-2, have been used in this study for the liquid bridge stretching experiments. The dependence of the ink viscosity on the shear rate is shown in Fig. 2.

Exemplary results of the bridge stretching are shown in Fig. 12, which are generated with the top view camera. In all the considered cases, the stretching leads to the instability of the liquid interface and subsequent fingering. Several typical sizes characterize the final observed pattern of instabilities. One is the smallest distance ℓ^* between the fingers near the initial bridge edge. The second size is the

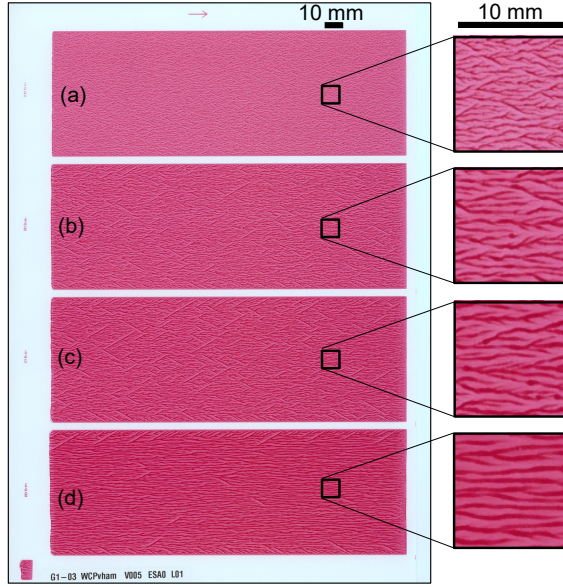


Figure 10. Exemplary scan of a printed sample from the industrial gravure printing machine, printed with ink B-1 at $v_p = 0.083$ m/s. Printing direction from right to left. The scan shows four full-tone patches of size 64 mm x 180 mm with different raster frequencies f_r : (a) 100 lines/cm, (b) 80 lines/cm, (c) 70 lines/cm, (d) 60 lines/cm. Zoomed-in images of the fingering patterns are shown on the right. The lower the raster frequency f_r , the larger the distance between the fingers ℓ .

typical finger length. While the finger length L^* is of the same order as the initial bridge radius R_0 , the distance between the fingers is determined by the length of the most unstable mode associated with the interfacial instability.

The mechanisms leading to the appearance of the interfacial instabilities, relevant to this study, are often explained by the balance of the forces associated with the positive pressure gradient at the interface and capillary forces. This includes well-known Rayleigh-Taylor instability, driven by the pressure gradient caused by the gravity or acceleration, and Saffman-Taylor instabilities (Saffman & Taylor, 1958) in which the pressure gradient at the interface is the result of the liquid flow through a porous media. The same mechanism leads also to the emergence of fingers in a liquid bridge formed between two plates, separating with a constant acceleration (Roisman et al., 2023). The typical distance l_{fingers} between the fingers is scaled well as

$$l_{\text{fingers}} \sim \sqrt{\frac{\sigma}{p_{,n}}}, \quad (5)$$

where the pressure gradient in the axisymmetric case is $p_{,n} = \partial p / \partial r$. This scaling agrees well with numerous experimental data (Brulin et al., 2020), including the data from this study, shown in Fig. 12. However, the theoretically predicted distance l_{fingers} is two orders smaller than the observed distance of the fingers on the printed samples, as shown in Fig. 11. Therefore, some additional physical factors play a role in the formation of these fingers. This is the topic of further analysis.

3. Flow of an ink of shear rate dependent viscosity

In this section, a flow of ink is considered in two main regions. First, the "inner solution" in the thin coating flow is analyzed accounting for the viscosity dependence on the local shear rate. Next, the inner

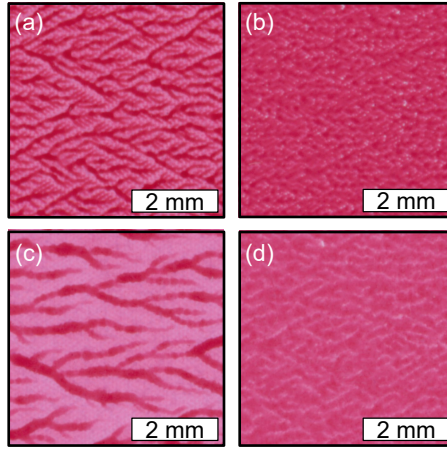


Figure 11. Exemplary finger patterns from the industrial gravure printing machine, printed with different combinations of printing velocity and printing ink: (a) $v_p = 0.083$ m/s, ink A-1, (b) $v_p = 2$ m/s, ink A-1, (c) $v_p = 0.083$ m/s, ink B-1, and (d) $v_p = 2$ m/s, ink B-1. The raster frequency is always $f_r = 100$ lines/cm. The printing direction goes from right to left.

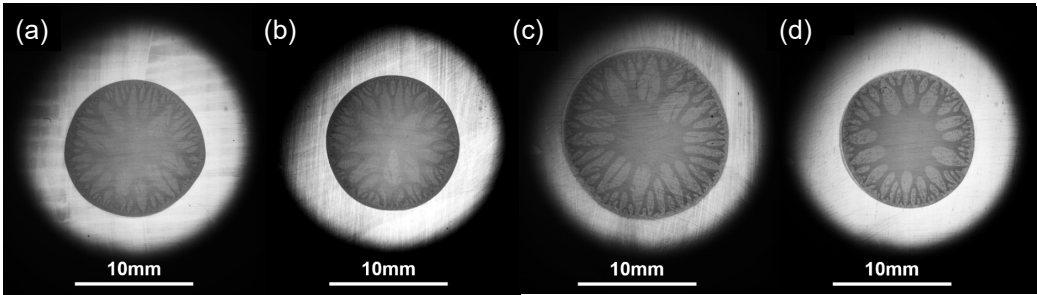


Figure 12. Liquid bridge stretching between two parallel plates. Examples of fingering after stretching of industrial inks used for gravure printing. The initial gap thickness H_0 is in the range from 40 to 55 μm in all presented experiments. The plate acceleration and the liquid are (a) $a = 5 \text{ ms}^{-2}$, ink A-2, (b) $a = 10 \text{ ms}^{-2}$, ink A-2, (c) $a = 5 \text{ ms}^{-2}$, ink B-2, (d) $a = 10 \text{ ms}^{-2}$, ink B-2, respectively.

solution is matched with the outer solution in the liquid bridge between two cylinders. The solution allows to estimate the width of the liquid bridge under various operating conditions.

Consider a steady liquid flow in a liquid bridge between two symmetric cylinders of radius R rotating with the same angular velocity ω . The sketch of the flow in the bridge is shown in Fig. 7.

The smallest distance between the cylinders at the symmetry plane is denoted $2h_0$. The distance d between the cylinders is determined by the transverse z coordinate. For small distances from the cylinders' symmetry plane the expression for d can be linearized, which yields

$$d = 2h_0 + \frac{z^2}{R}. \quad (6)$$

The distance of the liquid bridge meniscus to the axis plane is denoted L . The motion of the cylinders leads to their wetting by the printing liquid. The flow in the coating liquid film is similar to the well-known dip coating problem (Levich & Landau, 1942). This flow will be considered in the following section, accounting for the complex rheology of the industrial printing liquids.

3.1. Thin film flow

Consider a steady flow in a thin liquid film of ink, extruded from a steady meniscus by a rotating cylinder, as shown in Fig. 7(a). Denote a lateral coordinate x along the film and the normal coordinate y . The origin of the coordinate system is fixed at the free film interface. The system is curvilinear. However, since the film thickness $h(x)$ is much smaller than the cylinder radius R_0 , the curvature effects are neglected in this theory. The considered problem is steady, therefore the total volumetric flux Q per unit film width

$$Q = h_0 v_p \quad (7)$$

is constant in each film cross-section

$$\int_0^h u(y) dy = Q, \quad (8)$$

where h_0 is the asymptotic value for the film thickness far from the meniscus.

The long-wave approximation of the momentum balance equation yields

$$p_{,x}(x) = \frac{\partial \tau(\dot{\gamma})}{\partial y}, \quad \dot{\gamma} = u_{,y}, \quad \tau(\dot{\gamma}) = \mu(\dot{\gamma})\dot{\gamma}, \quad (9)$$

where the viscosity $\mu(\dot{\gamma})$ and the shear stress $\tau(\dot{\gamma})$ are functions of the shear rate $\dot{\gamma}$, which can be determined for a specific liquid. This approximation is based on the assumption, typical for the analysis of thin films, that the velocity gradients in y directions are much larger than the velocity gradients in the x directions. The latter thus can be neglected.

The no-slip boundary conditions at the wall $y = h$ and the zero-shear stress conditions at the free film interface $y = 0$ yield

$$u = v_p \quad \text{at} \quad y = h, \quad (10)$$

$$\dot{\gamma} = 0 \quad \text{and} \quad u = u_0 \quad \text{at} \quad y = 0, \quad (11)$$

where u_0 is an unknown velocity at the film free interface.

Note also that the momentum of the flow in the y direction, normal to the wall, is negligibly small. Therefore, also the pressure gradient $p_{,y}$ can be neglected. The pressure p is, therefore, a function only of the x coordinate, and is uniform along the film cross-section.

$$p_{,y} = 0. \quad (12)$$

The solution of (9) for the velocity field is obtained with the help of the conditions (10)-(12) in the form

$$y(\dot{\gamma}) = \frac{\tau(\dot{\gamma})}{p_{,x}}, \quad (13)$$

$$u(\dot{\gamma}) = u_0 + \frac{1}{p_{,x}} \int_0^\tau \dot{\gamma} d\tau. \quad (14)$$

We note that the shear rate $\dot{\gamma}$ is not constant across the liquid film, but is a function of y and the still unknown velocity u_0 of the film interface. The dissipation density function $f(\dot{\gamma}) = \int_0^\tau \dot{\gamma} d\tau$, which appears in the right-hand side of equation (14), and the shear stress $\tau(\dot{\gamma})$ encode the particular liquid

rheological properties of the film as a function of the shear rate $\dot{\gamma}$

$$\tau(\dot{\gamma}) \equiv \dot{\gamma} \mu(\dot{\gamma}), \quad (15)$$

$$f(\dot{\gamma}) \equiv \int_0^{\dot{\gamma}} \dot{\gamma} d\tau = \dot{\gamma} \tau(\dot{\gamma}) - \int_0^{\dot{\gamma}} \tau(\xi) d\xi, \quad (16)$$

where ξ is a dummy variable for integration.

Denote by $\dot{\Gamma}$ the limiting value of the shear rate $\dot{\gamma}$ at the wall $y = h$. Equations (13) and (14) can be written for the point $y = h$ with the help of the boundary conditions (10)-(11) in the following form

$$h = \frac{\tau(\dot{\Gamma})}{p_{,x}}, \quad (17)$$

$$u_0 = v_p - h\dot{\Gamma} + h \frac{\int_0^{\dot{\Gamma}} \tau(\xi) d\xi}{\tau(\dot{\Gamma})}. \quad (18)$$

Next, the definition (8) of the flux in the liquid film allows to write the expression for Q in terms of $\dot{\gamma}$ and the interface shear rate $\dot{\Gamma}$

$$Q = \frac{1}{p_{,x}} \int_0^{\dot{\Gamma}} u(\dot{\gamma}) \tau'(\dot{\gamma}) d\dot{\gamma}. \quad (19)$$

Substituting (14) and (17) in expression (19) yields

$$Q = u_0 h + \frac{h^2}{\tau^2(\dot{\Gamma})} \int_0^{\dot{\Gamma}} f(\dot{\gamma}) \tau'(\dot{\gamma}) d\dot{\gamma}. \quad (20)$$

Let us introduce for simplicity a function

$$\mathfrak{J}(\dot{\Gamma}) = \frac{h v_p - Q}{h^2}. \quad (21)$$

This function can also be expressed in terms of $\dot{\Gamma}$ and the rheological properties of the liquid. The expression is obtained by accounting for expressions (18) and (20) in (21)

$$\mathfrak{J}(\dot{\gamma}) = \dot{\gamma} - \frac{\int_0^{\dot{\gamma}} \tau(\xi) d\xi}{\tau(\dot{\gamma})} - \frac{\int_0^{\dot{\gamma}} f(\xi) \tau'(\xi) d\xi}{\tau^2(\dot{\gamma})}. \quad (22)$$

Function \mathfrak{J} allows us to determine the values of $\dot{\Gamma}$ and the corresponding wall shear stress $\tau(\dot{\Gamma})$ for a given volumetric flux

$$\dot{\Gamma} = \mathfrak{J}^{-1} \left(\frac{h v_p - Q}{h^2} \right), \quad \tau(\dot{\Gamma}) = \tau \left[\mathfrak{J}^{-1} \left(\frac{h v_p - Q}{h^2} \right) \right], \quad (23)$$

where $\mathfrak{J}^{-1}(\cdot)$ is an inverse function of $\mathfrak{J}(\cdot)$. In Fig. 13 the function $\tau[\mathfrak{J}^{-1}(\xi)]$, computed using (23) and the rheological properties of the corresponding liquid, is shown as a function of a dummy variable for integration ξ for both inks, used for the gravure printing experiments.

Next, the long-wave approximation of the Young-Laplace equation, valid for the case $h_{,x} \ll 1$, can be written in the form $p = -\sigma h_{,xx}$, where σ is surface tension. The evolution equation for the film thickness $h(x)$ is thus obtained by substitution of this relation in (17), and accounting for (23)

$$\sigma h_{,xxx} + \frac{1}{h} \tau \left[\mathfrak{J}^{-1} \left(\frac{h v_p - Q}{h^2} \right) \right] = 0. \quad (24)$$

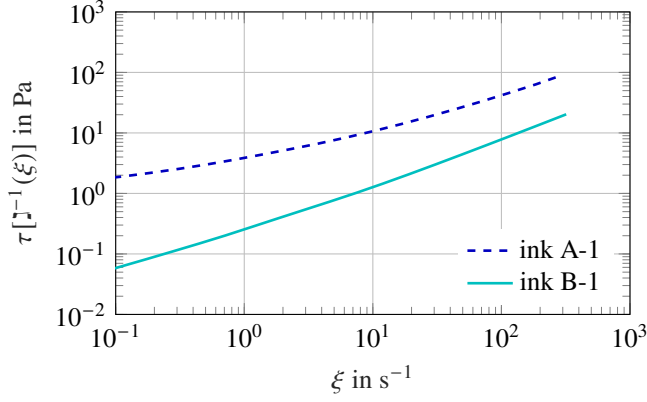


Figure 13. Function $\tau[\mathfrak{I}^{-1}(\xi)]$ computed for inks A-1 and B-1. This function is used in the expression for the wall stress (23) with $\xi = (h v_p - Q)/h^2$.

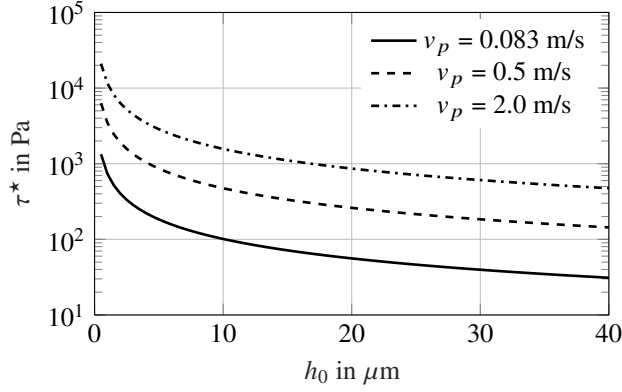


Figure 14. Characteristic shear stress τ^* in the film flow, defined in equation (26), computed for ink B-1 for different cylinder velocities v_p .

The argument $v_p(h - h_0)/h^2$ in the right-hand side of (24) has the maximum

$$v_p \frac{h - h_0}{h^2} = \frac{v_p}{4h_0}, \quad \text{at } h = 2h_0. \quad (25)$$

Therefore, a characteristic shear stress τ^* can be determined for a certain liquid rheology

$$\tau^*(h_0, v_p) = \tau \left[\mathfrak{I}^{-1} \left(\frac{v_p}{4h_0} \right) \right]. \quad (26)$$

Fig. 14 shows the characteristic shear stress τ^* in the film flow, computed for ink B-1 for different cylinder velocities v_p .

Moreover, equation (24) allows to define a characteristic length scale λ along the x axis

$$\lambda = \left[\frac{\sigma h_0^2}{\tau^*} \right]^{1/3}. \quad (27)$$

Now, the governing equation for the evolution of the film thickness can be written in the dimensionless form

$$H(X)H_{,XXX}(X) - T \left[\frac{(H-1)}{H^2} \frac{v_p}{h_0} \right] = 0, \quad (28)$$

where the dimensionless variables and functions are defined here

$$h = H(X)h_0, \quad x = -X\lambda, \quad (29)$$

$$\tau [\mathfrak{I}^{-1}(\xi)] = T(\xi)\tau^*. \quad (30)$$

Note that the sign of X is changed relatively to x to simplify computations.

The ordinary differential equation (28) can be integrated numerically for known values of Q , v_p , and the rheology of the liquids.

At high positive values of X and H , associated with the meniscus flow of the liquid bridge, the value of the second derivative $H_{,XX}$ asymptotically approaches a constant value denoted K , which is the dimensionless curvature of the meniscus. The value of the curvature is therefore

$$\kappa = K \frac{h_0}{\lambda^2}. \quad (31)$$

The value of K depends on τ^* and the ratio v_p/h_0 . For the liquids, used in this study, the computed value of K only very weakly depends on v_p/h_0 . This value is $K \in [1.667 - 1.67]$ for ink B-1 and is $K \in [1.6 - 1.66]$ for ink A-1 if $10^4 < v_p/h_0 < 10^7$. In further analysis, a constant average value $K = 1.67$ will be taken without significant precision loss.

Limiting case of Newtonian liquids

It can be shown that for a Newtonian liquid with constant viscosity $\mu = \mu_N$, the expression (22) for \mathfrak{I} can be reduced to $\mathfrak{I}_N(\dot{\gamma}) = \dot{\gamma}/3$. This expression with the help of (21) allows to derive the expression for the wall shear rate in the liquid film of Newtonian liquid

$$\dot{\Gamma} = 3 \frac{h v_p - Q}{h^2} \quad \text{if } \mu_N = \text{constant}. \quad (32)$$

Taking into account that $\tau = \mu_N \dot{\Gamma}$, the expression (24) is reduced with the help of (32) to the well-known evolution equation (Levich & Landau, 1942)

$$\sigma h_{,xxx} + 3\mu_N \frac{h v_p - Q}{h^3} = 0, \quad \text{if } \mu_N = \text{constant}. \quad (33)$$

This equation is not used in further analysis since the liquids used in the experimental part of the study are non-Newtonian. The limiting case of Newtonian liquid is considered here only to confirm the validity of the more general equation (24) and to make sure that no term is lost in complicated algebraic manipulations in this section.

3.2. Meniscus region of the liquid bridge and instabilities of coating

The curvature of the liquid film determines the position L of the meniscus of the liquid bridge. This distance is defined in Fig. 7.

For given liquid properties, thickness h_0 and velocity v_p , the distance L could be estimated from (6), taking into account that from geometrical considerations $d \approx 2/\kappa + 2h_0$. Using expressions (6) and (31)

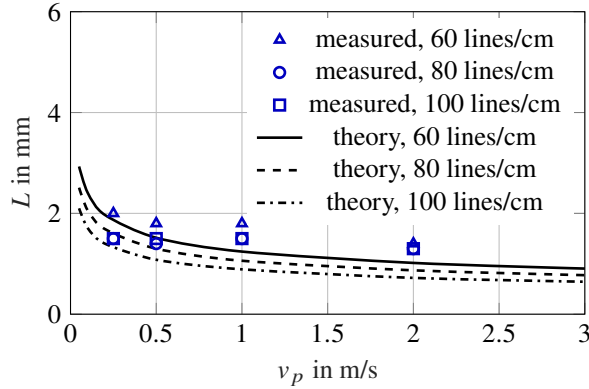


Figure 15. Meniscus length L over printing velocity v_p . The measured values for L come from selected high-speed observations of experiments with ink B-3 on the printing research platform. Theoretic values of L are computed using equation (34).

yields

$$L = \left[\frac{\sigma h_0^2}{\tau^*} \right]^{1/3} \sqrt{\frac{2R}{Kh_0}}. \quad (34)$$

Expression (34) has been obtained using an assumption that the curvature of the meniscus is constant for a certain case. The same assumption has been made in the theoretical study [Gaskell et al. \(1995\)](#) which allows to successfully predict the length L for Newtonian liquids. This assumption is valid only if the pressure gradient at the meniscus is not high enough to distort the meniscus shape. The pressure gradient in a viscous flow can be roughly estimated as $p_{,z} \sim \mu v_p / d^2$, where d is the gap width. Therefore, the corresponding capillary length Δ_c in the meniscus can be defined as

$$\Delta_c = \sqrt{\frac{\sigma}{p_{,z}}} = d \sqrt{\frac{\sigma}{\mu v_p}} \quad (35)$$

The meniscus distortion from a cylindrical shape will be small if the ratio $d/\Delta_c = Ca^{1/2}$ is small. Here Ca is the capillary number defined in (1). The observed length of the meniscus is approximately $L \approx 2$ mm. Therefore the width d , estimated from the geometrical considerations for the cylinder radius $R \approx 73$ mm is $55 \mu\text{m}$. The typical shear rate in the liquid bridge for $v_p = 2$ m/s is therefore $\dot{\gamma} \sim v_p/d \sim 10^3$ 1/s. The liquid viscosity at these shear rates is $\mu \sim 10^{-2}$ Pa s. Now, the capillary number, calculated for these values from (1) is $Ca \approx 0.3$. This is why the assumption of constant curvature is valid in our range of experimental parameters.

In this study, the value of L was estimated on the printing research platform. The values were determined from selected high-speed observations, as shown in Fig. 9. Unfortunately, the determination of the positions of the finger pattern tips on the cylinders, corresponding to $x = 0$ was not feasible with truly satisfactory precision. Therefore, in this study, we confined to only tentative estimation of the values of L , and was measured for various velocities v_p and different raster frequencies f_r . In Fig. 15, the measured values of L are compared with the theoretical predictions (34). The theory predicts very well the order of values for L , and explains the dependence of L on the cylinder velocity v_p and the film thickness h_0 . As mentioned above, some discrepancies can be attributed to the relatively limited precision of the measurements. Moreover, the theory does not account for the inertial effects and viscous stresses in the shearing liquid bridge between the cylinders. By this effect, the shape of the meniscus could have been deformed, and thus shift the true value of L . Nevertheless, the model takes into account the major physical influencing factors.

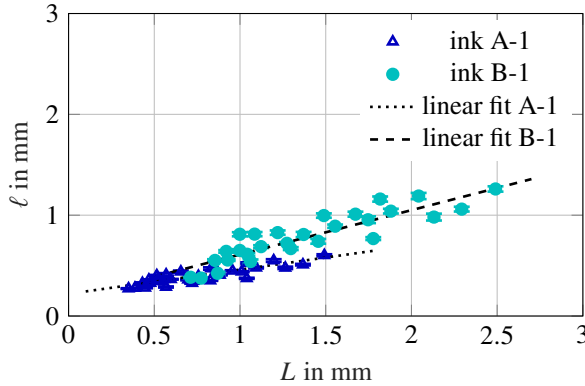


Figure 16. Dependence of the measured characteristic size of the printed pattern ℓ on the theoretically predicted width L of the liquid bridge, determined by equation (34). Each data point is based on eight measurements of ℓ . The error bars indicate the standard deviation of the measurements. The linear fit A-1 is $\ell = 0.22 + 0.24L$ and the linear fit B-1 is $\ell = 0.17 + 0.44L$.

Next, one can assume that the mechanisms of pattern formation in gravure printing are similar to the fingering instabilities observed as a result of the liquid bridge stretching, as shown in Fig. 12. In all the considered cases, the characteristic length of the observed tree network is comparable with the initial bridge radius R_0 . Following this assumption, the characteristic size ℓ of the printed pattern should be comparable with the liquid bridge width L for gravure printing. In Fig. 16 the measured values of ℓ are shown as a function of the theoretically predicted L for the range of the thicknesses $h_0 \in [3.5 - 7.4] \mu\text{m}$ and cylinder velocities $v_p \in [0.083 - 2] \text{ m/s}$. Each data point is based on eight measurements of ℓ and the standard deviation of the measurements is around 1 to 3 %, as indicated by the error bars. In all the considered cases, for both studied liquids, the ratio ℓ/L is between 0.35 and 0.8. Moreover, the values of ℓ clearly almost linearly correlate with L

$$\ell \approx \beta L. \quad (36)$$

The linear fit for ink A-1 yields $\beta = 0.24$ and the linear fit for ink B-1 yields $\beta = 0.44$, see Fig. 16.

The observed distance between ℓ the fingers observed during the printing process, cannot be modeled using the approaches based on the Saffman-Taylor-like instability. However, it correlates very well with the theoretically predicted meniscus length L . Moreover, their values are of the same order for all our experiments. Therefore, the appearance of the fingers in the printing samples is probably associated with the meniscus oscillations in the direction z , defined in Fig. 7. This is why the values of ℓ are limited by L .

4. Conclusions

This experimental study shows that the mechanisms of pattern formation resulting from the liquid bridge stretching between two parallel plates and from gravure printing can be understood as similar physical concepts. In both cases, the pattern consists of tree networks of finger-like instabilities of the printed surface.

To analyze the flow associated with gravure printing, we analyzed the meniscus in the printing nip of a gravure press and the patterns forming there. There are two characteristic length scales: the distance L of the meniscus from the center of the printing nip, and the finger distance ℓ of the viscous finger pattern, both being direct functions of printing velocity, transfer volume of the gravure pattern, and the rheology of the printing inks. The dependency of finger distance on these printing parameters has been established with high precision using large-scale industrial presses, whereas the meniscus distance L

was studied on a lab press where direct optical access to the nip was possible. We developed a theoretical model for coating flow in a thin film of non-Newtonian liquid is developed. The model is based on the lubrication approximation. It takes into account both capillary effects and the shear rate-dependent viscosity of the liquid. The evolution equation for a steady flow in the film is obtained, which satisfies the mass and momentum balance equations. The theory can predict the width L of the liquid bridge between the rotating cylinders for the complete set of considered printing parameters. It is shown that the characteristic pattern size ℓ correlates well with the theoretically predicted width L . Their values are of the same order. Moreover, the dependence between ℓ and L is nearly linear.

It appears likely that further progress on this point comes with improved observation methods for the meniscus details in the printing nip. This result indicates that the reason for the pattern formation during gravure printing lies in the oscillations caused by interfacial instabilities of the flow in the liquid bridge, especially in the contact region where it matches the flow in the coating film.

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Author Contributions. P.R.-B. and P.B. - design and performing of experiments, data interpretation, I.V.R. - conceptualization of this study, modeling, interpretation, J.H. and E.D. - scientific coordination, H.M.S. - conceptualization of this study, modeling, interpretation. All the coauthors contributed by paper writing.

Data Availability Statement. Raw data are available from the TUDatalib repository

Ethical Standards. The research meets all ethical guidelines, including adherence to the legal requirements of the study country.

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